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Journal of the
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THE VORTEX CHAMBER AS AN AUTOMATIC FLOW-CONTROL DEVICE

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SYNOPSIS

Coefficients of discharge for flow through horizontal orifices are greatly reduced when a vortex is present. Several flow control devices have been designed utilizing this phenomenon. This paper presents a rational explanation of the resulting coefficient variation from dynamic similarity. The results of laboratory experiments are shown, giving a means of predicting orifice flow characteristics from the inlet velocity condition.

INTRODUCTION

The free vortex has long been recognized as a significant phenomenon in the design of hydraulic structures. For the most part, however, it has been regarded as an undesirable possibility, and its consideration has been limited to the design of expensive appurtenances intended to eliminate, or at least reduce, the circular movements of the fluid.

In recent years, the properties of the free vortex have been utilized successfully to solve several flow control problems. Heim⁽³⁾ made studies of a "counterflow brake" for use in pumping installations. This device consists of a spiral shaped chamber with tangential and axial connections. The normal direction of flow is in at the center (axial) connection and out at the peripheral (tangential) connection. If the flow is reversed due to power failure, the resulting tangential inlet conditions cause a vortex motion within the chamber and therefore a greater resistance to flow. This device is thought of as a type of pipe fitting which may replace the check valve in special cases.

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Stevens⁽⁵⁾ made practical use of the vortex phenomenon in the design of a diversion structure for a combined sewer system. A small dam in the sewer at the downstream side of the diversion structure diverted all of the low secondary flow into an orifice chamber. The radial approach to the horizontal orifice resulted in normal orifice head-discharge relations. The high flow rate in the main sewer for a storm condition would pass over the dam, causing the entrance to the diversion chamber to be tangential and a vigorous vortex form. The resultant reduced orifice coefficient of discharge allowed only small portions of the storm water to pass into the interceptor.

Both of these control devices exhibit the advantage of relatively simple construction and maintenance because they have no moving parts. It should be expected that these ideas would find more popular use in hydraulic design except for the fact that to date very little information of design value is available concerning the characteristics of orifice flow with a significant vortex present.

Theory

The following signs and symbols are used in this paper and have been listed here for easy reference.

A ,	area of orifice
B ,	diameter of boundary ring
C ,	orifice coefficient of discharge
D ,	diameter of orifice
f_c ,	centrifugal force per unit volume
f_i ,	inertia force per unit volume
g ,	acceleration due to gravity
H ,	static head on center of orifice
p ,	pressure
R ,	Reynolds number
u ,	tangential component of velocity vector
v ,	discharge component of velocity vector
V ,	velocity vector
\underline{V} ,	vortex number
x, y ,	co-ordinates of a point in the flow field
α ,	angle the divergent jet makes with the horizontal
Γ ,	circulation
ρ ,	mass density
μ ,	dynamic viscosity

It has been shown by dimensional analysis⁽⁵⁾ that if the variables affecting flow through a horizontal orifice are known to be

$$f(D, B, p, \rho, V, \mu, \mathcal{T}) = 0$$

then the discharge equation will take the form

$$Q = CA \sqrt{2gH} \quad (1)$$

$$C = f\left(\frac{B}{D}, R, \frac{\mathcal{T}}{VD}\right) \quad (2)$$

\mathcal{T} is the circulation for the free vortex flow and is defined

$$\mathcal{T} = 2\pi x u \quad (3)$$

The drawdown of the water surface resulting from this vortex motion is (see Fig. 1)

$$y = \frac{u^2}{2g} \quad (4)$$

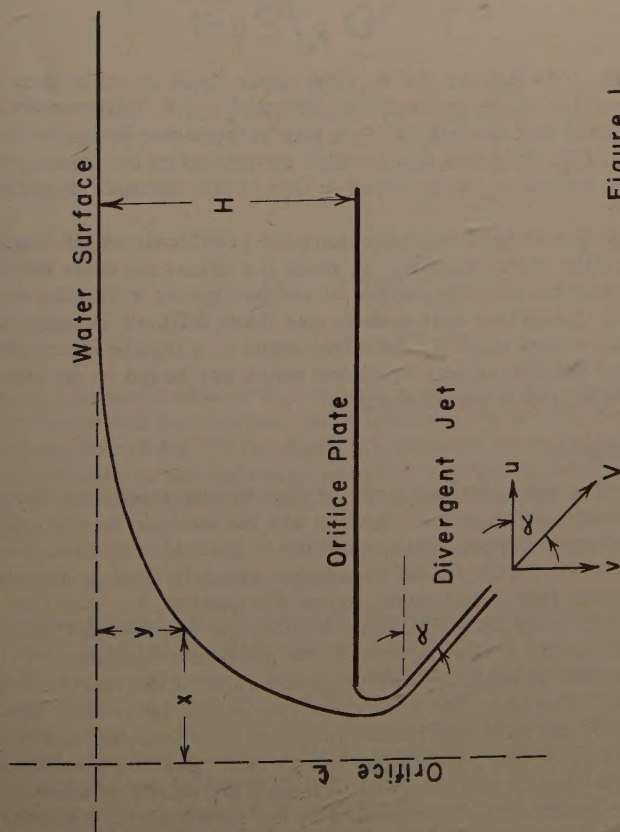


Figure 1

$$y = \frac{T^2}{8\pi^2 x^2 g}$$

The resulting water surface profile is hyperbolic. Superimposing radial flow on this vortex motion does not change the shape of the curve except in regions immediately near the orifice, because both components of the resultant velocity vary inversely with the radius.

In the earlier studies made at the University of Wisconsin, (5) it was noted that for vortex conditions strong enough to alter the orifice coefficient of discharge, the angle which the velocity vector makes with the radius, is such that the radial component is of negligible value in computing the drawdown.

By studying the water surface profile, and computing the circulation, Γ from Eq. (5), the coefficient of discharge in Eq. (2), was found to be

$$C = 0.686 - 0.218 \underline{V}$$

$$\underline{V} = \frac{\Gamma}{D \sqrt{2gH}}$$

between limits of $\underline{V} = 0.8$ and $\underline{V} = \pi$. The upper limit amounts to an extrapolation, since no data were accumulated beyond $\underline{V} = 2.7$, approximately. It might be expected that deviations from this relationship would be found at higher values of \underline{V} . It is felt that greater deviations of the actual profile from the theoretical should be encountered in this region because of greater surface effects.

No laboratory data have been accumulated previously which would allow computation of the vortex number, \underline{V} , from the actual chamber flow pattern. Measurements of the velocity profile at various points within the orifice chamber using immersion type meters are made difficult because of the disturbance of the vortex motion. This fact leads to a legitimate criticism of the design value of the curves and equations which are based on an assumed velocity-surface profile relationship.

Dynamic Similarity

In studying the dynamic similarity of flow through metering devices where the forces governing the particle motion are the surface forces represented by p and the viscous forces, which combine to give accelerative effects indicated by the inertia force, f_i , the parameter which is used to determine similarity of different flow cases is the Reynold's number, R . For flow through a particular device such as the pipeline orifice, the coefficient C does not vary greatly with R except in the regions of low Reynold's numbers.

This parameter is described as being the ratio of the inertial forces to the viscous forces. However, the division of one vector by another has no known mathematical or physical significance; hence, the relationship so expressed is the absolute or scalar ratio of the two forces. Besides insuring similarity of force magnitudes, it is necessary to insure similarity of kinematic effects (velocity, acceleration) as represented by the flow pattern, which amounts to accounting for the vector aspects of the problem. Ordinarily this is accomplished by specifying the geometric similarity with particular values of the

ratio of orifice diameter to pipeline diameter. This is permissible since for a given boundary configuration, the flow pattern is uniquely defined (if a sufficient approach length, measured usually in terms of orifice diameter, is specified in order to insure uniform flow in the approach to the orifice. This is another geometric effect.)

In the case of orifice flow with the possibility of tangential components to the velocity vectors, similarity cannot be defined in terms of the scalar Reynolds's number and the boundary geometry as indicated by $\frac{B}{D}$, because for a particular total acceleration as indicated by the Bernoulli equation solution,

$$V = \sqrt{2gH} \quad (8)$$

there is an infinite number of possibilities for the direction of the vectors. This problem can be handled by breaking the total inertial force into components and specifying

$$f_c = \rho \frac{u^2}{x} \propto \rho \frac{T^2}{L^3} \quad (9)$$

$$f_i = \rho V \frac{dV}{ds} \propto \rho \frac{V^2}{L} \quad (10)$$

the ratio

$$\frac{f_c}{f_i} \propto \frac{T^2}{L^2 V^2} \quad (11)$$

will determine kinematic similarity, and along with B/D (geometric), and R , dynamic similarity will be assured. This ratio will be seen to be the square of the vortex number in Eq. (2) or, since the velocity which is characteristic of the inertial effects is the discharge velocity, and the value of T/L which is characteristic of the centrifugal component is T/D (Eq. (3)), it is also the square of Eq. (7).

Experimental Methods and Results (Fig. 3)

The experimental work was done in the Marquette University Hydraulics laboratory and consisted of tests made in a sheet metal tank 5 feet square and 5 feet high. Orifices were cut in 2-foot diameter, 14 gage brass plates and mounted in the center of the bottom of the tank. Two inner boundary rings were used. Each was constructed of 14 gage sheet metal and had two diametrically opposite openings 6 inches wide to which were attached parallel sheet metal plates forming the entrance channel, which were fixed in position making an arbitrarily chosen angle of 30 degrees (or 20 degrees for some of the tests) with the tangent to the boundary ring. The larger ring was 3.5 feet in

diameter and 2.5 feet high and the smaller ring was 2.0 feet in diameter and 2.5 feet high. Water was admitted to the tank through 2 pipes 2 inches in diameter each of which branched near the floor of the tank into perforated 1/2 inch pipes which distributed the inflow uniformly around the outer boundary. Vortex profile measurements were taken as a check on the earlier University of Wisconsin tests⁽⁵⁾ with a moving point gage mounted on an aluminum bar 5 feet long with a 2-1/2 inch by 5/8 inch cross section. Discharge was measured by a 3-inch orifice in a 4-inch pipeline upstream from the test. For the low-flow tests a 2-inch nutating disc type water meter was used.

In the current series, 64 test runs were made in which the head was varied from 0.3 to 1.8 feet, \underline{V} varied from 0.85 to 2.6, and C varied from 0.2 to 0.5.

For the initial tests the larger boundary ring was used and the entrance channel vanes set at a 30 degree angle with the boundary tangent. Tracer dye and a small current meter were used to check for a uniform velocity within the entrance channels. No significant variation in velocity from the inside to outside of the channels was found; therefore, the average velocity was computed from the discharge and cross-sectional area, and this value was used in further computations. Within the boundary ring the velocity was found to be essentially tangential (negligible radial components except immediately near the orifice) and its magnitude the same as the entrance velocity. The change in direction of the fluid velocity vector had thus been accomplished with negligible change in momentum indicating negligible shear forces. For some of the tests, both entrance channels were used, and for others one was sealed shut. The results in either case were identical even though some noncentricity of the air core was noted for the one vane condition.

The results for the 20 degree approach angle exactly corresponded to those for the 30 degree angle. The velocity within the boundary (at a 1.75 foot radius from the center of the orifice) was found to be the same as that within the entrance channels, and so this value was used in computing the circulation from Eq. (3).

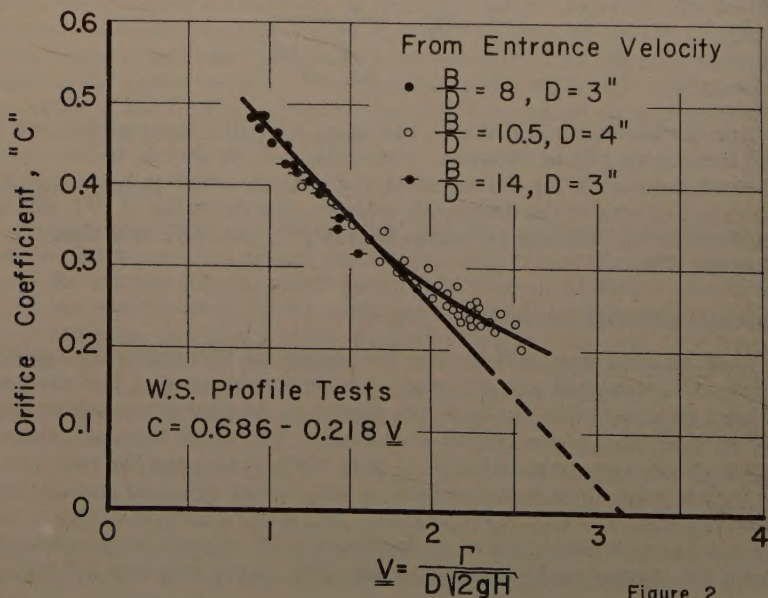


Figure 2

The curve (Fig. 2) shows the variation of orifice coefficient with vortex number for these data as computed from the entrance velocity. The data closely follow the straight line indicated by the water surface profile measurements for vortex numbers less than 2.0. Above this value a separate curve is evidently established. This deviation probably occurs because the profile computations were based on a measurement made only one diameter away from the orifice, more closely approximating the true energy condition in that region. High vortex numbers represent conditions of high velocity near the orifice and therefore greater shear losses as already noted.

As a further check on the normal acceleration relations, measurements of the angle α (Fig. 1) of the divergent jet were made. From Eq. (3)

$$\frac{T}{D} = \pi u \quad (12)$$

(because upon release, the circular motion becomes tangential) Substitution in Eq. (7) gives

$$\underline{V} = \pi \frac{u}{V} = \pi \cos \alpha \quad (13)$$

$$\cos \alpha = \frac{\underline{V}}{\pi} \quad (14)$$

Results of these measurements are shown below.

\underline{V}	$\cos^{-1} (\underline{V}/\pi)$	α (measured)
0.952	72.4°	70°
1.25	66.6°	64°
1.70	57.3°	60°
1.86	53.7°	52.5°
2.24	44.5°	46°
21.25	44.3°	49°



Figure 3

Because of the small clearance between the tank bottom and the floor of Hydraulic Laboratory, considerable difficulty was experienced in making the measurements. A complete series of such tests has thus not yet been attempted. Because the jet was not well defined (considerable wavering) the check indicated by these data is better than had been expected.

CONCLUSIONS

Prediction of orifice coefficients for vortex flow can be made on the basis of the velocity at inlet to the orifice chamber. The accuracy of such prediction for chambers which are concentric with the orifice should depend on the ability to estimate the inlet velocity. The curves shown should be extended to include larger vortex numbers. To do this it may be necessary to build an apparatus in which the head and inlet velocity can be independently controlled (such as a model of the Stevens combined sewer system which affords the advantage for laboratory purposes of by-passing part of the total flow.

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THE SETTLING PROPERTIES OF SUSPENSIONS

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ABSTRACT

This paper contains the results of an analytical and experimental investigation of the settling properties of suspensions of particles in fluid. The use of these properties in predicting the sedimentation of the particles is outlined.

1. INTRODUCTION

In many engineering problems it is necessary to deal with a flowing fluid in which particles are suspended. The engineer commonly encounters such problems in river channels, in water and sewage clarification, in reservoirs and in the delta regions of rivers. One of the tasks associated with these problems is to determine how much material will settle out of suspension and where that material will settle.

The factors that affect the settling of suspended particles can be divided into two groups. Those of the first group can be called the conditions of flow. They are the temperature and pressure in the suspension, the velocity distribution of the flow, and the nature of the turbulence of flow. The factors of the second group can be called the settling properties of the suspension.

The term "settling properties of a suspension" refers to how the particles behave in a given set of conditions of flow. Thus, for some specified temperature, pressure, fluid velocity and turbulence, the particles of a suspension will settle, flocculate or be diffused in some manner. The manner will vary from suspension to suspension, and must be determined experimentally for each.

Hence, the problem of determining how material will settle out of a flowing suspension can be divided into two parts. First the flow conditions must be

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measured, and second, the settling properties of the suspension must be measured. Neither of these two is an easy task, and considerable research and development is necessary before either will be done satisfactorily.

This paper deals only with the settling properties of a suspension and use in calculating the removal of the particles. Section 2 contains a general discussion of the problem of calculating the removal. This discussion points out how the settling properties are to be used and indicates the type of experimental information needed. The experimental determination of settling properties is discussed in Section 3. Some results already obtained are presented, and the experiments yet to be developed are outlined. In Section 4, the research is related to more practical engineering problems.

2. Calculating the Removal of Suspended Particles

a. The Problem

It was stated above that one of the tasks of the engineer is to determine in advance how the particles will settle out of a given flowing suspension. More specific, consider a suspension of particles in water which is flowing in an open channel. The fluid velocity and turbulence level are low enough for particles to settle to the bottom of the channel. The problem is to calculate the amount of settled material as a function of distance along the channel.

Such a calculation involves the solution of some form of continuity equation expressing the conservation of suspended matter at any point in the suspension. In order to derive the equation it is first necessary to consider the factors affecting the concentration of particles at a point. The first of these factors is the settling of the particles.

b. Settling of the Suspended Particles

The term "settling velocity of a suspended particle" refers to the velocity that the particle would have if it were settling in perfectly still fluid. A suspension may contain particles of many settling velocities and the distribution of these settling velocities is an important settling property of the suspension. In calculations related to settling it is just as reasonable to characterize particles by these settling velocities as by their size, shape, and density. For this purpose, the settling velocity that indicates how soon the particle is going to be removed from suspension by settling to the bottom.

Let w represent settling velocity and let it be positive downward. The settling velocities of the particles in a suspension will range from zero to the velocity of the fastest particle. For numerical calculations and theoretical discussion it is convenient to divide the range of values of w into a number of intervals Δw . Then w_1 can refer to a class of settling velocities between the values $(i-1)\Delta w$ and $i\Delta w$, and w_2 to those velocities between $i\Delta w$ and $(i+1)\Delta w$. In general, w_i will indicate velocities between the values $(i-1)\Delta w$ and $i\Delta w$. The particles with settling velocity w_i will be called i -particles.

The concentration of i -particles at any point in the suspension will be called f_i , the units of which are mass per unit volume of suspension. The concentration can vary from point to point and from time to time. Letting x , y , and z represent coordinates which locate a point, and t represent time, the concentration will be a function, $f_i(x, y, z, t)$. The term f_i indicates that f is a function which varies as i varies. Hence, f is the frequency function for

settling velocities at point x, y, z . In this paper it will be called the settling velocity distribution at x, y, z .

At any point in the suspension, the flux of i -particles due to settling will be given by

$$w_i f_i (x, y, z, t)$$

is the rate at which i -particles pass through a unit horizontal area at point x, y, z , and time t . The rate at which all particles pass through this area

$$\sum_{i=1}^{\infty} w_i f_i (x, y, z, t)$$

By definition, this total flux is simply the product of the local instantaneous settling velocity, $\bar{w}(x, y, z, t)$, and the local instantaneous particle concentration, $\phi(x, y, z, t)$. That is,

$$\bar{w}(x, y, z, t) \phi(x, y, z, t) = \sum_{i=1}^{\infty} w_i f_i (x, y, z, t) \quad (1)$$

ere

$$\phi(x, y, z, t) = \sum_{i=1}^{\infty} f_i \quad (2)$$

ence, the product $\bar{w}\phi$ describes the motion of particles due to settling.

Movement of Particle Due to Fluid Motion

At any point in the suspension, the particles also have a motion due to fluid flow at the point. If the flow is without turbulence, this motion is simply the resultant fluid velocity. Let this resultant be represented by the vector $\bar{u}(x, y, z, t)$. It will have x, y , and z -components of $U(x, y, z, t)$, $V(x, y, z, t)$ and $W(x, y, z, t)$ respectively.

When the flow of a suspension is turbulent, the motion of the particles is usually considered in two parts. First, the particles are considered as being carried along by the temporal mean fluid velocity, also called $\underline{U}(x, y, z, t)$. Superimposed on this motion is the diffusion of particles by the turbulent fluctuations of fluid velocity. The decision as to what is the main flow and what is turbulence will depend upon the flow pattern for each individual case. In order to deal with the particle motion caused by turbulence, it is common to define a coefficient of diffusion. Let e_{ik} be the coefficient for i -particles in the x direction. Then the flux of i -particles through a unit area normal to the x direction can be given as

$$-e_{ik} \frac{\partial f_i}{\partial x}$$

where there may also be an e_{iy} and e_{iz} , the coefficient will be called e_i when a specific direction is implied.

The coefficient e_i should not be confused with the coefficient for diffusion transfer of momentum between neighboring elements of fluid. For the sake

of differentiation the latter will be called e_m . A reasonable assumption, made, is that e_i is equal to e_m . For a detailed discussion of this point, the reader is referred to the experimental and analytical work of Vanoni(1) and Ismail.(2) Their work indicates that e_i not only differs e_m , but that the manner in which it differs varies with i . For coarse sand (w_i large) in fresh water, e_i was smaller than e_m while for fine sand (w_i small) e_i was larger. Nevertheless, the order of magnitude of the difference was such that it may be reasonable to assume that $e_i = e_m$ in engineering calculations. With this assumption there is no need for the subscript i and the diffusion coefficient for particles can be called e with an appropriate subscript when specific directions are mentioned.

Vanoni also found that e appears to vary with the particle concentration. As yet, there is so little information on this variation that it cannot be considered in the calculation of removal. However, it must be an important effect when ϕ is large. It is hoped that future research will yield quantitative information about the effect of ϕ on e . In this paper, the effect is ignored.

d. Equation of Continuity

The motion of particles due to settling, mean temporal fluid velocity and turbulent diffusion produces a flux of particles in one or all of the x, y , and z directions. The resultant flux can be represented by the vector. In order to describe the vector let \underline{i} , \underline{j} and \underline{k} be unit vectors in the x, y and z -directions respectively and let z be positive in the direction of positive w .

Consider first the flux of i -particle. The resultant is represented by the vector, \underline{a}_i , where

$$\underline{a}_i(x, y, z, t) = \underline{k} w_i f_i + \underline{U} f_i - \underline{i} e_{ix} \frac{\partial f_i}{\partial x} - \underline{j} e_{iy} \frac{\partial f_i}{\partial y} - \underline{k} e_{iz} \frac{\partial f_i}{\partial z}$$

The conservation of i -particles at x, y, z, t dictates that the divergence of \underline{a}_i plus the time rate of change of f_i minus any source flow of i -particles is zero. Hence,

$$\nabla \cdot \underline{a}_i + \frac{\partial f_i}{\partial t} - P_i(x, y, z, t) = 0$$

where

$$\nabla = \underline{i} \frac{\partial}{\partial x} + \underline{j} \frac{\partial}{\partial y} + \underline{k} \frac{\partial}{\partial z}$$

The term, P_i , in Eq. (3) represents a distributed source of i -particle at x, y, z . This term accounts for the effects of hindered settling and flocculation. If a particle of class w_i becomes attached to another in the process of flocculation it will generally experience a change in settling velocity. Hence, it appears from class w_i and appears in another class. Similarly a particle may have one settling velocity when the local concentration has a certain value and when the local concentration changes, hindered settling may change the velocity to w_i . It follows that P_i must represent the rate at which particles acquire settling velocity w_i less the rate at which i -particles acquire other settling velocities. The units of P_i are mass per unit time per unit volume.

For an incompressible fluid the expansion of Eq. (3) becomes

$$\begin{aligned} \frac{\partial f_i}{\partial t} + \frac{\partial (w_i f_i)}{\partial z} + U \frac{\partial f_i}{\partial z} + V \frac{\partial f_i}{\partial y} + W \frac{\partial f_i}{\partial z} - \frac{\partial}{\partial x} (e_{ix} \frac{\partial f_i}{\partial x}) \\ - \frac{\partial}{\partial y} (e_{iy} \frac{\partial f_i}{\partial y}) - \frac{\partial}{\partial z} (e_{iz} \frac{\partial f_i}{\partial z}) - P_i = 0 \end{aligned} \quad (5)$$

cept for the last term this equation is similar to that expressed by Robbins,⁽³⁾ McNown,⁽⁴⁾ Van Driest,⁽⁵⁾ and others.

There is an Eq. (5) for each value of i . By summing these equations and introducing Eqs. (1) and (2) into the sum, the continuity equation for total concentration becomes

$$\begin{aligned} \frac{\partial \phi}{\partial t} + \frac{\partial (\bar{w} \phi)}{\partial z} + U \frac{\partial \phi}{\partial x} + V \frac{\partial \phi}{\partial y} + W \frac{\partial \phi}{\partial z} - \frac{\partial}{\partial x} (\sum_i e_{ix} \frac{\partial f_i}{\partial x}) \\ - \frac{\partial}{\partial y} (\sum_i e_{iy} \frac{\partial f_i}{\partial y}) - \frac{\partial}{\partial z} (\sum_i e_{iz} \frac{\partial f_i}{\partial z}) - \sum_i P_i = 0 \end{aligned} \quad (6)$$

Since flocculation and hindered settling produce no mass,

$$\sum_i P_i = 0 \quad (7)$$

Substituting Eq. (7) into (6) and assuming that e_i is the same for all i gives the following equation:

$$\begin{aligned} \frac{\partial \phi}{\partial t} + U \frac{\partial \phi}{\partial x} + V \frac{\partial \phi}{\partial y} + W \frac{\partial \phi}{\partial z} + \frac{\partial}{\partial z} (\bar{w} \phi) - \frac{\partial}{\partial x} (e_x \frac{\partial \phi}{\partial x}) \\ - \frac{\partial}{\partial y} (e_y \frac{\partial \phi}{\partial y}) - \frac{\partial}{\partial z} (e_z \frac{\partial \phi}{\partial z}) = 0 \end{aligned} \quad (8)$$

The effects of flocculation and hindered settling do not appear explicitly in Eq. (8). These effects do appear implicitly, however, because they cause changes in \bar{w} . Eq. (8) cannot be solved without information on how \bar{w} will change with particle concentration, flocculation, differential settling, and diffusion. This information will be obtained from studies of the settling properties of the suspension. In short, the settling properties of the suspension occur implicitly in Eq. (8) in the term \bar{w} .

Initial and Boundary Conditions

The solution of Eq. (8) requires a knowledge of initial conditions or boundary conditions or both. It becomes necessary, therefore, to think about specific physical situations. For present purposes, consider the section of open channel shown in Fig. 1. The distance along the channel is called x and is taken positive in the direction of flow. The distance down from the free surface of the suspension is called z and is positive downward. The y -direction is normal to x and z in such a way as to produce a right-handed xyz system. For this channel, the initial condition is the settling velocity distribution in a plane $x = 0$. Hence, it is necessary to take a sample of suspension from a point at $x = 0$ and obtain the settling velocity distribution of the sample by experimental analysis. If the distribution is not constant over the channel cross-section, samples must be taken from several points in the section and each

sample must be analyzed. The result will be a spatial distribution of settling velocity distribution at $x = 0$, and this result is the initial condition for settling in the channel.

The boundary conditions occur at the top surface of the suspension and at the bottom of the channel. At the top, the condition is the rate at which particles cross the surface. Usually no particles pass through this surface. In a density current along the bottom of a reservoir or settling tank, material may be settling out of relatively still water above into the current below.

At the bottom of the channel, the boundary condition depends on the behavior of the particles which have already settled to the bottom. If particles which once settled to the bottom of the channel are picked up into the flow, the process is called entrainment or resuspension. Let the rate at which particles are picked up per unit area of bed be E . The boundary condition can be stated as

$$E = - (e_z \frac{\partial \phi}{\partial z}) \text{ at bed}$$

This equation is of the form used by Dobbins⁽³⁾ to describe pickup of suspended particles in a turbulence jar. Dobbins verified this equation experimentally for $E = \text{constant}$ and $E = 0$.

Theoretical work by Lane and Kalinske⁽⁶⁾ shows that E will depend upon the settling velocities of the settled particles, the roughness of the bed and the pattern of turbulence near the bed. Thus for a given flow and bed condition, E will depend on the properties of the suspension.

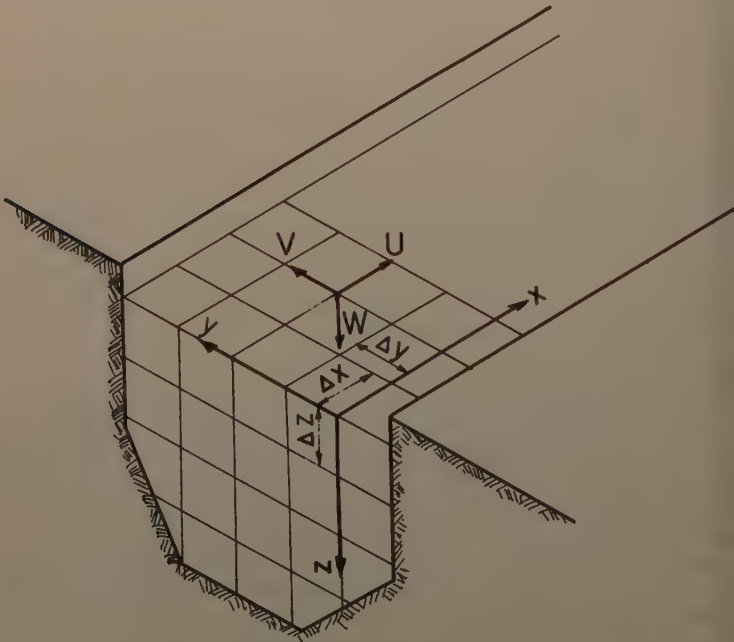


Fig. 1. Suspension flowing in open channel.

The resuspension of particles must be studied experimentally for each suspension. During the study, the flow conditions should be varied and resuspension measured. The end result should give the resuspension rates as a function of flow conditions for the suspension being studied. Experiments of this sort are currently in progress at the California Institute of Technology. It appears that a great deal of research is still to be done before the results can be used in calculating removal.

Solution of the Continuity Equation

Except for a few simple cases it is necessary to solve the equation of continuity by some numerical method. For this purpose, Eq. (8) is written in the form

$$\begin{aligned} \frac{\partial \phi}{\partial t} + \left(U - \frac{\partial e_x}{\partial x} \right) \frac{\partial \phi}{\partial x} + \left(V - \frac{\partial e_y}{\partial y} + \left(W - \frac{\partial e_z}{\partial z} \right) \frac{\partial \phi}{\partial z} \right. \\ \left. + \frac{\partial (\bar{w} \phi)}{\partial z} - e_x \frac{\partial^2 \phi}{\partial y^2} - e_y \frac{\partial^2 \phi}{\partial x^2} - e_z \frac{\partial^2 \phi}{\partial z^2} \right) = 0 \end{aligned} \quad (10)$$

In Eq. (10), U , V , W , e_x , e_y and e_z are all determined by measurements. In general, they will be functions of x , y , z . However, if turbulence is affected by particle concentration, the e 's will also be a function of ϕ . In such a case the relationship between e and ϕ will have to be determined experimentally.

Eq. (10) is in a form easily reduced to finite-difference form. For this purpose, the space occupied by the suspension is divided into rectangular parallelepipeds as shown at the upstream end of the channel in Fig. 1. Each parallelepiped has the dimensions Δx , Δy and Δz as shown. The corners of the parallelepipeds form a lattice of points where the value of ϕ is to be found. The subscript l will indicate that a point in the lattice has an x -coordinate $l\Delta x$, while subscripts m and n indicate coordinate $m\Delta y$ and $n\Delta z$ in the y - z -directions respectively. Using this notation, Eq. (10) can be written in the following form, for steady state settling.

$$\begin{aligned} \left(U - \frac{\partial e_x}{\partial x} \right)_{l, m, n} \frac{\phi_{l+1, m, n} - \phi_{l, m, n}}{\Delta x} \\ + \left(V - \frac{\partial e_y}{\partial y} \right)_{l, m, n} \frac{\phi_{l, m+1, n} - \phi_{l, m, n}}{\Delta y} \\ + \left(W - \frac{\partial e_z}{\partial z} \right)_{l, m, n} \frac{\phi_{l, m, n+1} - \phi_{l, m, n}}{\Delta z} \\ + \frac{(\bar{w} \phi)_{l, m, n+1} - (\bar{w} \phi)_{l, m, n}}{\Delta z} \end{aligned}$$

$$\begin{aligned}
 & - (e_y)_{l, m, n} \frac{\phi_{l+1, m, n} - 2\phi_{l, m, n} + \phi_{l-1, m, n}}{\Delta x^2} \\
 & - (e_y)_{l, m, n} \frac{\phi_{l, m+1, n} - 2\phi_{l, m, n} + \phi_{l, m-1, n}}{\Delta y^2} \\
 & - (e_z)_{l, m, n} \frac{\phi_{l, m, n+1} - 2\phi_{l, m, n} + \phi_{l, m, n-1}}{\Delta z^2} = 0
 \end{aligned}$$

To use Eq. (11) it is necessary to know ϕ and \bar{w} at all points in the $x = r = 0$. This knowledge constitutes the initial condition. Any resuspension of particles between the planes $x = 0$ and $x = \Delta x$ constitutes the boundary condition and must also be known. With these known quantities, it is possible to calculate ϕ at all points in the plane $x = \Delta x$ or $r = 1$.

In order to repeat the process, it is necessary to know $\bar{w}\phi$ at points in the plane $r = 1$. Hence, the fundamental part of this calculation is to determine how $\bar{w}\phi$ changes between the plane $r = 0$ and $r = 1$, or, in general, between two planes, r and $r + 1$. This change in $\bar{w}\phi$ will depend on the settling characteristics of the suspension.

Once the ϕ is calculated over cross sections at various values of r , it is a simple matter to compute the rate at which particles are removed in a distance, $r\Delta x$. Thus the problem stated in part (a) above can be solved.

The procedure outlined here requires more information than is usually available to the engineer. Its main value, at present, is in showing specifically what research and development are necessary for complete calculation. Methods of measuring fluid velocity and turbulent diffusion must be improved, methods of measuring resuspension must be developed, and methods for determining the settling properties of a suspension must be expanded.

Furthermore, the finite-difference calculation shows that information about the settling properties of a suspension must be obtained in a very specific form. This information must relate changes in the local mean settling velocity $\bar{w}(x, y, z)$ to particle settling, particle concentration, flocculation, hindered settling, turbulence and any other conditions prevailing in the immediate neighborhood of the point x, y, z . A program of research was initiated for the purpose of obtaining this kind of information about suspensions. The progress made to date is reported in Section 3.

Due to the problems involved in calculating removal, it is natural to turn to the study of settling in hydraulic scale models. However, the model studies can be misleading unless the similarity of settling is considered. Such similarity is discussed next.

g. Scale Models and Similarity in Sedimentation

The relationship between settling in the model and the prototype is maintained by reducing Eq. (8) to dimensionless form. All velocities in the model can be given in terms of a characteristic velocity U_0 and dimensionless velocities U^* , V^* and W^* . The result is

$$U = U^* U_0, \quad V = V^* V_0, \quad W = W^* W_0, \quad w_i = w_i^* U_0$$

Similarly, distance can be written in terms of a characteristic length x_0 , and diffusion coefficients can be written in terms of a characteristic coefficient, as follows:

$$y = y^* y_0, \quad \gamma = \gamma^* y_0, \quad z = z^* z_0$$

$$e_x = e_x^* e_0, \quad e_y = e_y^* e_0, \quad e_z = e_z^* e_0$$

Dimensionless concentration can also be written in terms of a characteristic value, ϕ_0 .

$$\phi^* = \frac{\phi}{\phi_0}$$

Dimensionless time can be written in terms of characteristic length and velocity.

Substituting these dimensionless quantities in Eq. (8) gives

$$\frac{\partial \phi^*}{\partial t^*} + U^* \frac{\partial \phi^*}{\partial x^*} + V^* \frac{\partial \phi^*}{\partial y^*} + W^* \frac{\partial \phi^*}{\partial z^*} + \frac{\partial (\bar{w}^* \phi^*)}{\partial z^*} + \frac{e_0}{U_0 x_0} \frac{\partial}{\partial y^*} (e_x^* \frac{\partial \phi^*}{\partial x^*}) + \frac{\partial}{\partial y^*} (e_y^* \frac{\partial \phi^*}{\partial y^*}) + \frac{\partial}{\partial z^*} (e_z^* \frac{\partial \phi^*}{\partial z^*}) = 0 \quad (12)$$

There are three kinds of similarity involved in Eq. (12). The first two are geometric and kinematic. Because of them the quantity $\frac{e_0}{U_0 x_0}$ will be the same dimensionless constant in model and prototype. Furthermore, the value \bar{w}/U_0 at a point in the model must be equal to that at a corresponding point in the prototype. Hence geometric and kinematic similarity demands that the settling velocities in the suspension be scaled in the same ratio as the fluid velocities.

The third type of similarity concerns the changes in settling velocities. Suppose that in the prototype, the mean settling velocity changes by an amount $\Delta \bar{w}$ in some distance Δx . For similarity \bar{w} must change in the model such that the dimensionless rate of change is the same for both model and prototype. Denoting scale model quantities by s and prototype quantities by p it follows that

$$\frac{(\frac{\Delta \bar{w}}{U_0})_s}{(\frac{\Delta x}{x_0})_s} = \frac{(\frac{\Delta \bar{w}}{U_0})_p}{(\frac{\Delta x}{x_0})_p}$$

$$\frac{(\frac{\Delta \bar{w}}{\Delta x})_s}{(\frac{\Delta \bar{w}}{\Delta x})_p} = (\frac{U_0}{x_0})_s (\frac{x_0}{U_0})_p \quad (13)$$

If, like most open channel models the model was designed according to Froude model law, Eq. (13) becomes

$$\frac{\left(\frac{\Delta \bar{w}}{\Delta x}\right)_s}{\left(\frac{\Delta \bar{w}}{\Delta x}\right)_p} = \left[\frac{(x_o)_s}{(x_o)_p} \right]^{-1/2}$$

The flocculation may be related to turbulent mixing which is scaled according to Froude's law as follows

$$\frac{(e_o)_s}{(e_o)_p} = \frac{(U_o x_o)_s}{(U_o x_o)_p} = \left[\frac{(x_o)_s}{(x_o)_p} \right]^{3/2}$$

Consequently the ratio of the effect of flocculation is equal to the negative power of the length ratio while the ratio of a mechanism contributing to flocculation is equal to the three-halves power of the scale ratio.

Eqs. (14) and (15) show that the response of the suspension to flow conditions cannot be the same in model and prototype. It follows that a suspension with "scaled" settling properties must be used in the model. Usually, however, the same suspension must be used in model and prototype. In this situation there are two approaches to using models.

The first approach is to use the model to predict only the flow in the prototype; separate experiments are performed to study the settling properties of the suspension. The measured settling properties are then used to calculate the removal that will occur in the predicted flow. This method is suitable for the study of such problems as silt deposition in reservoirs, sedimentation at the mouths of rivers, and the diffusion and sedimentation of sewage and other wastes in bays and estuaries.

In the second approach the settling is studied in the scale model. The model results are then "scaled up" to predict prototype results. In order to perform this scale-up, it is necessary to develop a scale equation giving removal in the prototype as a function of removal in the model. From the foregoing analysis it is obvious that such an equation should be based on the properties of the suspension as on the model laws. Therefore separate experiments on the settling properties should be conducted in conjunction with the model tests. This second approach is frequently used in pilot plant studies of settling tanks.

3. Research on Quiescent Settling

a. Purpose and Scope

The purpose of the research described in this section was to develop methods for determining the settling properties of an individual suspension. More specifically, the objective was to measure the settling velocity distribution of a suspension, the local mean settling velocity at various points

suspension, and the factors affecting the local mean velocity. Most of the work was confined to quiescent settling.

The approach was both experimental and theoretical. The experiments consisted of allowing a suspension to settle quiescently in a vertical tube. During settling, small samples were withdrawn from various locations in the tube, and these were analyzed for suspended-solids concentration. To supplement the experiments, theoretical analyses were made of the quiescent settling of discrete particles in a settling tube, the kinetics of flocculation during quiescent settling, and the analyses of data from the experiments.

Settling Velocity Distributions

The purpose of the first phase of the research was to measure the settling velocity distribution of many varied suspensions. The pipette analysis⁽⁷⁾ was used for the measurement. A portion of a suspension was shaken and allowed to settle in a vertical glass tube. During settling, samples were withdrawn from a known distance below the top surface of the suspension by means of a open-tip pipette or small glass tube.

The results of some of the experiments are shown in Figs. 2, 3 and 4. The ordinate represents the ratio of the particle concentration in the sample to the original concentration of the portion at the beginning of settling, while the abscissa indicates that the sample was taken at depth z below the surface at time t after the beginning of settling.

Under the following conditions, a curve from these figures represents a cumulative frequency distribution of settling velocities.

- (1) At the beginning of settling the particles of each class of settling velocities were uniformly distributed throughout the suspension.
- (2) No flocculation, turbulence, or hindered settling occurred during the experiment. In this case, the settling velocity distribution is the major settling property of the suspension and is often sufficient for calculating removal. Dobbins⁽³⁾ and Camp^(13,16) have calculated the removal for

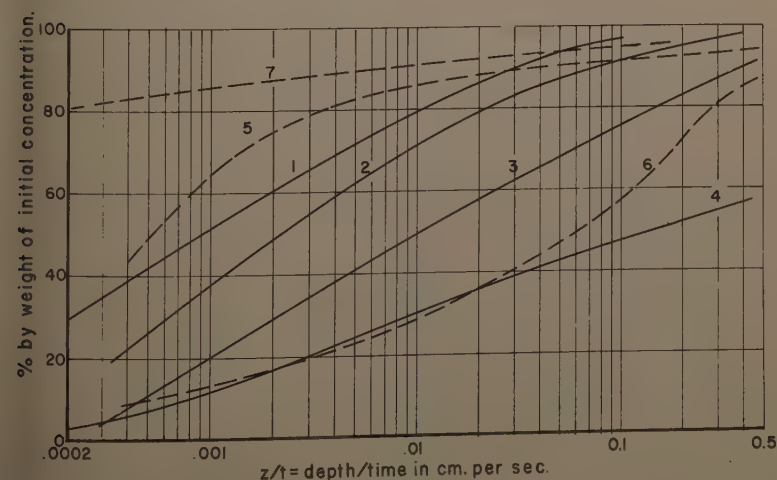


Fig. 2 Comparison of pipette analyses (see table I)

certain cases of particles settling without flocculation or hindering settling.

Fig. 2 shows the results of pipette analyses performed on primary effluent from two large sewage treatment plants, primary and secondary effluent from a third, and various mixtures of effluent with digested sludge and sea water with digested sludge. The analyses were made by the writer in connection with studies of the marine disposal of sewage and sludge in the vicinity of Los Angeles, California.⁽⁸⁾ These were made in an apparatus which maintained the temperature of the suspension at 31.5° C. The average depth of sample was about 40 cms for each analysis.

Even though the pipette analysis takes no account of flocculation or hindering settling, the curves of Fig. 2 indicate large differences between the settling properties of the various suspensions. For example, curves 1, 2 and 3 represent primary effluents from sewage treatment plants. However, the approximate median settling velocities of suspensions 1 and 3 differ by a factor of 10.

Pipette analyses were also performed on raw Pasadena sewage obtained from a trunk sewer of the Los Angeles County Sanitation Districts. This

TABLE 1

Sources of the Suspensions Represented in Figure 2

<u>Curve</u>	<u>Source</u>	<u>Initial Concentration</u> (equal to Susp. Solids)
1	Primary effluent from the treatment plants of the Orange County (California) Sanitation District	893 mg/l
2	Primary effluent from the Hyperion Sewage Treatment Plant (City of Los Angeles).	212 mg/l
3	Primary effluent from the Joint Disposal Plant of the Los Angeles County Sanitation Districts	314 mg/l
4	Primary effluent plus 1% by volume of digested sludge from the Joint Disposal Plant, LACSD	527 mg/l
5	Primary effluent, plus elutriation effluent from Hyperion Sewage Treatment Plant	289 mg/l
6	One part digested sludge from Hyperion Sewage Treatment Plant plus 19 parts sea water	32,200 mg/l
7	Secondary effluent from Hyperion Sewage Treatment Plant	33 mg/l

er serves most of Pasadena, San Marino, South Pasadena, and parts of contiguous communities, with a total sewered population of about 200,000. The samples obtained from the sewer were gross samples; they were taken in the morning and tested in the afternoon of the same day. Before the experiment, gross sample was poured into a large ceramic crock. When a smaller portion was required for testing, the gross sample was stirred, and while stirring continued, the smaller portion was taken.

A five-gallon gross sample was obtained on each of several days, and a portion was taken from each for pipette analysis. General information about analyses is given in Table 2 and the results are plotted in Fig. 3. These curves show a variation in settling properties of the sewage from one run to next. For some engineering purposes, it would be desirable to represent data from all four runs by a single curve. The median settling velocity for such a curve would be about 0.02 cm. per sec., and the median velocities for the individual curves differ from this value by a factor of 1.5 or less. This deviation is small compared to the differences between median velocities in Fig. 2. Hence, for the purpose of comparing Pasadena sewage with the suspensions of Fig. 2, all four runs could be represented by a single curve. From the curves of Figs. 2 and 3, two conclusions can be drawn. First, a suspension which may be considered as similar can have significantly different settling properties. Second, even for a suspension as heterogeneous as sewage, the difference between runs for a single suspension can be small compared to the difference between suspensions.

Table 2

Pipette Analyses of Pasadena Sewage

Settling tube - 1 liter graduate

Volume used - 1 liter

Temperature control - none

Size of samples withdrawn - 25 ml.

Method of withdrawing samples - 25 ml. broken tip pipette lowered into suspension by hand for each sample.

Depth of samples - 22 cm. below surface of sewage.

Date	Gross Sample Obtained	Pipette Analysis Began	Initial Concentration	Temp. During Test
ur. Oct. 21/54	8:30 am	2:35 pm	304 mg. / l	
ur. Oct. 28/54	8:45 am	11:40 am	320	22-27°C.
on. Nov. 15/54	9:00 am	1:45 pm	374	21-23.5°C
on. Nov. 22/54	9:00 am	1:00 pm	380	23-30°C

Part of the difference between the curves of Fig. 3 may be due to sampling error. Whenever a portion is withdrawn from a heterogeneous suspension such as raw sewage, the properties of the portion may not be the average properties of the suspension. To test this error, tests were made on two portions from a single gross sample. By means of two identical glass tubes placed in a single constant-temperature water bath, two similar portions of sewage from a single gross sample were subjected to identical pipette analyses. The portions were approximately four liters in volume.

The experiment was rerun for a second gross sample, and the results of both runs were plotted on the graph shown in Fig. 4. The difference between curves for one run is nearly as large as the difference between curves in Fig. 3. Hence, some of the variation in the latter curves is due to sampling errors.

Sampling errors can be reduced by increasing the volume of the portion used in the suspension. The portions used in these pipette analyses varied between 1 and 4 liters, and they appear to be too small. Hence larger samples are recommended for suspensions of considerable heterogeneity.

c. Factors Affecting \bar{W}

Unless the particles of a suspension settle without flocculation or hindered settling, the pipette analysis gives only a rough indication of the settling properties of a suspension. In a more general situation it is necessary to do an experiment that will show how flocculation and hindered settling affect settling velocities of the particles.

In hindered settling the concentration of suspended particles is high enough for the presence of one particle to affect the settling of its neighbors. Without flocculation, on the other hand, a faster particle overtakes a slower one and becomes attached to it. The two settle henceforth as a unit with a velocity usually different from the original velocity of either particle. These two effects cannot be separated in an experiment. If they occur simultaneously the result is a single change in \bar{W} . However, in order to find out precisely

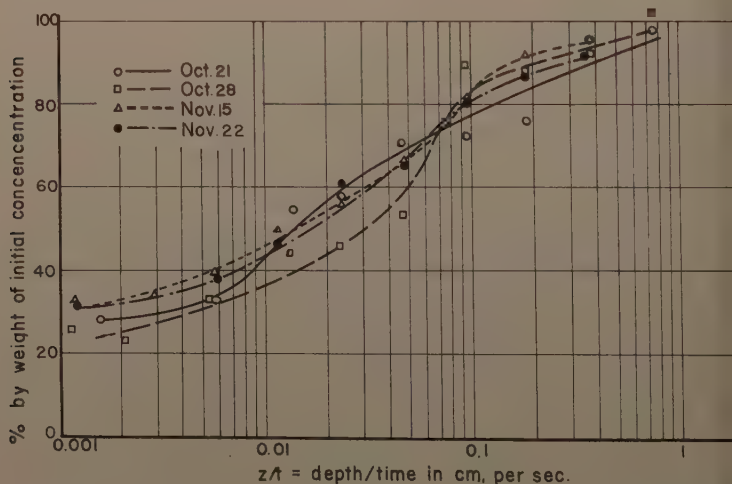


Fig. 3. Pipette analyses of Pasadena sewage (See Table 2).

produce the changes in \bar{w} it is convenient to think of them separately. The hindered settling of uniform particles without flocculation has been studied theoretically and experimentally by McNown and Lin⁽⁹⁾ and Steinhour,⁽¹⁰⁾ among others. The former found that the individual settling velocity of a particle depends upon the volume concentration of the particles and upon the Reynolds number of the particle. Steinhour, on the other hand, produced a formula giving velocity as a function of volume concentration alone. His formula was based on experiments where particles were settling in the Stokes range.

Both of these investigators considered cases where the particle concentration was uniform throughout the settling tube. Kynch⁽¹¹⁾ went further and made a theoretical analysis of uniform particles in a tube where the concentration varied with depth. By assuming that the individual particle velocity depended solely on the concentration in the neighborhood of the particle, he obtained results which agree with experimental observations on the subsidence of thick slurries.

On the basis of these studies it was concluded that the ratio of the settling velocity at concentration ϕ to velocity at $\phi = 0$ depends primarily on ϕ , with particle velocity as a secondary factor. Hence, in a suspension with particles of many velocities, a change in ϕ will cause the velocity of each particle to change by the same ratio. However, the magnitude of the change in velocity will equal the product of the ratio and the particle velocity. Thus, the magnitude will increase with particle velocity.

It follows, that for a given change in ϕ the mean settling velocity will change by an amount depending upon the distribution of settling velocities at the time of the change. This distribution can probably be characterized by the initial settling velocity and the standard deviation, σ , of the velocities. Therefore, changes in \bar{w} will be a function of \bar{w} itself, σ and the volume concentration of particles.

For flocculation no theories existed which were suitable for this research. The previous theories were all confined to predicting the rate of interparticle contacts, while ignoring the effect on settling velocities. Furthermore, they

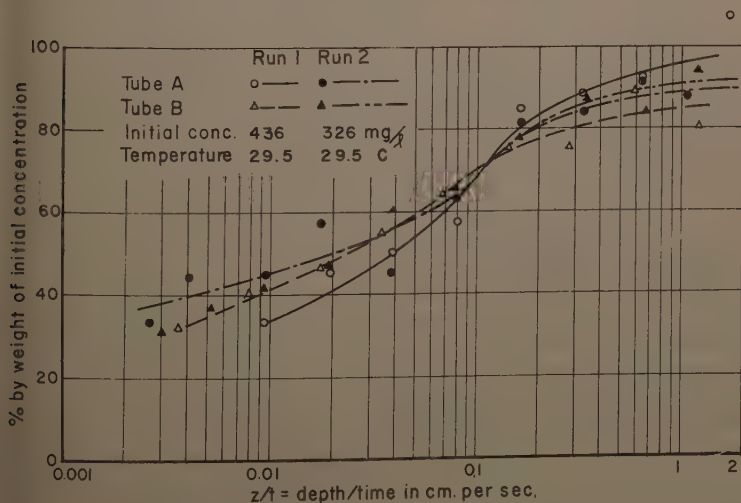


Fig. 4. Duplicate pipette analysis Pasadena sewage.

dealt with suspensions having velocities of only a few classes. Therefore kinetics of flocculation during quiescent settling was studied theoretically for the purpose of discovering how flocculation produces changes in \bar{w} . Because of the number of unknowns the theory could not be completed, but was carried far enough to show that the change in \bar{w} at a point depends on the following properties of the suspension at the same point.

- (1) The volume concentration of particles.
- (2) The mean settling velocity of particles.
- (3) The standard deviation of the settling velocities of the particles.
- (4) The fraction of inter-particle contacts that result in the union of particles.

Item (4) is simply a means of accounting for the surface chemistry of the particles.

The value of \bar{w} at a point will also change in free settling. That is, in the absence of flocculation and hindered settling it is easily shown⁽¹²⁾ that

$$\frac{\partial(\bar{w} \phi)}{\partial t} + \frac{\partial}{\partial z} (\sigma^2 \phi + \bar{w}^2 \phi) = 0$$

Thus, the change in \bar{w} depends on \bar{w} itself and on the standard deviation of settling velocities.

It appears, therefore, that the first three of the items listed above in connection with flocculation are also the most important items in connection with free and hindered settling. Consequently, an experimental study of quiescent settling should involve the determination of at least \bar{w} , σ , and concentration. The determination of the first of these, \bar{w} , is described in part which follows.

d. Measurement of \bar{w}

Since a measurement of \bar{w} has not been reported in the literature, some time was spent in devising a method. Finally, it was decided to use an experiment based on a suggestion by Camp.⁽¹³⁾ The suspension was allowed to settle quiescently in a vertical tube as in a pipette analysis. During the settling a series of samples were taken at each of several depths, and these samples were analyzed for particle concentration. This experiment was given the name multiple-depth pipette analysis or, simply multiple-depth analysis. Some very simple equations will show how such an analysis can be used to measure \bar{w} .

For the purpose of discussion, consider the hypothetical settling tube shown on the left in Fig. 5. All horizontal cross-sections of the tube are of uniform area and at any time, the particle concentration, ϕ , over any such cross-section is constant. Hence, ϕ is a function of z , the depth below the top of the suspension, and t , the time after beginning of settling.

At the beginning of the test ($t = 0$) pipette samples are withdrawn simultaneously at many depths. The results are plotted as the curve, $t = 0$, in the diagram on the right of Fig. 5. At $t = T_1$, another set of samples is withdrawn and the results are plotted as the curve, $t = T_1$. The curves, $t = T_2$ and $t = T_3$, represent similar operations at times T_2 and T_3 , respectively.

Physically speaking, each of these curves is a profile of the concentration at some given instant. When all the profiles from an analysis are put in the diagram, it can be called the concentration profile diagram. Mathematically

taking, this diagram is a plot of $\phi(z,t)$ as a function of z with t as a parameter. At any stage in the discussion, it is possible to give the diagram a physical or mathematical interpretation, depending on which is more illuminating. The concentration profile diagram is used in conjunction with the continuity equation for the calculation of $\bar{w}(z,t)$. First, it is noted that for the settling case, Eq. (8) is reduced to the following:

$$\frac{\partial \phi}{\partial t} + \frac{\partial (\bar{w}\phi)}{\partial z} = 0 \quad (17)$$

Integrating this equation with respect to z gives

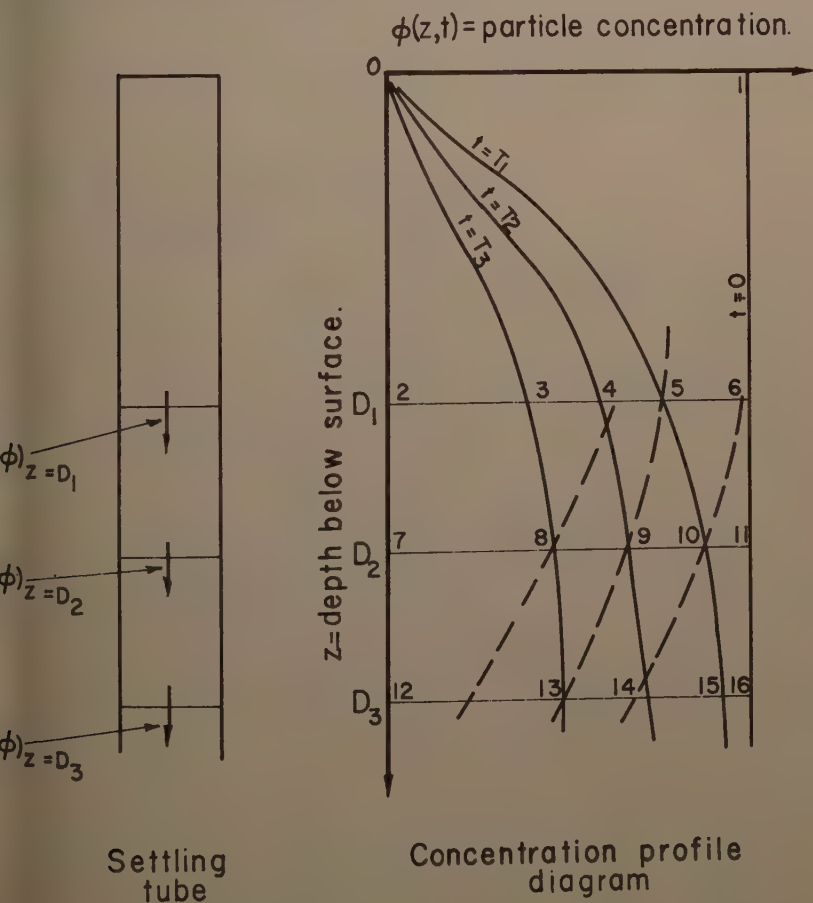


Fig. 5. Analysis of data from multiple-depth pipette analysis.

$$\begin{aligned}
 (\bar{w} \phi)_{z=D} &= - \int_0^D \frac{\partial \phi}{\partial t} dz \\
 &= - \frac{\partial}{\partial t} \int_0^D \phi dz
 \end{aligned}$$

Eq. (18) shows how \bar{w} can be calculated. A value of D is selected; for example, consider D_1 in Fig. 5. The areas 025, 024, and 023 are calculated; the values are plotted against t . If this is done for a sufficient number of files, the result will be a smooth curve giving the area under the ϕ profile above D_1 as a function of time. The slope of the curve is precisely the hand side of Eq. (18), and hence is equal to $\bar{w} \phi$ at $z = D_1$. The process is repeated for any value of D .

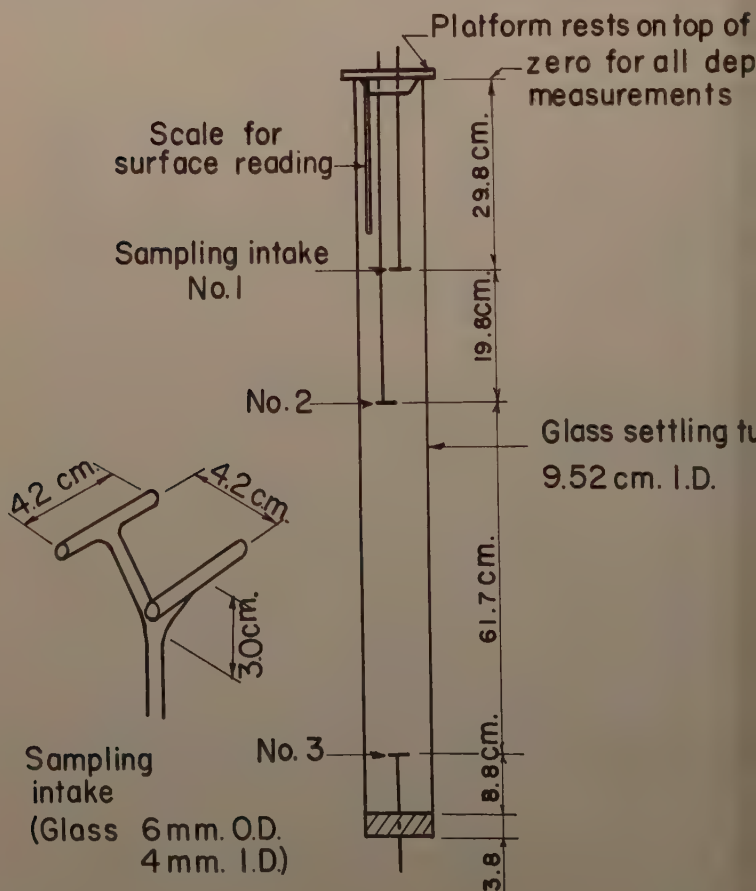


Fig. 6. Multiple-depth settling tube.

In order to test the feasibility of this approach to measuring \bar{w} , a pilot experiment was performed using the apparatus shown in Fig. 6. Sampling intakes, or pipettes, were located at three depths as shown. Samples could be drawn through the top two by means of a vacuum and through the bottom one by means of gravity flow.

The suspension to be tested was thoroughly mixed in a separate container and then poured quickly into the tube. Since the upper two intakes are attached to a platform which rests freely on the tube, these intakes could be put in place immediately after pouring. After they were in place, the elevation of the surface was read on the scale shown and the withdrawing of samples began. The surface elevation was read again after the withdrawal of each sample. With these readings it was possible to allow for the lowering of the surface in estimating the distance settled by particles. The estimated distance was used as z for each sample. Time, t , was measured from the end of pouring.

Table 3

Multiple-Depth Analysis of a Suspension of

Clay and Alum in Water

Time After Start of Settling	Intake	Depth of Intake z	Volume of Sample z/t	Temp. of Sample	Weight of Particles in Sample	% of Initial Conc.
sec.	No.	cm	cm/sec	ml.	$^{\circ}\text{C}$	mg
0					29.5*	
60	2	42.6	0.71	28		17.7
90	3	104.3	1.2	27		17.3
120	1	21.7	0.18	32 1/2		20.3
180	3	103.2	0.57	33		21.6
245	2	39.7	0.16	28		18.1
300	1	18.4	0.062	37		23.6
360	3	101.4	0.28	31 1/2		20.0
480	1	17.1	0.036	28		16.7
600	2	37.5	0.063	31 1/2	30.3	21.4
720	3	99.2	0.14	32	30.3	22.9
960	1	15.2	0.016	22		8.1
1200	2	35.6	0.030	25		11.2
1440	3	97.3	0.068	36		12.9
1920	1	13.2	0.069	27		7.7
2400	2	33.7	0.014	31		9.0
2880	3	95.4	0.033	29	31.5	7.2
3840	1	11.1	0.0029	31	32.0	5.1
4860	2	31.6	0.0065	32	32.3	5.0
5760	3	93.3	0.016	27	32.8	4.6
7740	1	8.9	0.0012	31	33.4	2.7
10100	2	29.4	0.0029	40	34.0	4.1
14100	3	91.1	0.0065	26	34.8	1.2

* Measured in mixer

Initial Concentration = 655 mg/l

Alum Concentration = 25 mg./l

This apparatus was used for the multiple depth analysis of a suspended bentonite clay in Pasadena tap water and alum. $(Al_2(SO_4)_3 \cdot 18 H_2O)$. initial concentration of clay was 655 mg. per liter, while that of alum was 10 mg. per liter. The results of the experiment are given in Table 3 and in the concentration profile diagram of Fig. 7. Since samples were taken only at three depths some intermediate plotting and interpolation were necessary to produce the profiles.

Once the profiles were obtained, the mean settling velocity $\bar{w}(z,t)$ was calculated in the manner described above. Values for three values of z are given in Fig. 8. It is possible to calculate the order of magnitude of

$\phi(z,t) = \% \text{ by weight of initial concentration}$

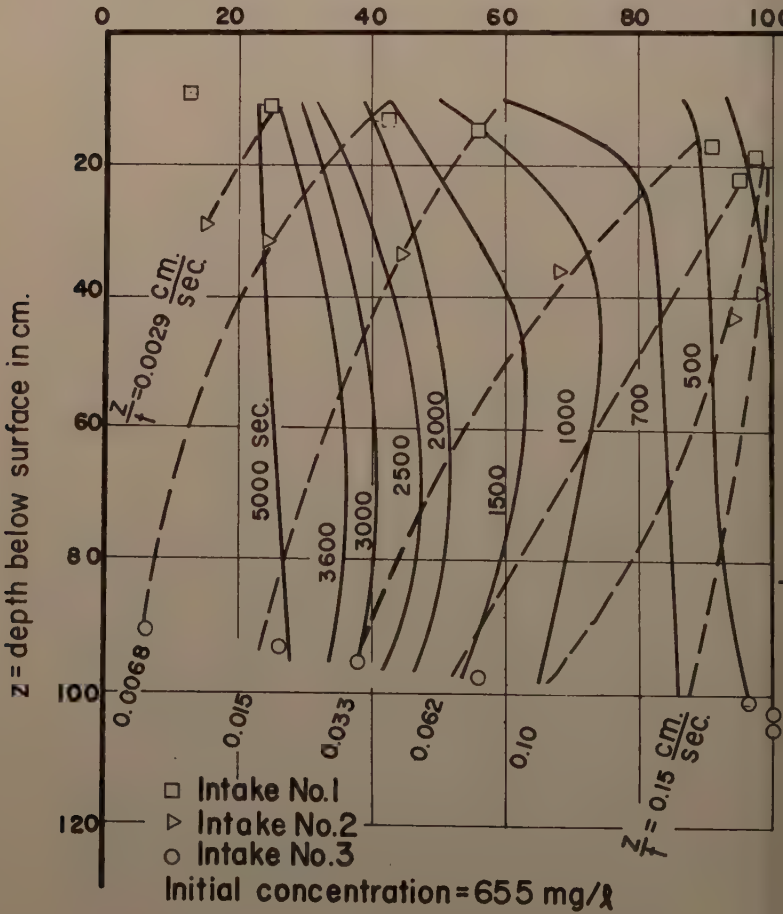


Fig. 7. Concentration profile diagram, Bentonite clay and alum in water.

reculation effects on local settling. At a depth of 90 cm. the mean settling velocity increases from 0.021 cm. per second at $t = 200$ seconds to 0.037 at 500 seconds. This change represents an increase of 75 per cent and an average rate of increase of 0.000053 cm. per sec. per sec. Furthermore, the change in concentration during the same time was only 3 per cent. The effect of particles settling out of suspension must have been small. Therefore, the change of concentration is primarily due to flocculation.

It is possible to tell directly that flocculation is causing the particles to speed up by looking at the dashed lines constant z/t in Fig. 7. The physical significance of these lines is best explained in the following manner. An observer starts at the surface of the suspension at $t = 0$ and descends through the suspension at a constant velocity. The concentration that he observes at various depths is given by a line of z/t equal to the velocity.

It can be shown (12) that when neither hindered settling nor flocculation occurs, these lines are straight and parallel to the z -axis. If hindered settling slows the particles down more than flocculation speeds them up, the lines slope away from the z -axis as depth increases. If the converse is true, the lines slope toward the z -axis as depth increases. For Fig. 7, then, it is seen that flocculation is the predominant effect.

The maximum mean settling velocity for $z = 90$ cm. occurs at 700 seconds. This value of 0.056 cm. per second is 2.8 times the mean settling velocity at $t = 200$ seconds. The average rate of change between $t = 200$ seconds and 700 seconds was 0.00007 cm. per sec. per sec.

A maximum value of \bar{w} occurs at all depths in the settling tube. After this maximum, the mean settling velocity decreases. Particles are settling out of suspension, and the loss of faster particles offsets the effect of flocculation.

While this work was being done, Fitch⁽¹⁴⁾ presented the results of a multiple-depth experiment on a suspension of whiting (CaCO_3) and ferrisulphate [$\text{Fe}_2(\text{SO}_4)_3 \cdot 9\text{H}_2\text{O}$] in water. The concentration of CaCO_3 was 400 parts per million (ppm), while the concentration of ferrisulphate was 15 ppm. This suspension

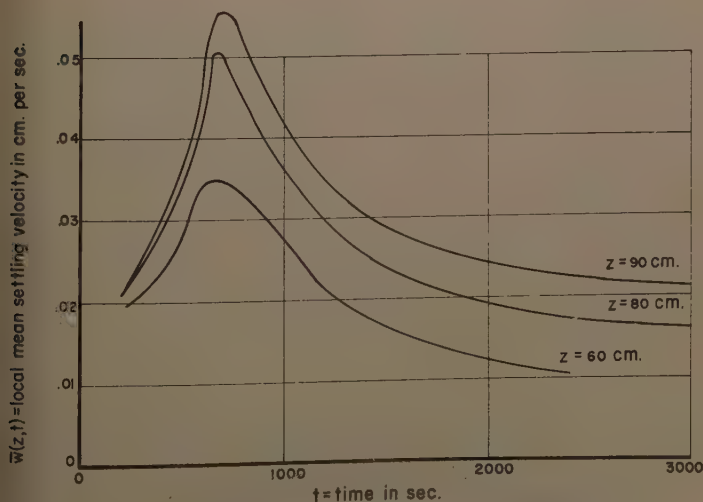


Fig. 8. Local mean settling velocity as a function of time. Bentonite clay and alum in water.

was allowed to settle in a tube seven feet deep and 5-1/2 inches in internal diameter. During the settling, samples were taken at seven depths by means of veterinary hypodermic needles which passed through the walls of the tube. For temperature control, the outside of the tube was covered with insulation one inch thick.

Fitch's results are presented as a concentration profile diagram in Figure 9. The solid curves are concentration profiles or lines of constant time t .

$$\phi(z,t) = \text{concentration in ppm.}$$

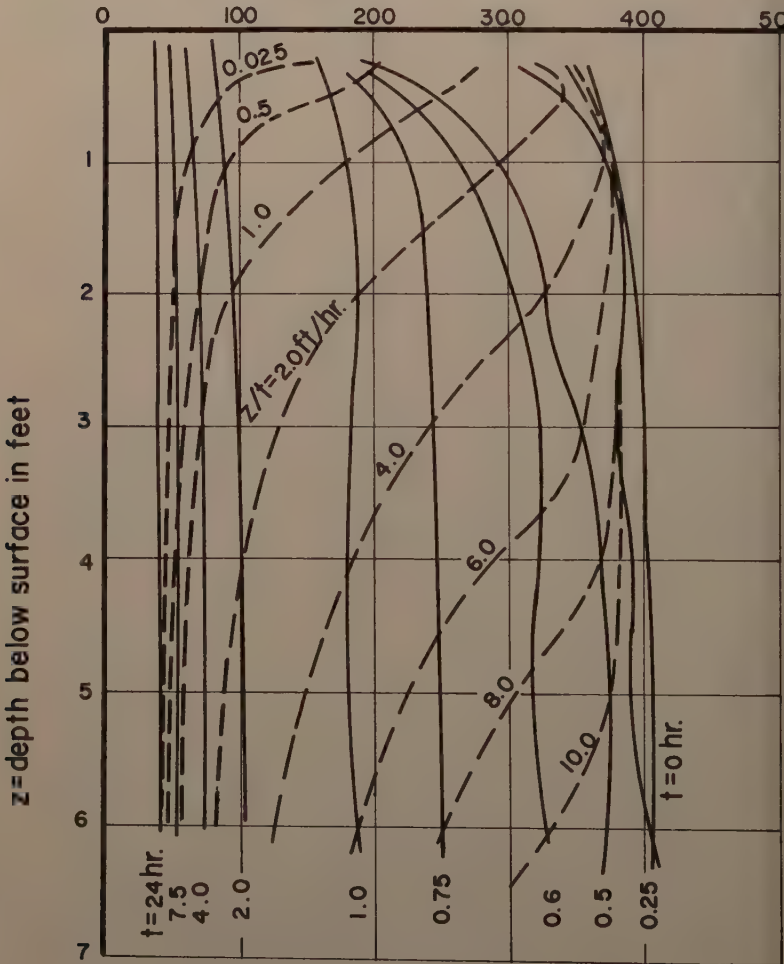


Fig. 9. Concentration profile diagram, CaCO_3 and $\text{Fe}_2(\text{SO}_4)_3 \cdot 9\text{H}_2\text{O}$ in water
(Based on data by Fitch)

ashed curves are curves of constant z/t . For $t > 0.25$ hours and $z > 2$ the constant z/t lines slope toward the z -axis, showing the effect of

Following the method described above, the local mean settling velocity was calculated. The mean velocities for $z = 3$ feet and $z = 6$ feet are shown in Fig. 10. At $z = 6$ feet, the mean settling velocity increased from 0.013 cm. per sec. at $t = 0.25$ hours to 0.035 cm. per sec. at $t = 0.5$ hours. The average rate of change during this time was 0.000025 cm. per sec. per sec. At $t = 0.5$ hours, the concentration at $z = 6$ feet is 94 per cent of the initial concentration. Subsequently, up to this time the change in \bar{w} is caused primarily by floccu-

n. It is interesting to compare the change in \bar{w} for whiting and ferrisul with the change for clay and alum. The comparison is valid only for those stages of settling for which the concentration has not decreased significantly. For whiting and ferrisul, the rate of change during this stage of settling was 0.000025 cm. per sec. per sec. For clay and alum, the corresponding rate of change was 0.000053 cm. per sec. per sec. The values differ by a factor of

The important practical conclusion can be drawn from these determinations. That is that the effect of flocculation increases with depth from the surface. This is shown by the curves for \bar{w} in Figs. 8 and 10. If the settling in either case had been deeper, the peak values of \bar{w} would probably have increased with depth until a limiting value was reached.

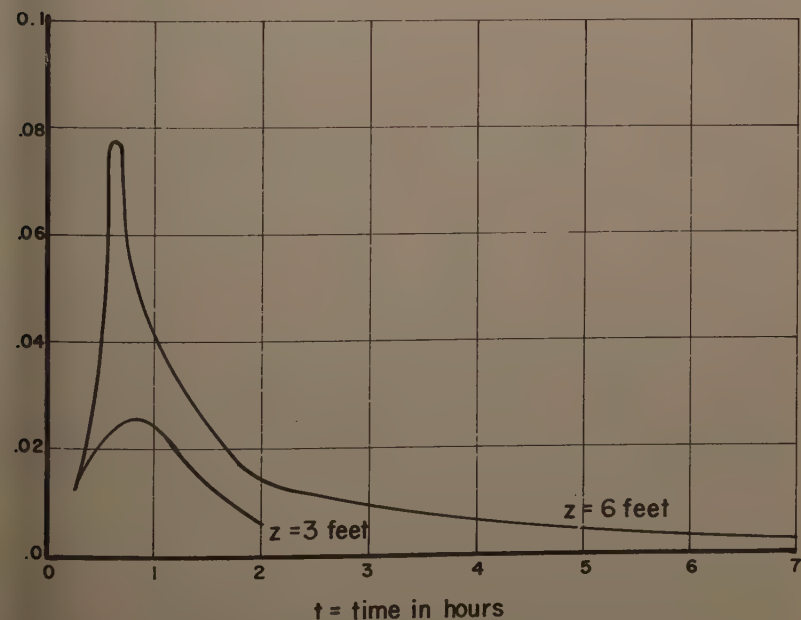


Fig. 10. Local mean settling velocity as a function of time (based on data by Fitch).

e. Proposals for Future Work

The next logical step in this work is to design an experiment in which it is possible to measure \bar{w} , σ , and particle concentration. For such an experiment, it is desirable to have a tube about ten feet deep with 7 or 8 sampling intakes. The sampling intakes should enter through the walls of the tube to cause a minimum obstruction to settling particles. The tube should be housed in a water jacket for temperature control, but both jacket and tube should have windows for visual observations.

The cross-sectional area of this tube should be at least equal to the area of a circle twelve inches in diameter. Because of this large area, the sample withdrawn during settling can be as large as 500 ml. Then each sample can be allowed to settle in a 500 ml. graduate for a simple pipette analysis.

Five or six 25 ml. samples from each 500 ml. graduate should be taken to give a settling velocity distribution for each 500 ml. sample. From this distribution it is possible to calculate the mean and standard deviation of settling velocities for each sample. Of course, flocculation may occur in the graduate, but the effect is small compared to that in the tube.

The same 500 ml. samples can be used for a determination of ϕ and particle concentration by volume. From the former the concentration profile can be drawn. The values of \bar{w} can then be calculated as shown above and compared against the value obtained from the simple pipette analysis.

Before this more elaborate multiple depth-analysis could be performed, the experiments on settling analysis were discontinued, and, as yet, no revision has been made for their continuance. It is hoped that someone will find the problem of sufficient interest to continue the work. If so, it will be economical to modify the tube to allow for the study of turbulence.

It is a simple matter to extend the proposed multiple-depth analysis to include effects of turbulence. Rouse⁽¹⁵⁾ and Dobbins⁽³⁾ have shown that a uniform field of turbulence can be created by placing a vibrating grid in the settling tube. Furthermore, Dobbins devised a method for controlling the resuspension of particle resuspension at the bottom of the tube. If these additions are incorporated into the apparatus proposed above, it will be possible to study flocculation in a turbulent fluid.

4. The Multiple-Depth Analysis in Approximate Calculations of Removal

a. Short-Cut Methods in Calculating Removal

There are times when the flow in a channel is such that $V = W = 0$ and is independent of depth. Then, the settling of particles is similar to settling in a vertical tube which is moving at a velocity U in the x -direction. The time t for particles to settle in the tube at time, t , corresponds to profiles in the channel at time, t . It follows that these profiles can be used directly to calculate removal.

b. "Ideal" Settling Tanks

Part (a) of Fig. 11 shows the settling zone of a rectangular settling tank of length L , width B , and depth D . By making certain simplifying assumptions about such a basin, Camp⁽¹³⁾ devised the concept of an "ideal tank". In the nomenclature of Section 2, these assumptions can be stated as follows:

$$U(x,y,z) = \text{constant}$$

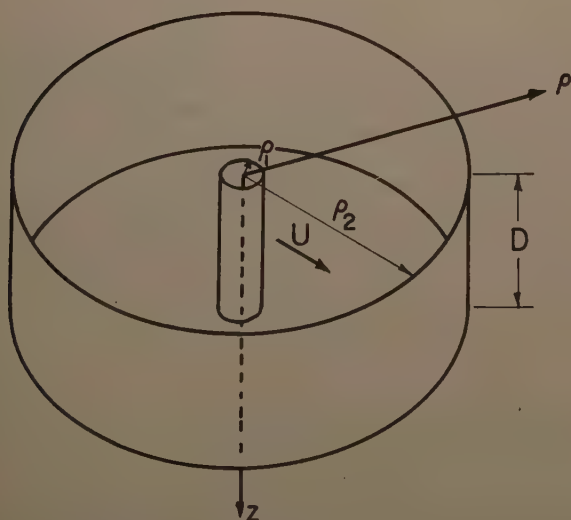
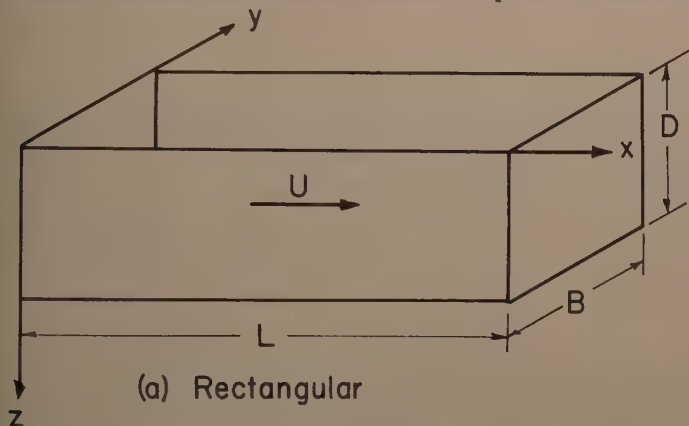
$$V(x,y,z) = W(x,y,z) = 0$$

$$e_x(x,y,z) = e_y(x,y,z) = e_z(x,y,z) = 0$$

$$f_i(0,y,z) = \text{constant for each value of } i$$

No resuspension at the bottom of the tank

Although these assumptions are sufficient to describe the tank, Camp also assumed that all particles in the tank settle without flocculation or hindrance. Since this last assumption is unnecessary, in this paper the term ideal tank will refer only to the assumptions about the tank. In place of the assumption



(b) Radial flow

Fig.II. Settling zones of ideal tanks.

about the settling, the results of a multiple-depth analysis can be used. This approach is based on the fact that in an ideal tank settling is identical to that in a vertical tube, and concentration profiles at distance x in the tank correspond to profiles at time t in the tube. If the concentration profiles from the tube are used, the removal in the ideal tank will be given by

$$R(x) = B U \int_0^D [\phi(z, 0) - \phi(z, t)] dz$$

where

$$t = \frac{x}{U}$$

and $\phi(z, t)$ represents concentration in the tube.

In settling tanks, the removal ratio, $r(x)$, is more important than the removal itself. The former is defined as the fraction of particles entering the tank that settle out in distance x . Hence,

$$r(x) = \frac{R(x)}{\int_0^D \phi(z, 0) dz} = \frac{\int_0^D [\phi(z, 0) - \phi(z, t)] dz}{\int_0^D \phi(z, 0) dz}$$

The integrals in Eqs. (19) and (20) can be represented on the concentration profile diagram by areas. Consider, for example, the diagram shown in Fig. 5. For a tank of depth D_1 , and a distance of UT_1 , the removal is represented by area 0561 while the removal ratio is represented by

$$r(x) = \frac{\text{area } 0561}{\text{area } 0261}$$

A similar approach can be used for radial flow tank. Part (b) of Fig. 6 shows what might be considered an "ideal radial flow tank". Fluid enters the tank through the surface of a cylinder of radius ρ_1 , centered along the vertical axis of the tank. It then flows radially and horizontally to the cylindrical walls of the tank. These walls have a radius of ρ_2 .

The flow conditions for this tank are assumed to be

$$U = \frac{b}{\rho} \quad (b = \text{a constant})$$

$$V = W = 0$$

$$e_x = e_y = e_z = 0$$

The initial conditions are assumed to be

$$f_i = \text{constant at } \rho = \rho_1 \text{ for each value of } i,$$

while the bottom boundary condition is the same as that for Camp's ideal tank.

Settling in this radial tank is identical to settling in a vertical tube. Concentration profiles at radius ρ in the tank correspond to profiles at

$$t = \frac{\rho_1^2 - \rho^2}{2b}$$

the tube. It follows that removal in the tank in radial distance, ρ , is given

$$\begin{aligned} R(\rho) &= 2\pi \rho_1 \int_0^D U(\rho_1) \phi(\rho_1, z) dz - 2\pi \rho \int_0^D U(\rho) \phi(\rho, z) dz \\ &= 2\pi b \int_0^D [\phi(z, 0) - \phi(z, t)] dz \end{aligned} \quad (22)$$

where t is given by Eq. (21). Eq. (22) shows that removal in the radial tank is presented by an area on the concentration profile diagram just as it is for the rectangular tank.

It is suggested that the combination of assuming an ideal tank and performing a multiple-depth analysis on the suspension will give a first approximation to the behavior of the tank. This approximation will often be better than making an elaborate study of the hydraulics of the tank while ignoring the properties of the suspension.

Ideal Tanks with Turbulence

For a better approximation to a rectangular or cylindrical basin, it is possible to add turbulence to the ideal tank. If the turbulent diffusion is assumed to be

$$\begin{aligned} e_x &= e_y = 0 \\ e_z &= \text{constant,} \end{aligned}$$

the settling in the tank will be equivalent to settling in a tube into which uniform turbulence is introduced.

Dobbins⁽³⁾ and Camp⁽¹⁶⁾ have assumed this sort of tank in a study of the effect of turbulence in retarding settling. Camp assumed that the particles settled without flocculation or hindrance. However, this assumption is unnecessary, since it is a simple matter to put the actual suspension in a settling tank as deep as the tank and introduce turbulence by means of a vibrating grid.

The combination of assuming an ideal tank and performing a turbulent multiple-depth analysis on the suspension is suggested as a second approximation to the behavior in a rectangular basin. For better approximations, it is necessary to obtain enough information for the step calculation outlined in Section 2.

Overflow Rate and Detention Time in Ideal Tanks

In the technical literature, one finds a great deal of discussion about whether a settling tank should be designed on the basis of overflow rate or detention time. The discussion about the merits of these two approaches is inconclusive, because it disregards the properties of the suspension. It will now be demonstrated that whether overflow rate or detention time should be used depends upon the nature of the suspension. Indeed, for a single tank, the removal ratio may be closely related to overflow rate for one suspension, to detention time for a second, and to neither for a third.

The detention is simply the average time that an element of fluid remains in the tank. Let it be called T . For the rectangular ideal tank,

$$T = \frac{L}{U}$$

while for the radial,

$$T = \frac{\rho_2^2 - \rho_1^2}{2b}$$

Overflow rate, on the other hand, is obtained by dividing the volume flow through the tank by the horizontal area of the tank. Let it be w_o . rectangular basin,

$$w_o = \frac{UBD}{LB} = \frac{D}{T}$$

while for the radial,

$$w_o = \frac{2\pi \rho_2 \frac{b}{\rho_2} D}{\pi(\rho_2^2 - \rho_1^2)} = \frac{D}{T}$$

Eqs. (25) and (26) show that for constant flow rate and overflow rate detention time varies with depth. Thus, if the effectiveness of the tank depends on overflow rate, $r(x)$ will not vary with D as long as D/T is constant. Conversely, if the effectiveness depends on detention time, $r(x)$ will not vary with D as long as T is constant.

To study the problem experimentally, a multiple-depth analysis is performed on the suspension concerned. The settling tube for this analysis should be as deep as the deepest possible tank to be considered. The results from the analysis are plotted on a concentration profile diagram as shown in Fig. 5.

In this figure, let D_1 , D_2 and D_3 represent the depths of three tanks considered. The detention times for these tanks are T_1 , T_2 and T_3 , respectively. The tanks all have the same overflow rate. Therefore,

$$\frac{D_1}{T_1} = \frac{D_2}{T_2} = \frac{D_3}{T_3}$$

and the dashed line 5, 9, 13 is a line of constant z/t . The other dashed lines are also lines of constant z/t .

If the removal ratio is to depend on overflow rate alone, then

$$\frac{\text{area } 0561}{\text{area } 0261} = \frac{\text{area } 0, 9, 11, 1}{\text{area } 0, 7, 11, 1} = \frac{\text{area } 0, 13, 16, 1}{\text{area } 0, 12, 16, 1}$$

Furthermore, this type of relationship must hold for any overflow rate for any dashed curve of constant z/t . It can be shown⁽¹²⁾ that Eq. (27) holds for arbitrary overflow rate if and only if the lines of constant z/t are straight and parallel to the z -axis.

If, on the other hand, removal ratio depends only on detention time, the equation of the form

$$\frac{\text{area } 0, 5, 6, 1}{\text{area } 0, 2, 6, 1} = \frac{\text{area } 0, 10, 11, 1}{\text{area } 0, 7, 11, 1} = \frac{\text{area } 0, 15, 16, 1}{\text{area } 0, 12, 16, 1}$$

hold for each profile. Eq. (28) will hold if and only if the profiles are straight lines parallel to the z -axis.

It follows that for quiescent settling in ideal tanks the problem of overflow and detention time is determined by the pattern of the profile diagram. This pattern, in turn, is determined by the settling properties of the suspension.

As an example of a suspension for which detention time is more important, consider the data obtained by Fitch and plotted in Fig. 9 above. The profiles are almost straight and vertical. Thus, for any tank up to six feet in depth, removal is affected more by detention time than by overflow rate. Calculations by Fitch substantiate this conclusion.

With only these curves at hand, one might ask the following question. What would be the situation when this suspension settles in a rectangular tank ten feet deep? To answer this question, it is necessary to try to sketch in the profiles of constant z/t between the depths of six and ten feet and draw the profiles from these. After a few trials, it becomes evident that the profiles are likely to change their shape drastically. Hence, removal will still depend primarily on detention time.

Detention Time and Overflow Ratio—General

The data by Fitch shows that when flocculation occurs in an ideal tank the removal ratio can be more closely related to detention time. Furthermore, in reference (12) it is shown that flocculation has a tendency to cause the vertical straight profiles which indicate the dependence on detention time.

The reason is quite simple. When particles settle without flocculation, the concentration at a given time will normally increase with depth. Hence, the profile slopes away from the z -axis. With flocculation, however, the faster particles settling through the slower ones gather up the slower ones and drag them out of suspension. This tends to cause the concentration at any given time to decrease with depth. The combined result can cause the concentration to be independent of depth, i.e., a straight profile parallel to the z -axis.

However, Camp⁽¹³⁾ has shown that when the particles settle without flocculation or hindered settling, removal depends on overflow rate alone. Between the suspensions discussed by Camp and those discussed by Fitch are suspensions for which both overflow rate and detention time are significant. The diagram of Fig. 7 represents an example of this last situation.

When turbulence during settling is to be considered, the studies of overflow rate and detention time should be based on results from turbulent settling analysis. As yet, the available experimental results of this type are too meagre to permit any conclusions.

Sediment Deposition in Canals

In the design of canals, it is necessary to consider minimum velocities for siltation. If the latter cannot be maintained, it is desirable to know where in the canal the silt will be deposited. Some useful information about this problem can be obtained from a turbulent multiple-depth analysis.

This analysis should be performed in a tube as deep as the canal. Inside the tube, the level and vertical distribution of turbulence should be as close as possible to that in the canal. The vertical distribution of turbulence can be obtained by varying the grid spacing and bar size along the vibrating grid. The turbulence level can be obtained by adjusting the frequency of the vibrations.

The tube can be used to find out what velocity is necessary for keeping particles in suspension. By letting the suspension settle in the tube various grid frequencies, it will be possible to decide upon the turbulence that keeps deposition to a reasonably small amount. The velocity in the tube which produces the turbulence is the necessary velocity in the canal.

For each grid frequency, the test should be run for a considerable time. After only a short time in the tube, the particles of a suspension may be reasonably well suspended at a given level of turbulence. However, if sedimentation occurs, the particles may increase the settling velocities and begin to settle out. This process may not be evident for thirty minutes or an hour.

If a minimum velocity cannot be maintained in the canal, the resulting analysis will show the pattern of silt deposition. For if the flow in the canal is fairly uniform, profiles at distance x in the canal correspond to profiles in the tube. Removal is calculated in the manner described for the ideal case.

When the level of turbulence in the canal is sufficiently low, the results of a quiescent multiple-depth analysis may give some information about the initial concentration. Fig. 12 represents the results of a quiescent analysis of bentonite.

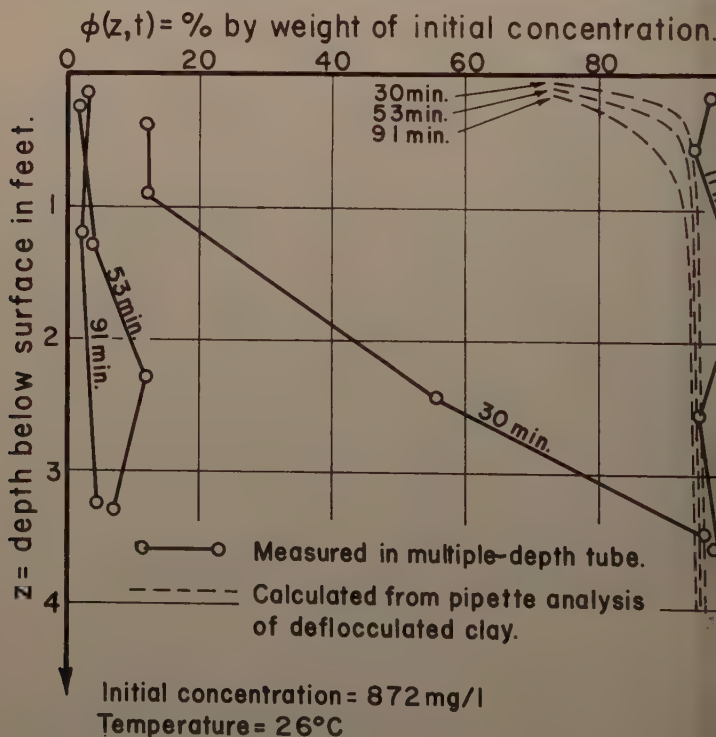


Fig. 12. Concentration profiles for bentonite clay in Pasadena tap water.

in Pasadena tap water. The analysis was performed in a lucite tube with an internal diameter of 3-3/4 inches and a depth of 4 feet. Thus the settling might correspond, roughly, to settling during tranquil flow in an irrigation canal 4 feet deep.

In preparing the suspension, the clay was blended thoroughly with 0.5 liters of water. This blended mixture was then shaken with enough water to bring the final clay concentration to 872 mg per liter. After two minutes of vigorous shaking the suspension was poured into the tube. This mixing might be similar to that which the suspension would receive in passing through a diversions works into the canal.

During the settling, samples were taken at four depths. The concentrations of these samples were used to plot the profiles shown as solid lines in Fig. 12. For the purpose of comparison, the settling velocity distribution of the original clay particles was obtained by means of a pipette analysis of a suspension of clay and deflocculating agent in distilled water. From the settling distribution, the dotted profiles of Fig. 12 were determined. These profile show how the particles would settle without flocculation.

On the basis of the dotted profiles, one would expect that the clay would be carried in suspension indefinitely. Therefore it would pass through the canal. In fact, however, most of the clay would have settled to the bottom of the canal in one hour.

5. CONCLUSIONS

The principal conclusions based on the research are as follows:

- (1) In order to predict the settling of particles in a flowing suspension, it is necessary to know the properties of the flow and the settling properties of the individual suspension. In general, it is best to determine both by direct measurement. If a choice must be made, it will often be better to determine the properties of the flow by calculation or reasonable assumption, and to measure the properties of the suspension.
- (2) In the initial stages of studying a suspension, it is profitable to obtain a settling velocity distribution by means of a pipette analysis or comparable experiment. When flocculation and hindered settling are negligible this distribution can be used in an estimate of removal; when they are not, the results of the analysis are still useful in comparing suspensions.
- (3) A multiple-depth pipette analysis can be used to study the effect of hindrance, flocculation, and turbulence on the settling. For quiescent settling, the analysis should include measurements of the mean settling velocity, the standard deviation of settling velocities and the particle concentration at various depths and times. For turbulent settling the analysis should include these three measurements plus any additional measurements related to the turbulence.
- (4) Whenever the settling in the flowing suspension can be approximated by settling in a quiescent or turbulent settling tube, the concentration profiles from a multiple-depth analysis can be used to calculate removal.
- (5) When a suspension flows through a settling tank, the removal of particles may depend upon detention time, overflow rate, or both. Whether one or the other is more important depends primarily upon the settling properties of the suspension. For quiescent settling in ideal tanks, a

multiple-depth settling analysis will indicate the relative import of overflow rate and detention time.

- (6) Settling in a hydraulic model will not be similar to settling in the type unless a suspension with scaled settling properties is used in the model. When this is not possible, two approaches are available. In the first, the model is used to predict the properties of the flow, and separate experiments are used to measure the properties of the suspensions. These properties are used to calculate what the removal will be in the predicted prototype flow. In the other approach, the prototype suspension settles in the model. The settling properties of the suspension are studied separately and these properties are considered in using the model results to predict prototype results.

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SUGGESTED LEGISLATION ON FLOOD PLAIN REGULATION^a

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ABSTRACT

Adequate statutory authority for city, county and regional planning is lacking in many areas. Such authority must be provided if communities are to regulate the use of flood plains in a manner that will limit and tend to reduce the damage from floods. This problem concerns all levels of government.

SYNOPSIS

The rapid growth of urban and suburban areas has caused rapid expansion of developments into the flood plains of this country. The increase in flood damageable improvements has outstripped all efforts to provide flood control and flood protection.

An increasing awareness of the necessity for limiting the flood damage potential is needed at the local level. The preparation of plans for the wise use of flood plains, and the adoption of regulations to ensure such use are required for proper and wholesome community development.

Not all states provide adequate statutory authority for city, county and regional planning. Such authority should be provided. In addition, states should provide controls over construction of dams, levees and encroachments on floodways, should collect basic flood data, and should advise local planning agencies regarding flood plain regulation problems.

Federal agencies should expand basic data collection programs, observe local flood plain regulations, and require enactment of appropriate flood plain regulations by local cooperating parties as a prerequisite to qualifying for certain types of federal programs.

^aNote: Discussion open until May 1, 1960. To extend the closing date one month, a written request must be filed with the Executive Secretary, ASCE. Paper 2315 is part of the copyrighted Journal of the Hydraulics Division, Proceedings of the American Society of Civil Engineers, Vol. 85, No. HY 12, December, 1959. Presented at the July 1959 Hydraulics Division Conference in Fort Collins, Col.

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INTRODUCTION

The rapid growth of urban and suburban areas since World War II has been an outstanding feature of the economic boom that has taken place during the post war years. This growth has been stimulated as a result of the substantial amount of building during the depression years of the thirties, the war time restrictions of the early forties, the very rapid increase in population following the war, and the favorable employment and earning levels that have existed during the past fourteen years.

The land requirements to accommodate this growth have been greatly enlarged by the trend toward sprawling one story houses, and by septic tank and sanitary system drainage requirements in suburban locations. These trends lead to lots two to three times larger than those that were considered adequate forty or fifty years ago. The greater land requirements have placed a premium on areas of flat or gently sloping topography which can be developed most economically. Undeveloped flood plains fulfill these requirements, particularly in the regions of the country where topography tends to be rolling or hilly.

For these reasons thousands of new homes have been constructed in areas subject to flooding. In many instances subdivisions have been developed without full knowledge of the dangers involved, and the developer has either ignored these dangers or considered them so remote that little consideration was given to be given to them. In addition, the planning commissions or other bodies with responsibility for approving new developments frequently have paid little attention to flood danger as an element to be considered in appraising the suitability of a site for residential, business or industrial development. In other instances the approving bodies have recognized the danger but have not known how to deal with it, except to bear with it and hope for the best.

Industry is a bit more cautious than many other developers in selecting sites for new developments or in expanding old installations and will try to avoid flood dangers or will protect against them. However, there are numerous instances throughout the country where such dangers have not been avoided. Many times industries have exposed themselves to possible damage because of other considerations that were deemed to be more pressing or more significant to their operations.

In addition to being exposed to flood danger, many developments also encroach on the floodway, reduce the efficiency and capacity of the floodway and consequently increase flood levels and damage to themselves and other properties.

In some areas, as flood heights have been reduced by flood control projects, levees and the danger of flooding diminished, developments have encroached farther into the flood plains and again have become subject to damage from lesser floods. Such use of the flood plains tends to nullify many of the benefits from flood control projects.

It has been pointed out by White⁽¹⁾ that although the Federal government has spent over \$4 billion for flood protection since 1936 there has been a decrease in the mean annual flood loss. On the contrary, flood damage has tended to increase significantly. He attributes much of this increase to the great increase in use of the flood plains.

If the flood damage potential is growing and outstripping the relief being obtained from flood control and protection projects, why are construction measures not being taken to more effectively control the improvements? create and increase the damage potential? The answer is in the lack of

recognition of the problem at the local level, lack of adequate legislative authority to deal with the problem, failure to use existing authority, and lack of adequate knowledge for dealing with the technical phases of the problem.

Local Regulations

If the problem of reducing the flood damage potential in our flood plains is considered to be one of controlling the unwise actions of people, its solution will involve the use of the police powers of the states and the extension of those powers to local governing officials. The use of police powers is still a realm that is reserved for the states and local governments under the terms of the Constitution. It therefore devolves upon local officials, who are in most direct contact with the people involved, to perform the primary policing functions.

There has been a growing awareness of the responsibility of local units of government to plan for the wise use of flood plains. During the past five years an increasing number of communities have included flood plains as special use areas in their community plans. However, the number of such communities is pitifully small compared to the total number of communities engaged in planning.

The lack of adequate statutory authority to permit proper planning is a factor that is retarding progress in some areas. The use of improper techniques or ineffective regulations provides inadequate controls in other areas.

The legislatures of 45 states (including Alaska and Hawaii) and the District of Columbia have expressly authorized community planning. Community-type planning on a regional basis has been authorized by 31 states.

Although planning enabling statutes have been enacted in 45 states, in 14 states there is no express statutory authority granted to extend planning activities beyond the corporate limits of cities; in 24 states there is no express statutory authority granted for final approval or adoption of a comprehensive plan by the city council or other local legislative body after it has been prepared.⁽²⁾

Some of the five states (Arizona, Florida, Missouri, Texas, and Wyoming) that do not have general laws authorizing the establishment of local planning agencies with power to prepare comprehensive community plans for urban areas, have provisions made for planning by other means. In Florida, authority to establish city and county planning commissions is found in special legislative enactments. Missouri provides for county planning under certain circumstances; and in Texas, city planning commissions are authorized by the charters of individual cities.

In addition to deficiencies in planning legislation, there are also deficiencies in authority for zoning and regulating subdivisions, particularly on a county or regional basis. State legislatures have passed enabling acts giving zoning powers to about half of the 3,000 counties in the United States.⁽³⁾ In some states all counties may zone; in others, only the more populous counties may do so.

It is readily seen that the lack of adequate authority to plan and zone will restrict the use of flood plain regulations in large sections of the country.

Of the communities that have authority to plan and zone, not all are taking advantage of the powers available to them and some are even hostile to their use. Such hostility is usually justified on the theory that community planning,

zoning and other regulations infringe on the right of the individual to exercise free choice in the use of his property and therefore should be opposed for the preservation of individual liberty. Fortunately, such attitudes are held by a decreasing minority as it becomes more evident that an increasing population density requires closer living associations with neighbors and that unrestricted individual liberty frequently produces conflicts between individuals or individuals and groups of people to the detriment of the general welfare.

The need for authority to plan land use and the need for courage and determination in taking steps to guide such use along wise paths indicate that there are four primary requirements for carrying out regulation of flood plains at the local level:

1. State enabling legislation to empower communities to prepare community development plans.
2. State enabling legislation to empower communities to enact zoning and other regulations with flood plain provisions.
3. The will to prepare the necessary plans and enact the necessary ordinances.
4. The will to enforce the ordinances after enactment and to resist granting exceptions to them.

These basic principles appear simple enough but their application is difficult. In zoning a flood plain to serve the needs of the community in the best manner possible, with a minimum of danger from flood damage and the future growth of the community considered, a careful analysis of the flooding situation is required. A distinction between the general "flood plain" and the floodway must be made. The flood plain is the area adjoining the river or stream which has been or in the future may be covered by flood water. The floodway includes the channel of a river or stream and those portions of the flood plains adjoining the channel which are reasonably required to efficiently carry and discharge the flood water or flood flow of the river or stream. It is readily seen from these definitions that the floodway must not be obstructed if its efficiency is to be preserved. On the other hand, the efficiency of the floodway may be increased by such measures as enlarging the channel, removing obstructions, and other works. Such works should receive consideration in some cases in planning the use of the flood plain and weigh the economic influences demanding the placing of improvements in the flood plain against the need to restrict flood damages and preserve the degree of efficiency of the existing floodway.

Such problems are highly technical and are usually too involved for the local zoning board to solve satisfactorily. In such situations, help should be solicited from the local engineering department if adequately staffed to deal with the problem, from consultants, and from state or federal water resources agencies.

The action required by individual cities and counties in obtaining effective regulation of flood plains has been very ably summarized by Murphy.⁽⁴⁾ He points out that local communities need to consider:

1. Forming a planning commission if one does not already exist and preparing a master plan of land use of the community.
2. Reviewing existing ordinances and codes that contain provisions concerning development in flood areas, revising them, and adding to them if they are inadequate to regulate development under major flood conditions.

4. Preparing zoning ordinances, subdivision regulations, and building codes, if not already in existence, so that certain definite and adequate provisions for regulation of flood-plain development are included.
5. Cooperating with federal and state agencies in the procurement of needed flood data.
6. Instituting a program of acquisition of appropriate portions of the most hazardous and frequently flooded lands.
7. Instituting a program of annual maintenance of stream channels.

In addition to the actions cited above, the will to do something about the problem and the will to enforce regulations are very necessary ingredients for achieving effective results.

State Legislation

In the discussion of local regulation of flood plains it was pointed out that adequate statutory authorization for planning, zoning and subdivision control at the city, county and regional levels is lacking in a number of states. Such authorization by the state is a first step toward enabling local communities to assume the duties that must be carried out at the local level. In addition to granting authority for local action, the states can take other steps to support effective flood plain regulation. This support can be in the form of promotion of local responsibility in flood plain regulation, technical and advisory help with flood plain problems, state supervision of certain types of developments, such as dams and levees, and state control of floodways under certain circumstances.

Lack of understanding of flood plain problems, ignorance of the potential danger in the flood plain, and local pressures to disregard recognized danger account for much of the inaction in dealing with flood plain problems. An effective program of contacting planning officials and zoning boards to inform them of the flood dangers in their communities, of steps they may take to combat those dangers, and of encouraging them to adopt the plans and ordinances necessary can be carried out at the state level. A state agency with one or two people, expert in these matters, can serve all the communities in the state in promoting better flood plain regulation.

The determination of the floodway requirements and boundaries is a technical task that involves a study of past and probably future flood discharges, characteristics of the floodway, and possible modifications of the floodway. History of past floods is frequently insufficient to furnish a safe guide for floodway capacity requirements, and future greater floods must be considered. The technical experience for making such studies will be lacking on most planning boards and on most city engineering staffs. However, most states have water resources agencies with responsibilities for making studies of floods and other water problems. These agencies, because of their familiarity with such problems can provide assistance and advice to local officials. Local zoning regulations should prohibit encroachments in floodways and the regulations should be supported by state encroachment laws. In addition, state encroachment laws should also protect those floodways not protected by local laws. Local planning and zoning officials may find it advantageous to require the approval of state agencies as a prior condition to granting exceptions to floodway zoning ordinances.

State laws should prohibit encroachment on floodways and should for adequate technical review and approval of all structures, such as dams, pipelines, etc., that must be constructed in floodways. All filling, excavating, or other constructions that might adversely affect the efficiency or capacity of the floodway should be prohibited by legislation and penalties should be provided for violations.

Dams, which can have a high hazard factor and can be a serious threat to developments on flood plains and to public safety, lack adequate regulation in many states. Only 45 states require permits for the construction of dams, and only 35 require approval of plans before construction, and only 35 provide for inspection after construction. Even though inspections are made, many states lack adequate statutory authority to order the abatement or removal of hazardous conditions or to assess penalties for noncompliance with orders or enforcement measures.

Levee systems protecting urban areas frequently suffer from inadequate design and maintenance with consequent danger to life and property. In many such structures often lack a responsible owner to provide necessary maintenance. This situation can best be remedied through state inspection and a requirement that local communities provide adequate maintenance. If local maintenance is deficient, the maintenance should be performed by the state and charged back to the local community.

Flood data can best be gathered and analyzed at the state level by full time specialized technical staffs, which cannot be financed by local government, can be provided for such purposes at this level. In many cases the restrictions of local governmental boundaries are not limiting factors in the extent and scope of investigations. The controlling features that determine flood heights are frequently at a considerable distance beyond the usual county limits that limit local jurisdictions.

It would appear that legislation at the state level should provide:

1. Statutory authority for the creation of local planning agencies to prepare comprehensive community plans including designation of flood plain and floodway areas.
2. Statutory authority for enacting zoning and subdivision control laws.
3. Enactment of floodway encroachment laws and regulation of construction in floodways.
4. Enactment of laws governing construction and maintenance of levees.
5. Establishment of a state flood control or water resources agency with regulatory powers, authority to make investigations, collect basic data, prepare flood-evaluation reports, provide advisory service to local planning agencies, and to cooperate with federal agencies.

The role of the state should be to support the local governmental agencies in any policing actions necessary for local regulation, and to provide coordination and regulation in those matters that require uniformity of action for justice and efficient operation and in those situations that transcend the limits of local governmental jurisdictions.

Federal Legislation

The historic approach of the federal government to the problem of reducing damages has been through the construction of flood control or flood protection structures. Little effort has been exerted to prevent the creation of flood damage areas or to encourage local responsibility in this field. However, there is an awakening at the federal level to the need for preventing creation of new sources of flood damage in addition to continuing the traditional effort to provide protection to old areas.

The Tennessee Valley Authority in its recent report "A Program for Reducing the National Flood Damage Potential" cites the futility of continuing to take structural measures for flood reduction or flood protection in certain areas and at the same time permitting new damageable developments to take place at an even faster rate in other areas.

The TVA report suggests a three-pointed plan for placing more emphasis on control of the growth of the flood damage potential:

1. An expanded program for collection, analysis and wider use of flood damage data.
2. A broad approach to flood damage abatement which includes preventive as well as corrective measures.
3. More active participation by state and local governments in solving the flood problem.

To implement the program the TVA recommends that:

1. A national flood damage prevention policy should be developed. The objectives of the policy should be (a) to prevent the unnecessary spread of buildings and other improvements to new flood hazard areas, combined with continued efforts to alleviate damages in those flood hazard areas which have already been developed, and (b) to encourage appropriate state and local bodies to assume the responsibility for initiating and developing plans and programs for local flood damage abatement through construction of protective works and establishment of land use controls as necessary. The policy should also provide a basis for technical and financial assistance by federal agencies.
2. If protective works are economically justified as a part of the local program for flood damage abatement, the federal financial contribution to such projects should be contingent upon the requirement that the city or state initiate and prepare engineering plans of the proposed works and that such plans be submitted to an appropriate federal agency for review and approval.
3. As a prerequisite for the contribution of federal funds to local flood protective works, state and local governments should be required to adopt and administer flood plain zoning and other such controls as are needed to prevent the unnecessary spread of buildings and other improvements in areas subject to flood damage.
4. Provision should be made to insure that all federal agencies having responsibility related to site selection or financing of physical structures observe local flood plain zoning and other standards in carrying out their responsibilities, and in cities without land use controls but subject to flood damage require establishment of such controls before federal programs are approved and put into execution.

5. An expanded and systematic program of collecting and analyzing basic flood information by existing federal agencies should be authorized which will meet the needs of states, local communities, industries and others in adjusting to flood conditions.

6. For the purpose of providing cities and communities with basic information on which to prepare plans for growth and development and the prevention of local flood damages, appropriate federal agencies should be authorized, where such authorization is necessary, to prepare and request local flood studies which would describe and analyze the local flood situation.

The TVA recommendation, in addition to providing for greater federal activity in collecting basic data and providing basic flood information to local communities, would make flood plain regulation a requirement for qualification for certain types of federal assistance. This requirement would provide considerable stimulus to local regulation.

The Chief of Engineers, U. S. Corps of Engineers, has recently instructed all the Division and District offices to give greater consideration to flood plain regulation. He directed that survey reports of the Corps of Engineers will discuss adequately the problems of encroachments on channels and developments in flood plains. Where encroachments and developments are expected, appropriate practicable provisions are to be included in proposed plans of improvements and in requirements of local cooperation to protect flood-carrying capacity of channels and floodways.

These moves by federal agencies are in the right direction and in general need no legislation to place them in effect. However, there are deficiencies at the general policy level that detract from their effectiveness. A policy declaration by Congress that would endorse the recommendations of the Chief of Engineers and extend the recommended program to all federal agencies would greatly assist in promoting better flood plain regulation.

SUMMARY

Perhaps the greatest hindrance to accomplishing more widespread flood plain regulation is the lack of adequate authority for planning and zoning at city, county and regional levels in many states. The acceptance of planning and zoning methods for flood plain regulation is still in its infancy and greater educational effort is needed to bring about greater use of these methods for reducing the flood damage potential.

State laws to provide adequate planning and zoning authority should be enacted promptly where necessary. Such laws should be made adequate at city, county and regional levels.

Although all states have provisions for administering water resource activities in some state agency to some degree, many of the agencies lack adequate powers to deal with flood problems, to collect basic data or to enforce regulations. In addition, many are inadequately financed and staffed. Federal states' effort should be to support local flood plain regulations, collect flood data and prepare flood evaluation reports, provide advice and technical service to local planning agencies in flood plain matters, administer floodway-encroachment and dam-safety provisions, provide levee inspection and require adequate levee maintenance for urban areas, and cooperate with federal agencies.

The federal government should adopt a policy that will require local regulation of flood plains where indicated in federal flood control projects, enlarge the program for the collection of basic flood data, prepare flood-evaluation reports particularly in regions that cross state boundaries, and establish criteria for determining the extent to which loans on private construction in flood plain areas would be guaranteed by federal agencies.

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Proceedings of the American Society of Civil Engineers

IMPROVED TUNNEL SPILLWAY FLIP BUCKETS

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SYNOPSIS

This paper discusses several buckets, of the type used to deflect or flip tunnel spillway discharges downstream, in terms of their desirable or undesirable features. Several new types of buckets developed from hydraulic model tests are then described. Using data from these tests, dimensionless curves are presented to aid in determining the jet trajectory length, the spreading of the jet, the tail water drawdown at the bucket, and the pressures on the floor and side walls of the bucket.

INTRODUCTION

A tunnel spillway is composed of two basic parts—an upstream spillway crest, free or controlled, and a downstream tunnel, part of which is sloping and part near horizontal. From the standpoint of economy the tunnel diameter must be kept to a minimum. Since the tunnel is never allowed to flow full because of the possibility of siphonic action producing dangerous flow conditions, it is necessary to keep flow velocities high and to prevent turbulent areas in the tunnel. Spillway tunnels are usually designed to flow from $3/4$ to $7/8$ full at maximum discharge, making the outflow at the tunnel portal relatively deep. The combination of depth and velocity produces the highest possible concentration of energy and increases the difficulty of obtaining satisfactory flow conditions where the flow spills into the river. As an example, on the Glen Canyon tunnel spillways,¹ the maximum discharge of 276,000 cfs produces 19,000 horsepower per foot of width at the tunnel portals. On Grand Coulee,²

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Hydr. Engr., Div. of Eng. Labs., Bureau of Reclamation, Denver, Colo.
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See Table 1, for references 1-8.

an overfall spillway, where the maximum discharge is 1,000,000 cfs, gy per foot of width is only 15,650 horsepower, or one-tenth that on Canyon.

If it were feasible to construct an efficient hydraulic jump stilling the end of one of the Glen Canyon tunnels, the basin depth, from approach water elevation, would need to be 170 feet deep. The hydraulic jump would be over 1,000 feet and would require a basin 700 to 800 feet long more. Basin appurtenances such as baffle piers could not be used effectively because the high entrance velocity, 165 feet per second, would produce turbulence problems. The cost of a structure this size would be prohibitive. It is readily seen why other types of structures are used at the end of tunnels and spillways. Buckets have been the most common of these structures and are probably derived from the slight upturns placed at the base of early overfall spillways. It is not clear whether the designers intended that these buckets operate free or submerged. In some cases, the upturn was too slight to produce a measurable effect on a thick jet, but probably the intended purpose was to deflect the jet downstream to prevent undermining of the spillway structure. Buckets of this type are referred to variously as "ski-jump," deflector, bucket, fusar, trajectory, or flip buckets. For ease of expression, the term flip bucket will be used in this paper.

Flip buckets are not a substitute for energy dissipators because a flip bucket is inherently incapable of dissipating energy within itself. The primary function of a flip bucket is to throw the water downstream where the river-bed is usually certain to occur, does not endanger the safety of the powerplant, or other structures including the flip bucket itself. In accomplishing this primary function, buckets are also designed to spread the flow over as much of the downstream channel as is considered desirable in order to produce riverbed damage as much as possible. The jet trajectory is modified as necessary to cause the jet to impinge on the tail water surface at the desired location, and when possible, the steepness of the jet trajectory at the point of impingement is selected to produce horizontal and vertical velocity components which produce most favorable flow conditions in the river channel.

Although with the present state of knowledge it is impractical to give a complete design of flip buckets, it is intended in this paper to present certain facts which have been found to be true as a result of extensive hydraulic

Table 1

DESCRIPTION OF PROJECTS					
Reference: No.	Name and Location	Agency	Maximum discharge	Fall : max headwater: to : bucket invert:	Tunnel diameter
1	:Glen Canyon Dam : Colorado River Storage : Project, Arizona	:Bureau of Reclamation	: 276,000 cfs:	: 588'	:2 tunnel : each 4
2	:Grand Coulee Dam : Columbia Basin Project : Washington	:Bureau of Reclamation	:1,000,000 cfs:	: 420'	:Overfall : 1,650'
3	:Hungry Horse Dam : Hungry Horse Dam Project : Montana	:Bureau of Reclamation	: 50,000 cfs:	: 488'	:31' diam
4	:Yellowtail Dam : Missouri River Basin Project: : Montana	:Bureau of Reclamation	: 173,000 cfs:	: 512'	:20.5' di : horse
5	:Yanhee Dam, Thailand	:Kingdom of Thailand	: 212,000 cfs:	: 402'	:2 horses : 37.08'
6	:Wu-Sheh Dam : Taiwan, China	: Ministry of Agriculture : Royal Irrigation Department:	: 66,000 cfs:	: 387'	:27' diam
7	:Fontana Dam : North Carolina	:Tennessee Valley Authority	: 180,000 cfs:	: 423'	:2 tunnel : each 3
8	:Trinity Dam : Central Valley Project : California	:Bureau of Reclamation	: 24,000 cfs:	: 475'	:20' diam

sting and prototype observation. It is hoped that extension of the ideas presented, in the form of discussions, will help to provide a better understanding of the requirements necessary for improved flip bucket design.

Bucket Design Problems

It is usually difficult or impossible to predict the flow pattern to be expected from a particular bucket by mere inspection of the bucket shape. Because of variations in velocity and depth, the spreading and trajectory characteristics of a given bucket can be determined only by testing in a hydraulic model. Since the authors have had the opportunity to test various types of buckets and to observe or hear first hand of their performance in the field, the findings of these tests should, therefore, be of interest to designers who must often select a bucket type before the hydraulic model tests are made.

In the course of developing and improving bucket designs, a number of difficulties have been found and overcome. The following examples indicate the problems which may be encountered in bucket design and which may not be generally known.

The flip buckets on the tunnel spillways at Hungry Horse Dam³ and Yellowtail Dam⁴ of the Bureau of Reclamation projects, and Yanhee Dam⁵ and Wucheh Dam⁶ being built in Thailand and Formosa, respectively, are similar and are what may be called a "standard" type. The buckets are placed downstream from a transition which changes the circular or horseshoe-shaped tunnel to a flat bottom to correspond to the flat bottom of the bucket. High velocity flow in the tunnel makes it difficult to design a short transition; long transitions are usually costly. If the transition is not carefully designed, and preferably checked by model studies, there is the possibility of dangerous sub-atmospheric pressures occurring in the corners. The transition, therefore, becomes as much of a design problem as the bucket.

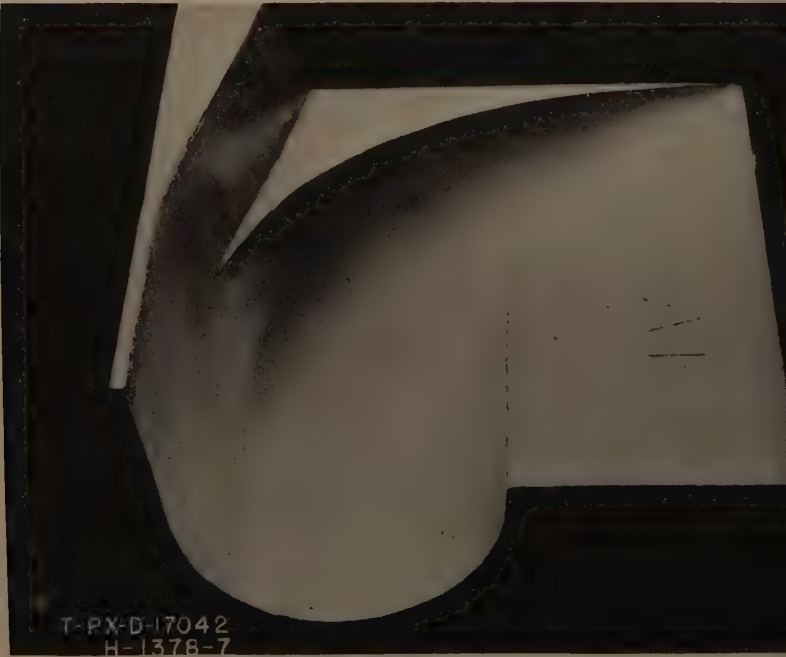
The Fontana Dam⁷ spillway buckets do not have an upstream transition. The bucket inverts are circular, the same as the tunnel inverts, Fig. 1.* The buckets were shaped by trial in a 1:100-scale model tested in the TVA Hydraulic Laboratory. The curved surfaces of the finally developed buckets could not be defined by ordinary dimensioning or even by mathematical equations. That the buckets were well designed has been proved by subsequent operation of the structure, but the methods necessary to convert the model dimensions at a scale of 1:100 to prototype dimensions were quite laborious. Because of the high velocity flow in the bucket, dimensions taken from the model could not be "scaled up" directly. Any small irregularity or misalignment when multiplied by 100 could have been sufficiently large to produce cavitation in the prototype bucket. It was, therefore, necessary to convert the dimensions to a 1:10-scale bucket, and after smoothing these, to convert the corrected dimensions to a 1:1 scale.

On some buckets, particularly those on foreign dams, a serrated or toothed edge has been placed at the downstream end of the bucket. The teeth are to provide greater dispersion of the jet before it strikes the tail water surface. High velocity flow passing over the sharp edges may produce cavitation damage on the concrete surfaces.

*Photographs from Technical Monograph No. 68, Hydraulic model studies, Fontana Project, Tennessee Valley Authority.



Bucket used at Tunnel 1 outlet



Bucket used at Tunnel 2 outlet

Fig. 1. Fontana Dam Spillway Flip Buckets

The problem of draining a tunnel which has a flip bucket at the downstream provides a challenge in design. The drain must be placed in a surface exposed to high velocity flow. Even though it is possible to design or develop in laboratory a drain opening which will not produce cavitation pressures, it is difficult to obtain field construction to the necessary tolerances to prevent cavitation from occurring. An ideal bucket design would be self-draining and would not present a cavitation problem at the drain structure.

Improved Bucket Designs

Recently a number of tunnel spillway flip buckets have been developed in the Bureau of Reclamation Hydraulic Laboratory that seem to offer simple but very effective methods of directing the flow away from the structure and which overcome, in part, the difficulties described in the preceding paragraphs. Although no single bucket eliminates all of the undesirable features of a bucket, the use of the principles described in the following pages will help the designer to provide an improved bucket on a particular structure. To review, an ideal bucket should provide (1) easy drainage of the tunnel, (2) a bucket shape which can be defined and expressed in prototype size by ordinary dimensioning on ordinary drawings, (3) no need for an upstream transition, and (4) an impingement area which may be shaped, by simple additions to a basic bucket, to fit the existing topographic conditions. Some of the buckets described are unique and probably cannot be generally used without some adaptation; however, the others are basic in type and have only minor additions to accomplish some specific function.

One of the unique designs was the Trinity Dam⁸ spillway bucket developed at a 1:80-scale model. The spillway tunnel enters one side of a wide shallow river channel and the flow tends to cross the river diagonally. It was necessary to discharge the flow into this channel without creating excessive eddies that might erode the riverbanks or cause disturbances in the vicinity of the powerhouse tailrace. The spillway is an uncontrolled morning-glory and, consequently, the flow can vary from a few second-feet to a maximum of 1000 cfs. The velocity at the bucket is 122 feet per second. Because small flows may occur for days, it was desirable for low flows to leave the bucket close to the riverbed elevation as possible to prevent excessive erosion near the base of the structure. On the other hand, large flows should be dispersed downstream away from the structure with as much dispersion as possible to prevent erosion and induced eddies from damaging the structure. In the standard flip bucket, a hydraulic jump forms in the bucket for small flows and the water dribbles over the bucket end and falls onto the riverbed. This could cause erosion which would undermine the structure. When the jump is first swept out of the bucket, the jet usually lands near the structure and erosion and undermining of the structure may still occur. At Trinity Dam, the foundation conditions at the end of the tunnel were such that it was deemed necessary to protect against the possibility of erosion and undermining. In order to place the bucket near riverbed level, the semicircular channel constructed downstream from the tunnel portal was curved downward in a trajectory curve, and the flip bucket structure was placed at the end, Fig. 2. The flip bucket surface consisted of three plane surfaces so placed that they spread and shape the jet to fit the surrounding topography. Large flows are spread into a thin sheet having a contact line with the tail water surface a considerable distance

downstream, Fig. 3A. However, even small flows are thrown downstream well away from the base of the bucket.

A training wall was used to prevent spreading of the jet on the high side, of the bucket. There was no wall on the low or river side of the bucket. At flows less than 1,000 cfs, a hydraulic jump formed over the horizontal surface and part way up the slope of the bucket; the flow spilled out on the side of the bucket into the river channel. The open side of the bucket was 4 or 5 feet above the river. Had the flow been confined on both sides, the flow would have been forced to spill out the end, the drop would have been over 40 feet and additional protection of the bucket foundation would have been required. At discharges greater than 1,000 cfs, the jump swept out of the bucket without hesitation with sufficient velocity that the flow was carried well downstream away from the structure. As the discharge increased, the jet was flipped farther downstream and became increasingly dispersed. The long contact line between the jet and the tail water reduced the unit forces on the tail water, and the forces induced at the ends of the contact line were thereby found to be a minimum. Since one side of the bucket is entirely open, the bucket is self-draining. Other advantages of this design are that the bucket may be defined for prototype construction with a few simple dimensions, and no curved or warped forms are necessary for prototype construction.

Another unusual type of flip bucket was developed for the Wu-Sheh tunnel spillway. Construction schedules and geologic conditions in the area made it necessary to modify this bucket from the standard type described earlier in this paper. After the line of the tunnel had been established, construction of the tunnel started, it was found necessary as a result of tests to change the direction of the flow entering the river channel. Rock slides, during the diversion period, made it necessary to construct retaining walls in the tunnel portal area which restricted the length of the bucket. Hydraulic model studies were made to determine how much the jet was required and whether the turning could be accomplished in the tunnel. The tests showed that it was undesirable to turn the tunnel and

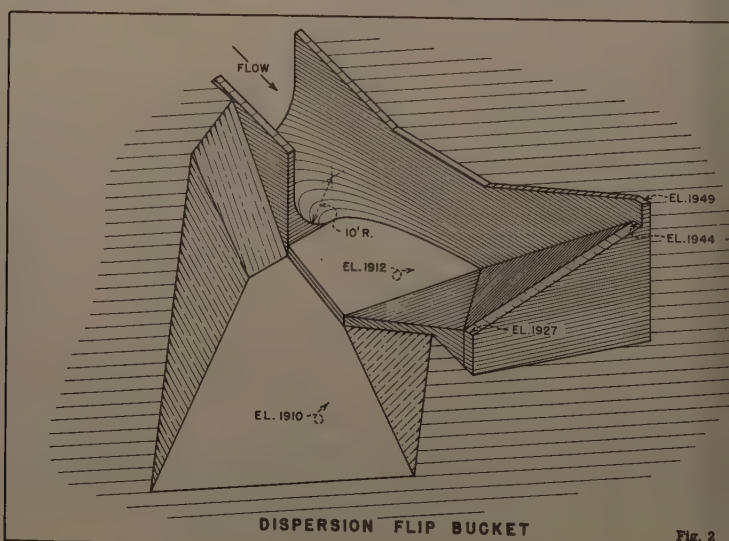


Fig. 2



Fig. 3A. Dispersion-Type Flip Bucket

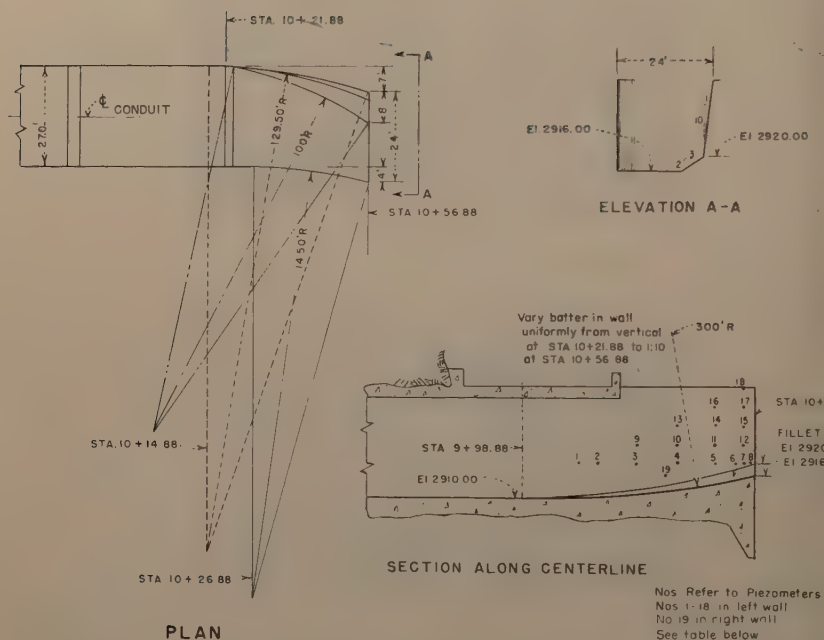


Hydraulic Jump in Basin—Discharge 12,000 cfs



Q = 13,000 cfs, Basin Acts as Flip Bucket

Fig. 3B. Combination Hydraulic Jump Basin-Flip Bucket



10 0 30
SCALE OF FEET

WU SHEH DAM
TUNNEL SPILLWAY
HYDRAULIC MODEL STUDIES
1/4125 SCALE MODEL
RECOMMENDED BUCKET

PRESSURES IN FEET OF WATER

PIEZ No	5,000 cfs	20,000 cfs	35,000 cfs	50,000 cfs	66,000 cfs
1	—	—	+9.0	+28.4	+50.2
2	—	+0.2	20.0	43.6	66.6
3	0	12.1	43.2	70.3	90.9
4	+2.3	23.3	50.6	70.7	89.0
5	6.4	36.5	59.2	72.7	84.0
6	6.1	30.8	45.4	53.6	62.0
7	4.4	23.8	35.0	41.2	46.8
8	2.1	13.9	19.4	22.6	26.3
9	-0.7	0	11.7	38.8	65.6
10	0	6.4	26.8	51.2	74.1
11	+0.9	20.5	42.4	58.0	70.2
12	2.7	17.7	32.6	41.0	46.9
13	—	1.0	4.3	21.1	47.5
14	—	-2.1	+5.2	21.1	37.1
15	—	+8.4	17.6	28.2	36.2
16	—	—	0.3	8.0	24.7
17	—	2.6	8.6	16.1	26.8
18	—	—	2.5	4.0	9.0
19	+3.6	+13.8	19.5	22.5	26.8

Fig. 4

turning should be accomplished in the bucket. The final bucket determined from model studies used curved walls to turn the flow—a batter in the left wall to prevent congestion in the bucket and reduce hydraulic loads at the larger discharges, and a fillet at the junction of the left wall and floor to smooth up or control the jet undernappe, Fig. 4. The resulting bucket was “tailor-made” to direct the flow to impinge near the middle of the river channel and to obtain the greatest dispersion possible at all discharges. The surfaces in this bucket could also be defined by ordinary dimensioning.

Piezometers placed in the side walls of the bucket showed above-atmospheric pressures at all discharges. The maximum pressure recorded on the left wall was 91 feet of water, Fig. 4. Before the wall was battered, the maximum pressure probably would have been much larger due to a more direct impact on the converging wall.

The Yellowtail Dam tunnel spillway flip bucket is a dual purpose bucket similar in some respects to the standard buckets. The tunnel is a curved bottom horseshoe-type conduit. Two hundred fifty feet upstream from the portal, the tunnel changes to a flat bottom horseshoe conduit, and the invert drops 26 feet by means of a combination transition-trajectory curve $172\text{--}1/2$ feet long. The bucket has a flat horizontal floor 130 feet long and a $62\text{--}1/2$ -foot-long upward sloping sill, Fig. 5. At spillway flows up to 12,000 cfs, a hydraulic jump forms in the bucket and relatively quiet water is discharged into the downstream channel. As the spillway discharge increases, the jump moves downstream and at 13,000 cfs sweeps out of the basin, Fig. 3B; for greater discharges and up to the maximum, 173,000 cfs, the basin acts as a flip bucket. The basin or bucket is placed low in solid rock so that discharges in the unstable zone, 12,000 to 13,000 cfs, cannot undermine the structure. This basin was developed in the Hydraulic Laboratory to serve the specific purpose of acting as a hydraulic jump basin for the most prevalent spillway discharges—discharges expected to be exceeded only every 10 years, and acting as a flip bucket to prevent damage to the structures during large floods.

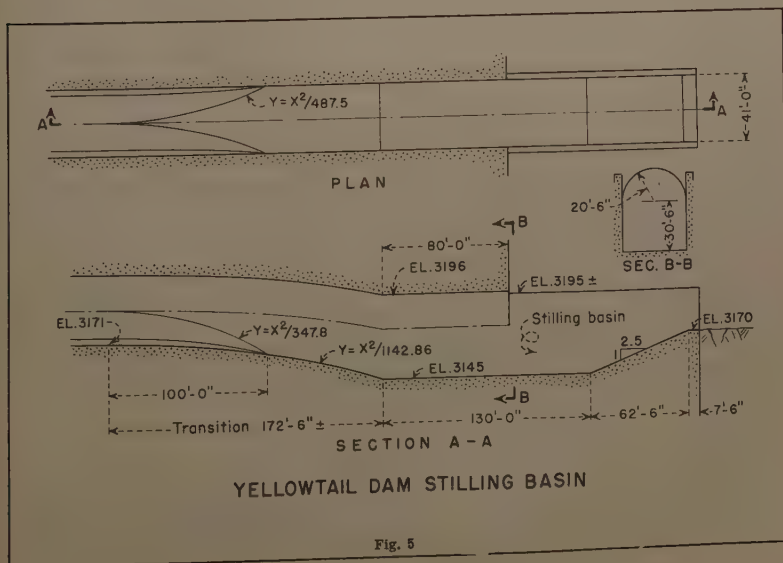


Fig. 5

The reason for using the hydraulic jump for part of the discharge range to protect the river channel against clogging with talus which was present in the canyon in large quantities and was expected to move if a high velocity stream contacted it. Following large discharges, it was expected that reopening of the channel to achieve full power head would be necessary.

The flip buckets for the Glen Canyon Dam tunnel spillways are an example of buckets developed to eliminate the tunnel transition and the need for a bucket invert. The buckets at the portals of the 41-foot-diameter tunnels are on opposite sides of the river and are aimed to discharge at acute angles toward the center of the river. The left bucket is farther downstream than the right. Each bucket is designed to handle the maximum discharge of 138,000 cfs at a velocity of about 165 feet per second. This represents over 13,000,000 horsepower in energy released into the river during maximum discharge.

In the preliminary design, there was a 70-foot-long transition between the circular tunnel and the rectangular channel containing the flip bucket. Hydraulic model studies indicated that the transition was too short, and that the atmospheric pressures would be sufficiently low to produce cavitation and damage to the structure. Two alternatives were developed during the model studies; one was to use a 100-foot-long transition in which the change in cross section was accomplished without dangerous pressures occurring and the other was to eliminate the transition by continuing the circular tunnel in a straight line downstream to intersect the upward curve of the flip bucket. The latter scheme was developed and will be used in the prototype structure; identical buckets will be constructed on the twin spillways. In effect, the transition buckets are combined into the bucket structure without complicating the design of the bucket.

Because the flat-bottom portion of these buckets diverges in plan, side flows are spread laterally more than for the flat-bottomed bucket. As discharge increases, the rate of spreading decreases so that it is easier to accommodate the jet for flood flows in a relatively narrow channel. Figure 6 shows a comparison of the flow from the two types of buckets. In the flat-bottomed bucket which is preceded by a transition, the flip curve extends across the full width of the bucket for its entire length; all of the flow elements at a given elevation are turned simultaneously. In the alternate bucket, the flip curve turns the lower flow elements in the center of the stream first and gradually widens its zone of influence as the flow moves downstream, resulting in greater dispersion of the jet. In effect, the flow along the center of the bucket is turned upward while the flow elements on either side of the center are turned upward and laterally. Training walls may be used to control the lateral spreading. In subsequent testing, deflectors were added to the bucket training walls to make the jets conform to the shape of the river channel and surrounding topography, Fig. 7.

The flip bucket used on the Flaming Gorge Dam tunnel spillway was the same type as used on the Glen Canyon spillways. The maximum design discharge for Flaming Gorge spillway is 28,800 cfs; the velocity of the flow at the outlet of the 18-foot-diameter tunnel is about 140 feet per second. The energy of the jet at the flip bucket is equivalent to 1,000,000 horsepower. In operation, the flow appearance of the Flaming Gorge bucket was entirely different than that of the Glen Canyon buckets. The Flaming Gorge jet was well dispersed at lower discharges and became more compact as the discharge increased. Figure 8. The Glen Canyon jets were well dispersed for all flows, and the lateral spreading with discharge was not so apparent. In the Flaming Gorge

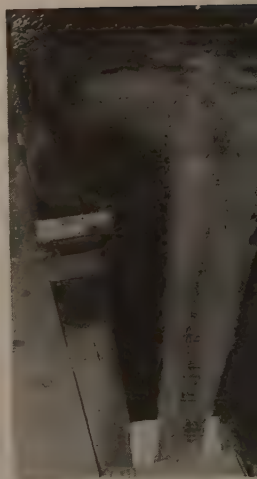
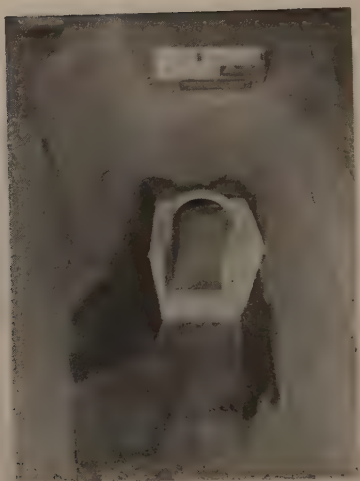


Standard Flat Bottom Flip Bucket
Flow at Froude Number 7.89



Transition Flip Bucket
Flow at Froude Number 5.64

Fig. 6. Glen Canyon Dam
Flip Bucket Studies

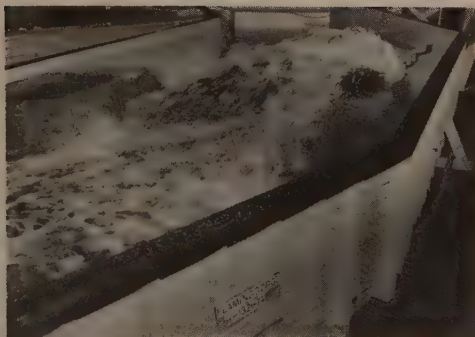


Transition Flip Bucket with Side Wall Deflector
Flow at Froude Number 5.64

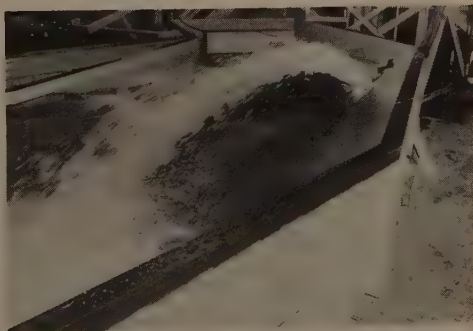


Typical Jet Profile
35° Transition Flip Bucket

Fig. 7. Glen Canyon Dam
Flip Bucket Studies



Jet is Well Dispersed at Low Flows.
Froude Number 10.3



A More Compact Jet Forms for Larger Flows.
Froude Number 6.8—Maximum Discharge

Fig. 8. Flaming Gorge Dam

Flip Bucket Studies

35° Transition Bucket

bucket, the water rose on the sides of the bucket at low flows, forming a "U"-shaped sheet of water in which the bottom and sides were of the same thickness. The vertical sides of the "U" followed the line of the bucket walls after leaving the bucket, while the bottom sheet of water had a tendency to diverge to either side. The vertical fins had a shorter trajectory than the lower sheet and on falling would penetrate the lower jet, tending to spread and disperse it; this can be seen in the photographs in Fig. 8. As the discharge increased, the size of the fins relative to the thickness of the lower sheet became insignificant and no longer had this spreading effect. The difference between the Glen Canyon and Flaming Gorge jets might be explained by the fact that the flow depth for maximum discharge was about 61 per cent of the diameter of the Glen Canyon tunnel and 81 per cent of the diameter of the Flaming Gorge tunnel. For a flow $0.61D$ in Flaming Gorge, the jet was still well dispersed.

Both the Flaming Gorge and Glen Canyon buckets were modified by increasing the height of the river side wall. The Flaming Gorge bucket is located well above the maximum tail water elevation so that the wall could be built to the spring line of the tunnel invert curve without tail water interference. The effect was to eliminate the fin that formerly rose along the wall, and spread out evenly to the right and was better dispersed than before. The Glen Canyon buckets are located more closely to the maximum tail water elevation and to prevent the tail water from interfering with the jet, the river side wall be cut down to only 5 feet above the spring line of the tunnel invert. The side wall remained to train the jet and very little difference in the flow pattern could be detected.

Design Considerations

Tunnel spillways usually make use of part of the river diversion tunnel. The downstream portion of the diversion tunnel becomes the horizontal portion of the spillway tunnel—the bucket is added after diversion needs have been satisfied. Since the diversion tunnel is one of the first items of construction it is often necessary, because of time limitations and construction schedules, to determine line and grade for the diversion tunnels before the details of the spillway are known. Care in selecting the exact position and elevation of the diversion tunnel, while keeping in mind its ultimate use as a spillway, will help to provide a dual purpose tunnel which will satisfy the temporary as well as the final demands with the least amount of modification when the bucket is added.

The following sections cover the items which should be considered in design and which will help to provide a simple bucket structure having desirable performance characteristics.

Elevation of Bucket Invert

It is desirable to construct the bucket and tunnel inverts at the same elevation. Since diversion requirements make it necessary to keep the tunnel low to provide the diversion capacity, the greatest danger is that the tunnel will be set too low for ideal spillway operation. This will require raising up the bucket lip to prevent the tail water from submerging the bucket. As a general rule, maximum tail water should be no higher than the elevation of the center line of the tunnel. If the bucket is set lower, difficulty may

experienced in obtaining free flow at low spillway discharges. The shape of the water curve will determine the exact requirements. The drawdown in water elevation at the bucket caused by the ejector action of the jet may affect the vertical placement. Drawdown is discussed later.

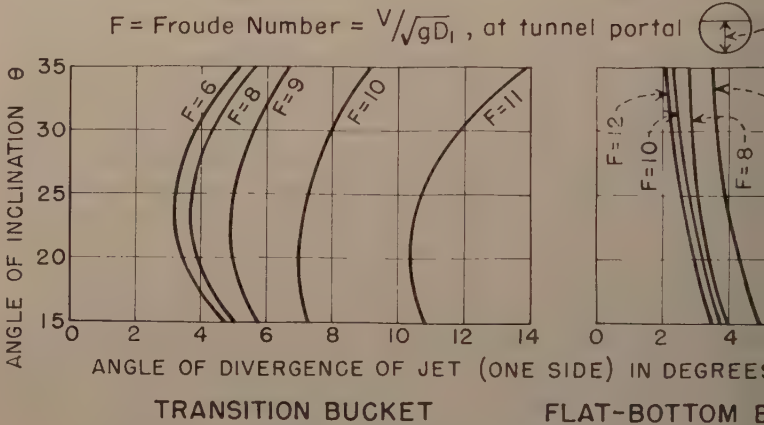
Direction

The bucket center line should be a continuation of the tunnel center line, the portion of the diversion tunnel used for the spillway tunnel should be straight. It is, therefore, desirable to aim the diversion tunnel so that it may be used without change for the spillway tunnel. The tunnel direction should be so that spillway flows will be aimed downriver and so that the design discharge impinges on the tail water in the center of the discharge channel. The jet should be directed to minimize the diameter of induced eddies at the sides of the jet since these can be very damaging to channel banks. In an ideal arrangement, the jet will be as wide as the channel so that there will be little return flow from the downstream tail water.

Fig. 9A shows the angle of divergence of one side of the jet leaving the bucket for two types of buckets—the flat-bottom type and the transition bucket used on Glen Canyon and Flaming Gorge spillways. In both cases, the angle of divergence is plotted versus the angle of inclination θ for a range of Froude numbers (of the flow entering the bucket). The flat bottom bucket produced very little change in angle of divergence for a range of Froude numbers or inclination angles. The transition bucket showed considerable change in divergence angle—from 4° to 12° for a Froude number range of 6 to 11. Since higher Froude numbers occur at low discharges, the transition-bucket-jet divergence is greatest at low flows. As the discharge increases, the Froude number becomes smaller and the divergence angle decreases. In most designs this is a favorable characteristic and results in improved river flow conditions for all discharges.

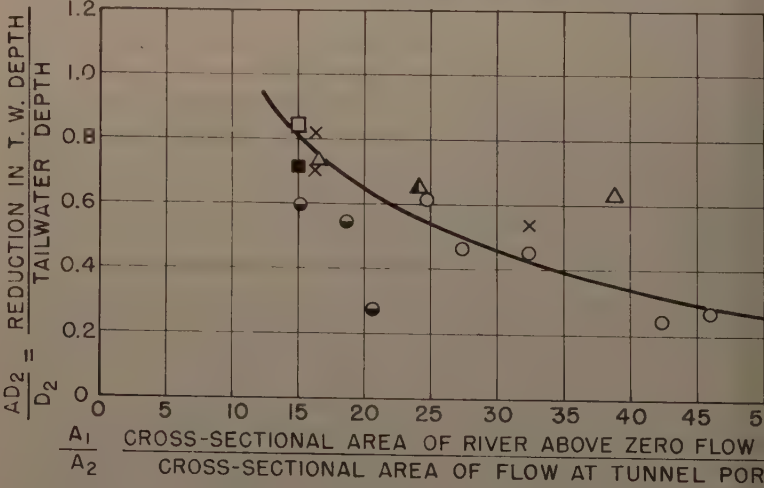
Drawdown

For the conditions just described, the jet will act as an ejector to lower the water upstream from the jet impingement area. From the Hungry Horse model tests, 26 feet of drawdown was predicted for 35,000 cfs discharge, and it was recommended that a weir be constructed in the powerplant tailrace to prevent unwatering of the turbines. Prototype tests made for 30,000 cfs showed 25 feet of drawdown and demonstrated that the weir was indeed necessary. At Hungry Horse the flow leaves the bucket at a 15° angle, making the jet trajectory relatively flat, Fig. 10; the jet is as wide as the downstream channel. The drawdown is maximum under these conditions. At Glen Canyon spillway jets do not occupy the entire width of channel, but the jet trajectory is steeper, and the discharge is considerably greater. Hydraulic model tests have indicated that up to 25 feet of drawdown may be expected. Other hydraulic model bucket tests have shown the drawdown to be appreciable, particularly when the jet occupies a large proportion of the channel width. No means have been found to calculate the amount of drawdown to be expected, except by making careful measurements on a hydraulic model. However, by using measurements obtained on several model studies and from prototype observations, the curve in Fig. 9B was derived. It is presented here as a means of estimating the drawdown that can be expected with a channel spillway and flip bucket.



A. SPREADING OF JET

- YAN HEE MODEL
- FLAMING GORGE MODEL △ HUNGRY HORSE MODEL
- WU SHEH MODEL ▲ HUNGRY HORSE MODEL & PRO
- x GLEN CANYON MODEL □ FONTANA PROTOTYPE



B. TAILWATER DRAWDOWN

Fig. 9



Hungry Horse Spillway Tests. Spillway Discharge
30,000 cfs. Side View of Jet.



Hungry Horse Spillway Model—Discharge 35,000 cfs
Model-prototype Comparison

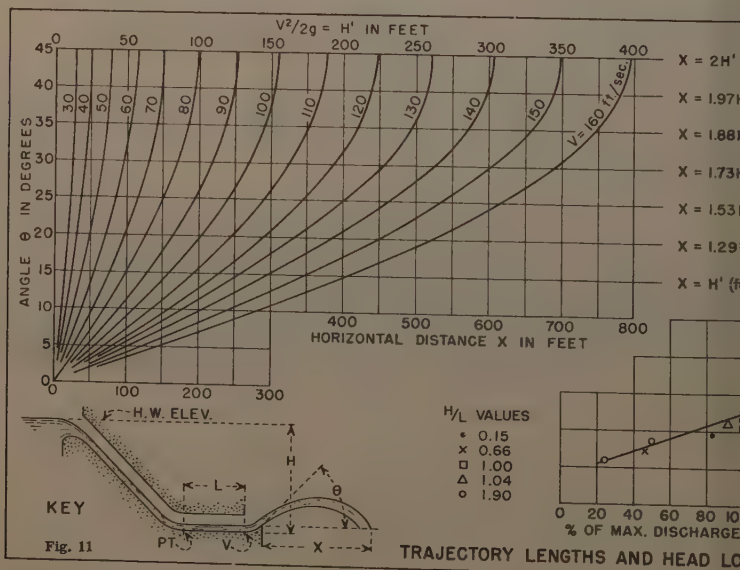
Fig. 10. Hungry Horse Spillway
Flip Buckets

The intensity of the ejector action and the resultant lowering of the water at the bucket have been found to be a function of the energy in the jet and the amount of resistance encountered when the jet strikes the tail water. In the curve of Fig. 9B the abscissa is the cross-sectional area of the flow near the point of impact of the jet divided by the cross-sectional area of the flow at the tunnel portal. The river flow area is the product of the difference between the no-flow tail water elevation and the tail water elevation at the discharge being investigated, and the average width of the river at the point of impact. The area of the flow at the portal is obtained by dividing the spillway discharge quantity by the average velocity. The ordinate is the amount of drawdown to the depth of tail water. The depth of tail water is the same depth used to determine the river cross-sectional area.

The curve, defined by the test points shown, indicates with reasonable accuracy the drawdown at each dam site for which data were available. The test points include various shapes and depths of channel and various types of bucket jets. Further, the two prototype tests on Fontana and Hungry Horse Dams showed good agreement between model and prototype test results. However, in predicting drawdown at future sites, the curve should be used with caution until more data are available.

Effect of Trajectory Shape

In addition to the effects of drawdown explained above, the jet trajectory is important in other ways. The angle of the bucket lip with respect to the horizontal determines the distance the water will be thrown downstream. How steep the angle, the more the jet will be broken up and slowed down by resistance. Both of these effects cause the jet to enter the tail water at a steeper angle. With a steep entry, the vertical component of velocity is greater, and the jet will tend to dig into the channel bottom. With flatter trajectories, the horizontal component will be greater, the drawdown will



ter', and the forward velocity will be higher. High velocity channel flow persist downstream from the impingement area for a considerable distance if the channel bottom does not erode to produce a deep pool. High velocity flow along the channel banks will then occur. If the bottom erodes, an energy dissipating pool will be formed, and flow downstream will be smoother. Bucket flip angles are usually constructed from between 15° and 35° . Angles less than 15° do not give enough lift to clear the bucket structure, and little is gained, from any viewpoint, by increasing the angle beyond 35° .

Fig. 11 contains a family of curves which may be used to estimate the trajectory length for inclination angles up to 45° and velocities up to 160 feet per second. These curves were obtained from the simple equations for the path of a projectile, $X = V^2 \sin 2\theta/g$. For a given angle θ the equation may be simplified as shown by the equations to the right of the trajectory curves. For 15° , $X = H'$; for $\theta = 45^\circ$, $X = 2H'$ etc., where H' is the velocity head at the bucket entrance. To estimate H' , the curve in the lower right of Fig. 11 may be used. Here, H' , expressed as a percentage of the total head H , is plotted versus the percentage of maximum tunnel discharge. H' is seen to vary from about 61 per cent for 20 per cent of maximum discharge, to about 75 per cent at maximum discharge. Maximum discharge is considered to occur when the tunnel is about three-fourths full at the outlet portal. The points which determine the curve have ratios of vertical drop to horizontal tunnel length, L , from 0.15 to 1.9.

Trajectory lengths taken from these curves have been found to be reasonably accurate when checked by hydraulic models. Some difference between model and prototype trajectory lengths may be expected to occur, however. It is known regarding model and prototype trajectory length agreement, but no measurements estimated or scaled from photographs, and from actual measurements reported by the author,⁽¹⁾ it appears that the differences are usually not critical in nature. The prototype trajectory is shorter than the model or theoretical jet and has a steeper angle of entry into the tail water. The difference is believed to be caused by the greater air resistance encountered by the high velocity prototype jet. From sketchy information on a few structures, the trajectory length in the prototype for 20 per cent of maximum discharge is believed to be 15 to 20 per cent shorter than in the model, etc. 12. There also are indications that the difference becomes less as the prototype discharge increases.

In determining the radius of the bucket curve, it is necessary to provide a radius at least four times as great as the maximum depth of flow. This provides an incline sufficiently long to turn most of the water before it leaves the bucket and provides assurance that the jet will be thrown into the desired area downstream.

Pressures in the Transition Bucket

Because of the simplicity and effectiveness of the transition bucket, it will probably be used on many future tunnel spillways. Extensive pressure measurements were therefore made on several buckets having two different inclination angles, 15° and 35° , to indicate that the buckets were safe against cavitation pressures and to provide data for structural design. The results of these tests have been summarized in Figs. 13, 14, and 15 and may be helpful in making preliminary designs.



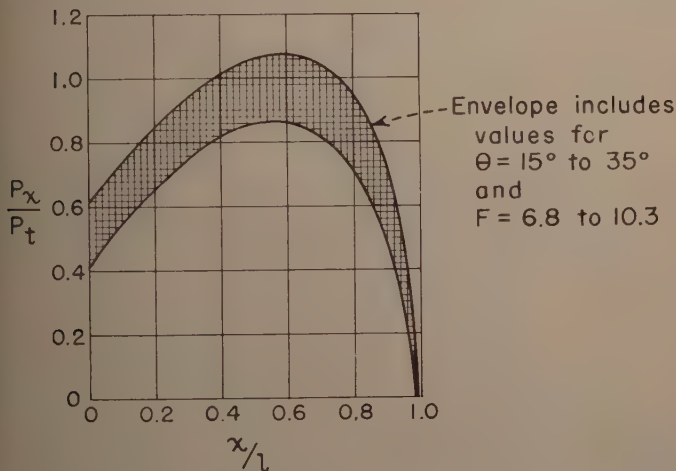
Both Prototype Tunnels Discharging 10,000 cfs Each



Both Model Tunnels Discharging 12,500 cfs Each

Fig. 12. Fontana Dam Spillway
Flip Buckets

Model-prototype Comparison



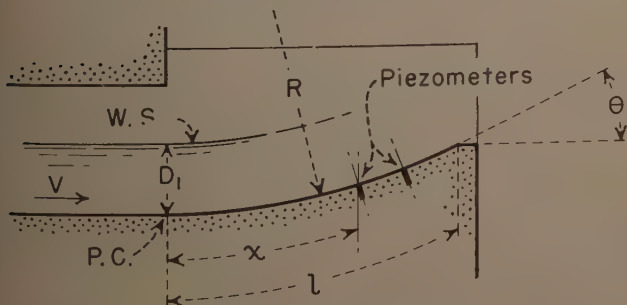
P_x = Measured pressure

P_t = Theoretical pressure; $(1.94\omega^2 R + 62.5) D_1$;
where $\omega = V/R$

x = Developed distance from P.C. to piezometer

l = Developed distance from P.C. to end of bucket

F = Froude number, computed from V and D_1 at P.C.



SECTION ALONG C

PRESSURES ON TRANSITION BUCKET FLOOR

Fig. 13

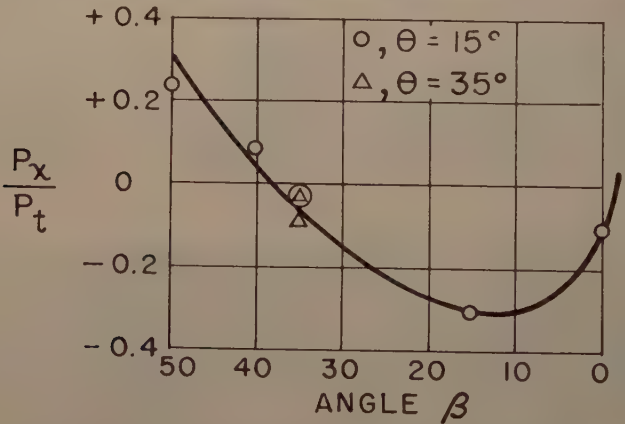
Fig. 13 shows pressures along the center line of the transition bucket floor. The envelope curve includes inclination angles from 15° to 35° flows in the Froude number range of 6.8 to 10.3, the usual range of conditions. The maximum pressure was found to be slightly greater than by Gumensky⁽²⁾ from theoretical considerations. The theoretical P_t is expressed:

$$P_t = (1.94 w^2 R + 62.5) D_1$$

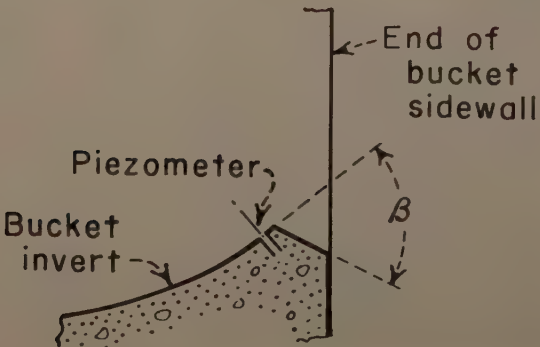
where

$$w = \frac{V}{R}$$

This maximum pressure occurred about 0.6 of the bucket length from stream end. Pressures rapidly became less toward the downstream the bucket and reached atmospheric at the bucket lip.



P_x = Measured pressure at end of bucket
 P_t = Theoretical pressure (See figure 13)



PRESSURES AT END OF BUCKET

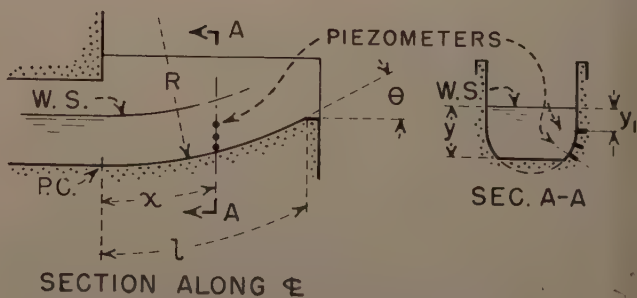
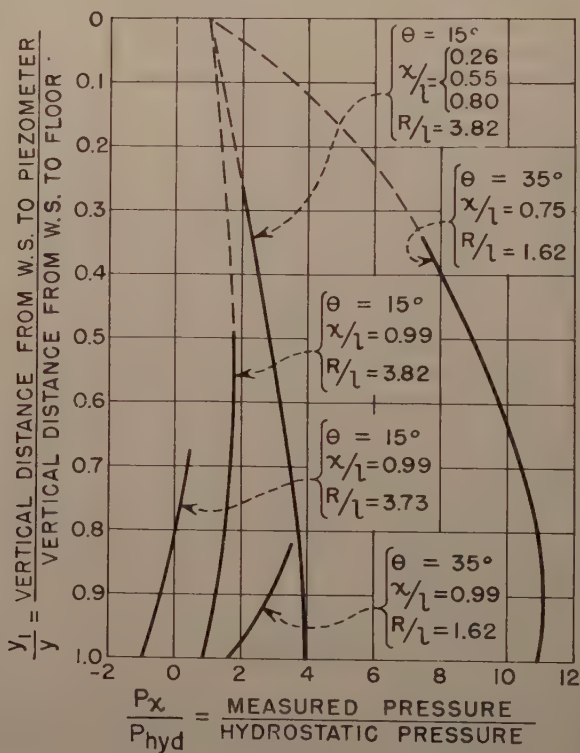
Fig. 14

For some tests a piezometer placed just upstream from the bucket lip, Fig. 14, indicated below atmospheric pressures, a phenomenon which has not been satisfactorily explained. Experiments on model buckets showed that the pressure on this piezometer was affected by the shape or angle of the downstream portion of the bucket lip. The curve of Fig. 14 shows the relation between pressure and the angle β . The curve indicates that for a given angle of inclination θ , β should be 35° or more to insure atmospheric pressures or less at the lip piezometer. The curve also indicates that if β is 0° the pressure will be atmospheric. This is not a practical solution, however, since if β is 0° the piezometer will then be upstream from the lip and a new problem will be created at the end of the extended bucket. It should be noted that the bucket side walls extend beyond the lip piezometer as shown in Fig. 14. The curves of Fig. 15 indicate the pressures to be expected on the side walls of the transition bucket from the base of the wall to the water surface. At an inclination angle θ of 35° the maximum pressure is about 11 times as great as hydrostatic and occurs near the base of the wall at about the three-quarters point, $x/l = 0.75$, of the bucket length. At the end of the bucket, $x/l = 0.99$ the maximum pressure is only four times as great as hydrostatic. At $\theta = 15^\circ$ the maximum pressure is four times greater than hydrostatic at $x/l = 0.26, 0.55$ and 0.80 and is only twice as great as static at $x/l = 0.99$. Experimental data are presented for different bucket radii, R/l values, and stations along the bucket, x/l values. Although the data are not complete, sufficient information is presented to make a preliminary structural design. On the Manning Gorge Dam spillway bucket, one side wall was cut down to the spring line of the tunnel without objectionable spreading of the jet occurring when the jet depth exceeded the height of the wall. This procedure simplified the structural design of the bucket by reducing the overall load on a wall which had no rock behind it.

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Design of Sidewalls in Chutes and Spillways, by D. B. Gumensky, Paper No. 2675, Transactions of ASCE, Vol. 119, 1954.



PRESSURE ON SIDEWALL OF TRANSITION BU

Fig. 15

Journal of the
HYDRAULICS DIVISION

Proceedings of the American Society of Civil Engineers

TIDAL CHARACTERISTICS FROM HARMONIC CONSTANTS

Bernard D. Zetler¹

ABSTRACT

Harmonic analysis of tide observations determines harmonic constants (amplitudes and epochs) of the harmonic constituents of the tide. Formulas using the constants to obtain various ranges are furnished and techniques obtaining typical curves are illustrated for various ports. These are compared with predicted curves.

INTRODUCTION

Many engineers have an occupational interest in tides since tide observations provide a basic control in hydrography, are important relative to maintenance of rivers and harbors, and are a factor in pollution studies. In recent years, projects involving storm-water levels and tidal-power development have emphasized the importance of tides to engineers. The tide tables furnish adequate predictions for most engineering problems but there are some aspects of tides that can best be generalized directly from the harmonic constants.

Harmonic constants are the amplitudes and epochs of the harmonic constituents that make up the tide at any place. Through a modified Fourier analysis, using the periodicities of the movements of the moon and sun and seasonal meteorological variations, the observed hourly heights of the tide for a period of observations are operated on to separate the tide into elementary harmonic constituents.

The determination of mean ranges, spring ranges (average near the times of new and full moon), perigean ranges (average when the moon is closest to

¹Discussion open until May 1, 1960. To extend the closing date one month, a written request must be filed with the Executive Secretary, ASCE. Paper 2317 is part of the copyrighted Journal of the Hydraulics Division, Proceedings of the American Society of Civil Engineers, Vol. 85, No. HY 12, December, 1959.

Chf. Currents and Oceanography Branch, Tides and Currents Div., Coast and Geodetic Survey, U. S. Dept. of Commerce, Wash., D. C.

the earth), etc., and the typical shape of a tide curve are best determined by the harmonic constants. It is the purpose of this paper to indicate what the harmonic constants represent and ways in which they can be used.

General Description

The tide is the periodic rise and fall of the water that results from the gravitational attraction of the moon and sun acting upon the rotating earth. The tide-producing force, varying with the constantly changing positions of both sun and moon relative to the earth, can be resolved into a number of simple periodic forces. To simulate the complicated motions of the sun, there are substituted a number of hypothetical tide-producing bodies, each having a fixed period for its circular orbit around the earth in the plane of the equator. Each hypothetical body produces a constituent tide with a period that can be calculated accurately from astronomical data. The actual tide at any place may be conceived as being made up of a number of constituent tides, the period or speed of each being determined by the appropriate hypothetical tide-producing body.

Fig. 1 shows the theoretical tide on an earth completely covered by a layer of water when the moon is on the equator. Friction and inertia have been disregarded so it is assumed that the waters have responded instantaneously to the attraction of the moon. Note that there is a high water bulge on the side of the earth nearest the moon, another high water bulge on the opposite side, and a low water belt circling the earth in between, going from pole to pole. The existence of a bulge on the opposite side can be shown mathematically, on the premise that the tide producing force at any point is the difference between the gravitational attraction of the tide-producing body on a particle at

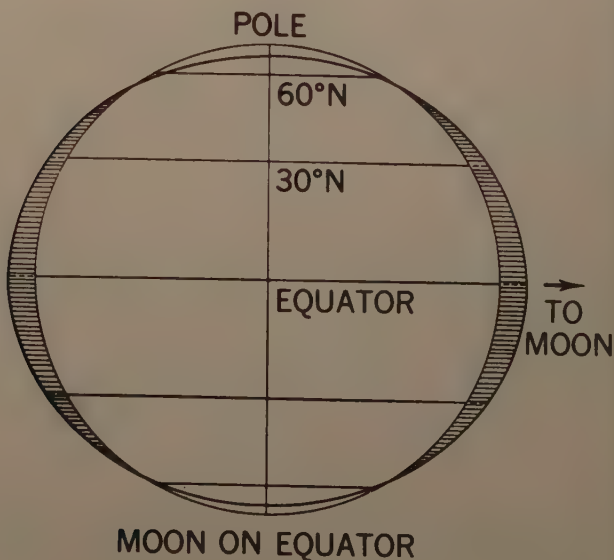


FIGURE 1

and the attraction on a particle at the center of the earth. In this mathematical development it is also shown that the tide-producing force varies inversely as the cube of the distance, rather than as the square of the distance of the gravitational attraction.⁽¹⁾

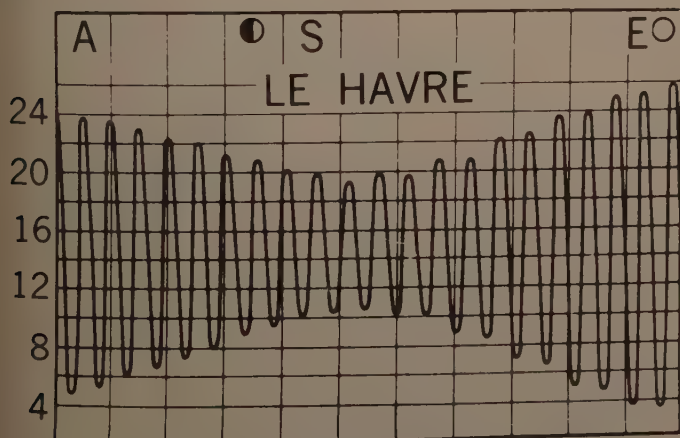
Period of Tidal Components

Inasmuch as the average interval between consecutive transits of the moon (over or lower) is 12 hours and 25 minutes, this is the period of the principal lunar constituent. The component is designated by the symbol M_2 , the M showing it is related to the moon's motion and the subscript showing that it is diurnal (period of about half a day). The principal solar constituent is a component named S_2 with a period of exactly 12 hours.

The components M_2 and S_2 will be in phase at both full moon and new moon. From Fig. 1 it is apparent that it will make little difference whether the moon and sun are in line on the same side of the earth (new moon) or in line with the center of the earth but on opposite sides (full moon). The large ranges of new and full moon are called spring tides. With the moon at first or last quarter, the M_2 high water coincides with the S_2 low water. This results in smaller than average semidaily ranges of tide called neap tides.

The periods of M_2 and S_2 of 12.42 and 12 hours respectively (corresponding to angular speeds of 28.98° and 30.00° per hour) will bring these components in conjunction or opposition at the proper phases of the moon.

A simple example of a curve consisting predominately of M_2 and S_2 is shown in Fig. 2 for Le Havre, France. Note that at spring tides, with large ranges, not only are high waters higher than average but low waters are lower than average. At neap tides, high waters are lower than average and low waters are higher.



TIDE CURVE
FOR LE HAVRE, FRANCE

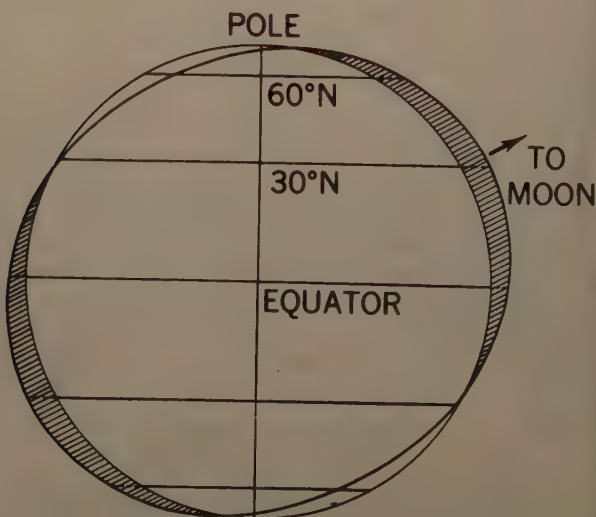
FIGURE 2

The moon moves in an elliptical orbit rather than a circular one. distance from the earth varies, its tide producing force varies accordingly being greatest when the moon is closest to the earth (221,000 miles) and smallest when it is at maximum distance from the earth (253,000 miles at apogee). The period from perigee to next perigee averages about 27.3 days. To simulate this changing attraction of the moon, we introduce component N_2 with a period of 12.66 hours (speed of 28.44° per hour) in conjunction with M_2 at times of perigee and in opposition at times of apogee. N_2 is known as the larger lunar elliptical semidiurnal component.

The distance of the earth from the sun also varies during the year. The degree of variation is considerably less than the moon from earth. Exponents representing this variation as well as numerous other lesser variations in the apparent paths of the moon and sun are used in tide predictions but will not be defined in this paper.

Thus far we have dealt with semidiurnal tidal variations, restricting the moon and sun to the plane of the equator. However the declination of the moon varies annually between about 23.452°S and 23.452°N while the moon reaches extreme declinations of 28.597°S and 28.597°N every 19 years.

Note in Fig. 3 what happens when the moon is at extreme north declination. The high water bulges are still present under the moon and on the opposite side. The low water belt still circles the earth in between, but no longer pole to pole. When the earth rotates on its axis, a point no longer experiences the same high tide with both upper and lower transits of the moon. At high latitudes there are two high waters of unequal height and at no other latitudes just one high and one low water a day. Thus there is a diurnal variation depending on the declination of the tide producing body. The principal components introduced to simulate these variations are K_1 and O_1 . The combined luni-solar diurnal component with a period of 23.93 hours



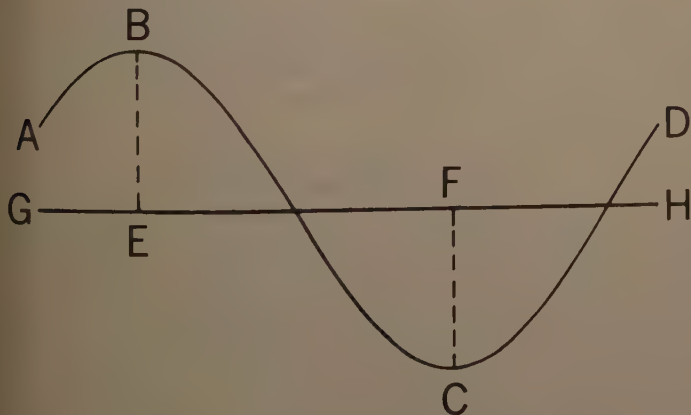
MOON AT MAXIMUM DECLINATION

FIGURE 3

la. speed of 15.04° per hour. O_1 is a lunar diurnal component with a period of 25.82 hours and an angular speed of 13.94° per hour. The sum of K_1 and O_1 angular speeds equals the M_2 angular speed, so that each day a diurnal high water comes in the same position relative to the two semi-diurnal high waters. Furthermore the speeds of K_1 and O_1 are so chosen that they are in conjunction when the moon is at extreme north or south declination and in opposition when the moon is on the equator.

Determination of Tidal Characteristics

In the early part of this paper there was specified an earth with uniformly distributed water and resulting instantaneous response to a tide-producing body. However our earth does have land masses impeding the progression of tide waves and the water areas are not uniformly deep. As a consequence the theoretical development gives us only the basic periods of the components of tide and some approximate relations between components but the amplitudes and epochs must be determined from observations by a special form of harmonic analysis.⁽¹⁾ Fig. 4 shows a complete period of a component tide. The amplitude is the average semi-range, BE or CF, the vertical height from the disturbed sea level, GH. If A is the time of meridian passage of a hypothetical tide-producing body, the time from A to maximum high water B, is the epoch or lag. It is expressed as an angle which is $(GE \div GH) 360^\circ$. Three systems of reference for the epoch of a component are used, each of which is designed for a particular purpose. Since the available constants may be referred to any of these, each will be briefly mentioned. If A represents the time of passage of the hypothetical tide-producing body over the meridian of the place for which tidal constants are furnished, the epoch is denoted as "K" for Kappa. If A represents the time of passage over the



COMPLETE PERIOD
OF COMPONENT TIDE

FIGURE 4

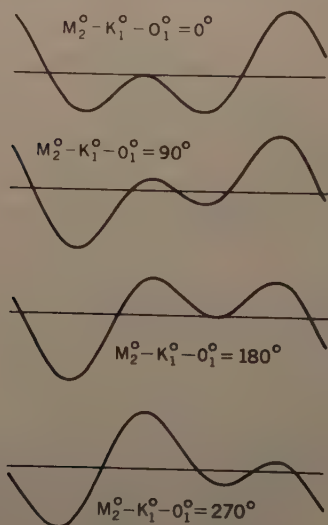
meridian of Greenwich, the epoch is noted as "G" for Greenwich time. Normally if A represents the time of passage over the meridian of the standard time zone (for example 75° W for Eastern Standard time in New York), the epoch is noted as a lower case "g" and the corresponding time difference is indicated. Formulas for computing either K, G or g, given any two of the three, are furnished in Coast and Geodetic Survey publications.

Once an amplitude and epoch are determined for each component, the tide can be represented by a cosine curve. These curves are summed on a tide-predicting machine and tide tables can be prepared for months in advance provided that the related hydrography of water basins does not change. Ordinarily about 20 components are used for a station by the Coast and Geodetic Survey although the machine is designed for 37 components. Various combinations of components are used for different stations. The components of significant amplitude, as obtained from analysis of tide observations, are used for each station predicted.

Examples of Combining Various Components

The amplitudes and epochs of the five components described previously, M_2 , S_2 , N_2 , K_1 , and O_1 , can be combined to approximate a typical tide curve, and to approximate various ranges and mean intervals.

As noted previously the combined hourly speed of K_1 and O_1 equals the hourly speed of M_2 . Consequently, although the combined K_1 and O_1 will vary in amplitude depending on the declination of the moon, the high water will always be fixed in position relative to the two semidiurnal water of M_2 . Fig. 5 shows some possible combinations of $(K_1 + O_1)$ curves given equal amplitudes and various phase relationships of the $M_2^{\circ} - K_1^{\circ} - O_1^{\circ}$. The epochs may be Kappa, G or g, provided all a



COMBINED M_2 & $K_1 + O_1$ CURVES

FIGURE 5

the same meridian. Note that when $M_2^\circ - K_1^\circ - O_1^\circ = 0^\circ$ the inequality is in the high waters, the low waters being equal. When the difference is in the inequalities are equal and the sequence is higher high to lower low. With a difference of 180° , the high waters are equal, the inequality being in the low waters. With a difference of 270° , the inequalities are again equal, the sequence is lower low to higher high. Using these four sets of curves as guides, it is possible to visualize the relationship for any value of $M_2^\circ - K_1^\circ - O_1^\circ$. In combining the epochs, multiples of 360° may be added to arrive at any value between 0° and 360° . Similar relationships will hold for M_2 and $K_1 + O_1$ amplitudes that are unequal; Fig. 5 may aid in visualizing various combinations.

The amplitudes of the harmonic constants can be used to approximate various average ranges. These have been derived from formulas and tables in the United States Coast and Geodetic Survey Special Publication 260, Manual of Harmonic Constant Reductions, 1952.

Mean range (long period average)	$= 2.2M_2$
Spring range (at new and full moon)	$= 2.1 (M_2 + S_2)$
Neap range (at quadrature)	$= 2.1 (M_2 - S_2)$
Perigean range (moon closest to earth)	$= 2.2 (M_2 + N_2)$
Apogean range (moon farthest from earth)	$= 2.2M_2 - 1.7N_2$

When M_2 is small relative to K_1 and O_1 ,
 Tropic range (diurnal range at extreme declination of moon) $= 2(K_1 + O_1)$

If the epochs of the various components that combine to form spring tides, neap tides, etc., were equal, these particular ranges would occur exactly at new and full moon, perigee, etc. This is not ordinarily the case, and the ranges can be calculated exactly from the epochs. The lag in hours for spring tides is found by the formula $0.984 (S_2^\circ - M_2^\circ)$, the lag in perigean tides by the formula $1.837 (M_2^\circ - N_2^\circ)$; and the lag in tropic tides by $0.911 (K_1^\circ - O_1^\circ)$. A positive lag means that the tide follows the appropriate astronomical event; a negative lag indicates that the tide precedes the event.

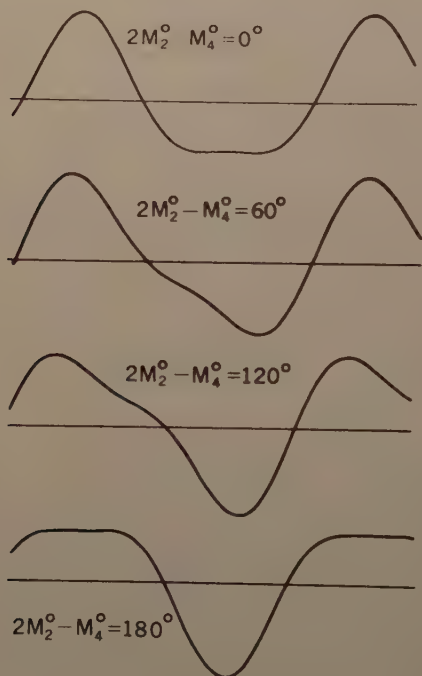
Influence of Tidal Basin on Tidal Characteristics

Some tidal components of significance are not astronomical in nature. M_4 and M_6 are harmonics of M_2 , their angular speeds being exactly two and three times that of M_2 . These are called shallow water components and reflect the distortion of the cosine-type tide curve because of the physical features of certain basins. The period of a basin is determined by its length and depth. When the period of a basin approximates a harmonic of the tidal period, it introduces a repetitive disturbing wave that more or less uniformly distorts the shape of the tidal wave. Figs. 6 and 7 show the shape of the curve as affected by M_4 and M_6 respectively. The degree of distortion depends on the ratio M_4/M_2 or M_6/M_2 and the portions of the curve principally affected depends on the phase difference $(2M_2^\circ - M_4^\circ)$ or $(3M_2^\circ - M_6^\circ)$.

Meteorological Effects

Finally there are variations in mean sea level due to periodic meteorological forces. Seasonal changes in prevailing winds have a measurable effect

on monthly mean sea level. S_a and S_{sa} are two components designed one cycle and two cycles respectively each year. The annual sea level can be approximated by a combination of these two components which have little effect on the times of tides because they change so slowly. Because of the slow rate of angular change for these components, approximate results are obtainable disregarding whether the epochs are $Kappa$, G or g . The epoch S_a divided by 30 denotes the maximum point of the annual variation in sea level in months after the vernal equinox (about March 21). Thus an epoch of 40° represents a maximum on about May 1. The epoch of S_{sa} is divided by 60 to obtain a semi-annual peak in months after the vernal equinox. A peak follows six months later. Thus an S_{sa} epoch of 140° represents a minimum sea level peaks on about June 1 and December 1. S_a and S_{sa} are determined from a number of years of observation. The published values at some places have been based on one year's observations; when this is the case, they may be unreliable.



SHAPE OF CURVE AS AFFECTED BY M_4
 $M_4/M_2 = 0.3$

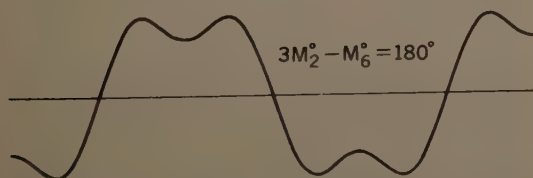
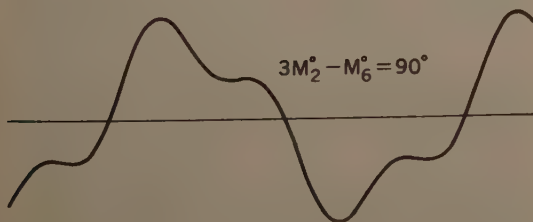
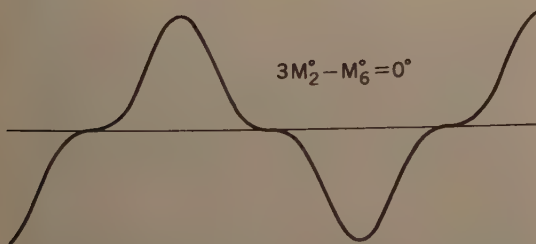
FIGURE 6

SUMMARY

me of the more important techniques can be demonstrated using the harmonic constants listed below and comparing the relationships with curves in 8 and 9, taken from the Coast and Geodetic Survey East Coast and West Tide Tables. The symbols at the top of these figures represent from right, apogee, last quarter, on equator, and new moon.

Principal Diurnal and Semidiurnal
Harmonic Constants for Various Places

K ₁		O ₁		M ₂		S ₂		N ₂	
H	G	H	G	H	G	H	G	H	G
feet	deg.	feet	deg.	feet	deg.	feet	deg.	feet	deg.
.46	206	.36	187	4.48	111	.74	146	.96	80
.41	58	.41	49	.06	172	.02	179	.01	54
2.73	278	1.50	256	3.53	12	.84	37	.71	341
2.22	340	1.17	324	11.56	106	3.16	149	1.97	78



SHAPE OF CURVE AS AFFECTED BY M_6
 $M_6/M_2 \ 1/3$

FIGURE 7

For Boston the most obvious relationship is the large amplitude for M_2 compared to K_1 and O_1 . This indicates relatively little diurnal inequality and a large semidaily range, as is brought out by the curves. Somewhat peculiar to the New England coast, N_2 is larger than S_2 , causing a larger mean perigean range than the mean spring range.

At Pensacola we have a completely different tidal regime. Although K_1 and O_1 are small, the semidaily components are relatively negligible. This results in a small diurnal tide that becomes negligible when the moon is on the equator because K_1 and O_1 oppose each other at that time.

At Seattle both daily and semidaily components are large and we have a substantial diurnal inequality. Combining epochs, $M_2^\circ - K_1^\circ - O_1^\circ = 198^\circ$ ($2 \times 360^\circ$ have been added to obtain a positive value). Going back to Fig. 5, we have a value between 180° and 270° , but considerably closer to the lower value. Thus we expect a high water inequality substantially smaller than the low water inequality and a sequence of lower low to higher high to higher low to lower high. Looking at the curve in Fig. 9 we find the above except during a transitional period when the moon is near the equator.

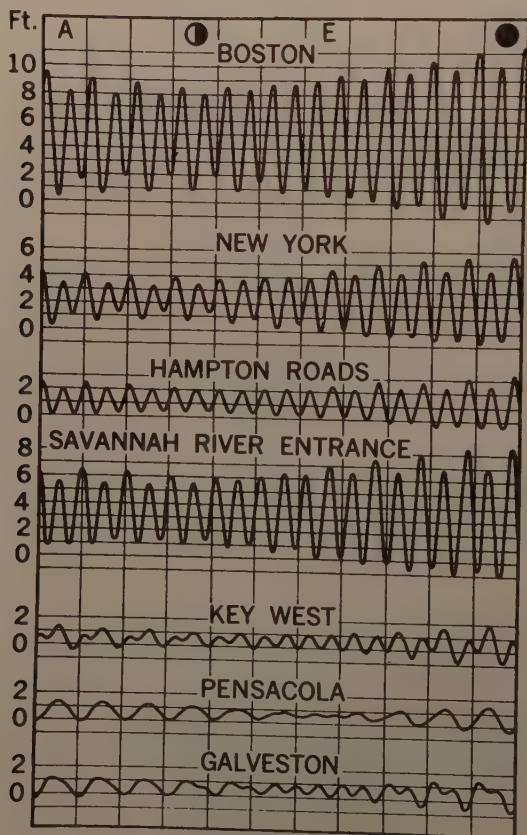


FIGURE 8

Finally at Anchorage the outstanding characteristic is the large amplitudes of the semidaily components. It follows that there is a large semidaily range with relatively small inequality and, more typically than at Boston, larger spring ranges than perigean ranges. Since $M_2^\circ - K_1^\circ - O_1^\circ = 162^\circ$, the inequality, although small, is larger in the low waters than the high waters. Fig. 9 bears this out.

The above are illustrated as examples of how data on typical shapes of the tide curves can be readily estimated from the harmonic constants listed. Other deductions on shape of curve can be made from the harmonic constants M_4 and M_6 , and of seasonal variation in sea level from the components S_a and S_{sa} .

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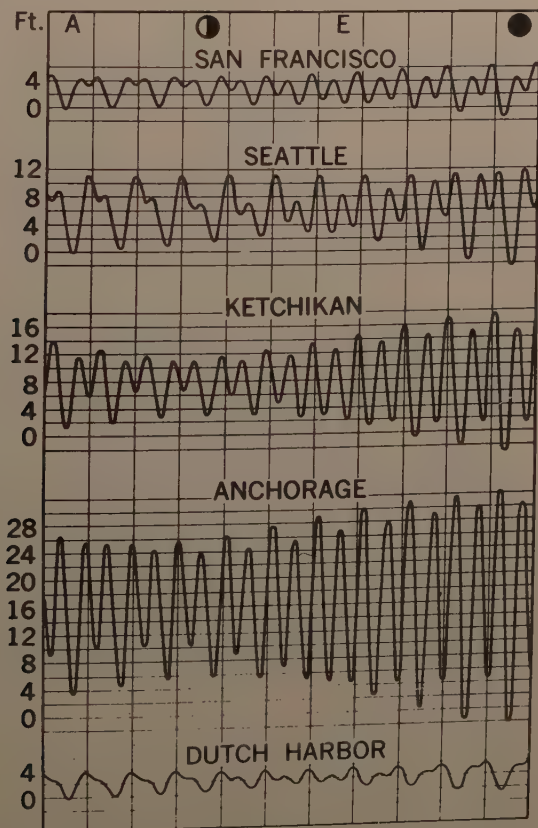


FIGURE 9

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HYDRAULICS DIVISION
Proceedings of the American Society of Civil Engineers

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(Over)

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INTERIM CONSIDERATION OF THE COLUMBIA RIVER ENTRANCE^a

Closure by John B. Lockett

JOHN B. LOCKETT,¹ F. ASCE.—The author wishes to thank Mr. R. E. Hickson for his discussion. During preparation of the paper, Mr. Hickson was furnished a copy of the final draft for his information and he took that opportunity to furnish his comments thereon. His published comments, although they are in substantial disagreement with the views expressed by the author, are welcomed since honest, constructive criticism is believed to be one of the keystones of human progress.

The paper points out that conditions at the Columbia River entrance have been undergoing and are still continuing to undergo a process of change under the influence of completed corrective and training works. Completion of the original South Jetty in 1895 was followed by extension of that structure in 1913, construction of the North Jetty in 1917, and construction of Jetty "A" in 1939, all of which were provided under the 30 and 40-foot entrance projects. This, followed by extensive dredging under the 48-foot project adopted in 1954, has subjected the entrance area to various degrees of control, resulting in continuously changing conditions within the entrance as river and ocean forces attempt to reach a state of equilibrium. Hence, within the time of permanent improvement, it is not believed logical to select any one short period in its history as representing an optimum stage of development. It is, therefore, believed that it may never be possible to restore entrance conditions to those prevailing during the period 1916-32, mentioned as a goal to be achieved. Instead, it is believed that we must work with conditions as they exist today and then determine how these conditions may be further improved for tomorrow.

Without casting discredit on the engineers responsible for the design and construction of existing Columbia River entrance structures, who certainly performed miracles of accomplishment considering the prevailing conditions and the limited resources available, it must be admitted that practically all of this work was, of necessity, planned, designed, and constructed largely on the basis of judgment considerations. Some of this work has been extremely successful while the true effectiveness of other work is still in doubt. Accordingly, it was considered advisable in connection with the adoption of the 48-foot entrance project, which includes provisions for construction of spur Jetty "B", that further work, including dredging, be based on known engineering fact rather than on judgment considerations as governed in the past. In view of the rapid advances made in the field of tidal hydraulics during the past 10 years elsewhere in the nation, it followed that a new approach to the problem at the

a. Proc. Paper 1902, January, 1959, by John B. Lockett.

1. Head, Special Projects Investigation Section, Planning and Reports Branch, Engr. Div., U. S. A. Engr. Div., North Pacific, Portland, Ore.

Columbia River entrance was indicated and that this new approach should include a searching engineering study of all possible phenomena which might have an important bearing on the problem. Mr. Hickson's position in this regard, however, may be summarized by the following extracts from his initial comments on the final draft of the paper:

"... it appears we are making a very complicated problem out of one which is relatively simple. It is simply a problem in river hydraulics in which the principles of hydraulics apply. It differs from inland river control only to the extent that winds, waves, tidal currents, density currents, and littoral currents distract our attention from the basic consideration, which is that the river outflow will produce and maintain a satisfactory channel if properly directed as to alignment and width between controlling bank or constructed works. This has very successfully been done on the upriver ship channel of the Columbia and other streams. . . . It appears that the problem need not be confused, or a solution made more difficult by attempts to make a detailed analysis of all the forces which are at work and what part each may have in producing or changing conditions. . . ."

This concept raises a question as to how we may properly direct the river flow to produce and maintain a satisfactory entrance channel when we know neither the full nature nor magnitude of the forces controlling the regimen of the estuary, let alone the effects of these forces on any further regulatory structure which may be devised.

The author, in general, does not believe that any strong exception could be taken to the discussion provided it was based on conditions prevailing prior to improvement at the Columbia River entrance when the forces of nature were such as to prevent saline intrusion into the estuary. The subsequent construction of corrective works and dredging, has, however, upset the balance of natural forces and has allowed the penetration of saline influences to the point where they are believed to represent the major dominant force controlling the regimen of the estuary today. This is a logical sequence of events which has been confirmed by histories of estuary improvement elsewhere.

There are, however, several points brought out by the discussor which relate quite fundamentally to the principles of tidal hydraulics and should, therefore, be considered in further detail. First of all, it is not believed that the design of any estuary structure can ignore saline influences as represented by density currents. These currents are created by differences in specific gravity of the fresh river water and the salt water of the sea. The magnitude of these currents is difficult to fully describe but may be visualized to some extent if we consider the hydrostatic conditions created by the meeting of the fresh river water weighing about 62.4 pounds per cubic foot with the salt water of the sea weighing about 64.0 pounds per cubic foot. The difference in specific gravity, reflected by the difference in weight of the sea and fresh water or 1.6 pounds per foot of depth, creates a hydrostatic head in an upstream direction with force of about 80 pounds per square foot at the 50-foot depth level. This is equivalent to a total resultant force of over 5,000,000 pounds when applied through the authorized one-half mile channel width at the Columbia River entrance. The true magnitude of this force becomes somewhat greater than 5,000,000 pounds when we consider that the full width between the Columbia jetties is at least two miles. This force, evidenced by density currents, acts throughout the region of saline penetration within the

estuary and is kept in balance by counteracting forces of friction along the estuary bottom and inertial energy of flowing fresh water. Until this density current force is finally dissipated at the upstream limit of saline penetration, it remains a factor deserving of detailed consideration in the planning of estuary structures. Therefore, the author cannot agree with the concept that we should not allow saline influences to distract our attention in the consideration of this problem.

Considerable emphasis has been placed in the discussion on the interpreted results of the 1932 current survey which was undertaken along a single range across the estuary about five miles above the outer ends of the Columbia jetties. Equipment employed to measure current velocities at different depths at each of the five measurement stations along this range consisted of the ordinary Price current meter suspended from the metering vessels. Facilities to determine or measure the direction of subsurface currents were, of course, not available at that time and, as a consequence, the actual direction of subsurface currents was largely a matter of personal judgment. Further, the survey did not include concurrent measurements of salinity and temperature along this range and, hence, it was not possible to confirm the judgment decisions of current direction with measured indications of salinity. It is regrettable that a substantial part of the detailed field records made during this current survey has been lost over the years and, accordingly, its value has been reduced by circumstances beyond control. Even though all these observations were available today, it is not believed, due to changes occurring within the interim period, that they could be accepted as representative of conditions presently prevailing in the estuary.

In this connection, the discussor states that, in general, the same littoral and density currents prevailed in the entrance during the period 1916 to 1932 as prevail today. Although the magnitude of the littoral effect is still unmeasured and, therefore, the accuracy of this part of the statement can neither be confirmed nor denied, there is logic to the belief that the density current pattern is substantially stronger today than it was during 1916-1932. Examination of condition surveys undertaken during that period reveal that controlling depths in the entrance, except between 1925 and 1927, were less than 44 feet and in 1916 were only 36 feet. In 1925 and 1927, controlling depths were 48 and 49 feet, respectively. It is a recognized principle of tidal hydraulics that when depths of estuarine channels are increased for navigation or other purposes, the effective density head at the entrance is increased and density current action within the estuary becomes more pronounced. Since 1932, the controlling depth between the jetties has been increased by at least 6 feet and it is, therefore, logical to expect greater density current action now than during the referenced period. Even though the channel section were held constant by the construction of dikes or contractive works, such as the proposed Jetty "B", deepening of the entrance channel must increase density current effects in the saline region of the estuary.

The discussor also states that salinity measurements through a tidal cycle show that most of the salt water is washed out by each ebb tide. He refers to Fig. 3 of his discussion showing in graph form measurements of specific gravity taken opposite Fort Stevens, Oregon, near the Point Adams Coast Guard Station, river mile 7.5, in 1936 as support for this conclusion. Although it is known that actual density of water varies somewhat with its temperature, there appears to be no record of concurrent temperature measurements taken at that time. Therefore, in the absence of temperature data, it is doubted that

these observations can be considered conclusive. In any event, it is difficult to reconcile these observations with those made at practically the same location in 1955 by a disinterested agency as reported in Table 4 of the paper under discussion. It is noted that, in forwarding his report on the 1936 observations to the Committee on Tidal Hydraulics, Corps of Engineers, Mr. Hixon, in his letter memorandum of April 10, 1953, stated that ". . . It is realized that the basic data secured were not comprehensive enough to admit drawing firm conclusions". Some six years later, however, these same observations are presented as firm data warranting the conclusion that ebb force remove most of the saline influences from the entire estuary. No work has been undertaken since 1953 which might lessen the magnitude of saline influences in the Columbia River estuary and there appears to be no basis for an appraisal of the value of the 1936 observations. Even though these particular observations were of such nature as to permit the formation of a firm conclusion, such a conclusion could only apply to conditions prevailing in the vicinity of Fort Stevens, river mile 7.5, but could not be indicative of conditions occurring elsewhere in the estuary.

The discussor also stresses the need for early construction of spur Jetty "B" as a regulatory structure designed to reduce dredging and provide a straight channel through the entrance. He presents cross-sections of the estuary in the general vicinity of this proposed structure and indicates thereon the approximate flow area that would be cut off by the structure. He is of the opinion that Jetty "B", by denying the river use of the area thus cut off, would contract the available cross-sectional area to the point where increased river velocities would solve the shoaling problem by erosion of Clatsop Spit shoals along the opposite shore.

Examination of the accompanying Fig. 1 shows that proposed Jetty "B" would lie along the north shore of the estuary, almost parallel to the North Jetty, with its extreme end approximately bisecting the distance between the ends of Jetty "A" and the North Jetty. Fig. 2 indicates that Jetty "B" would be constructed in depths averaging less than 25 feet below MLLW and in a maximum depth of 40 feet at its southernmost extremity. Of a total cross-sectional area of about 521,000 square feet, Jetty "B" would occupy about 160,000 square feet, or roughly 31 per cent of the total section. Under conditions of normal river flow, where there are no saline influences or reversals of currents to distract our attention, such a reduction in section could be expected to produce some effect on the direction of river flow by increasing current velocities to the point where they might erode less firm areas along the opposite shore.

Considering, however, flow conditions as they are believed to exist in this part of the Columbia River estuary, the prospect of solving the problem by this relatively simple expedient becomes somewhat less than promising. This is due to the fact that reversals of flow, created by tidal action, introduce factors that complicate or nullify the application of the principles of pure river hydraulics. Under such conditions, it may be seen that the confining effect of Jetty "B" would be limited to that portion of the time in which the entire flow throughout all depths in this restricted cul-de-sac area, as well as in the main Columbia channel to the south, would be in an ebb direction. It is believed that this condition, however, could be expected to prevail for only relatively short periods of time due to bottom flood flows generated by density current action, even during an ebb tide. During that portion of the time in which the flow in the cul-de-sac would be in a flood direction, it appears that Jetty "B" would be unable to affect the critical portion of Clatsop Spit lying to the west

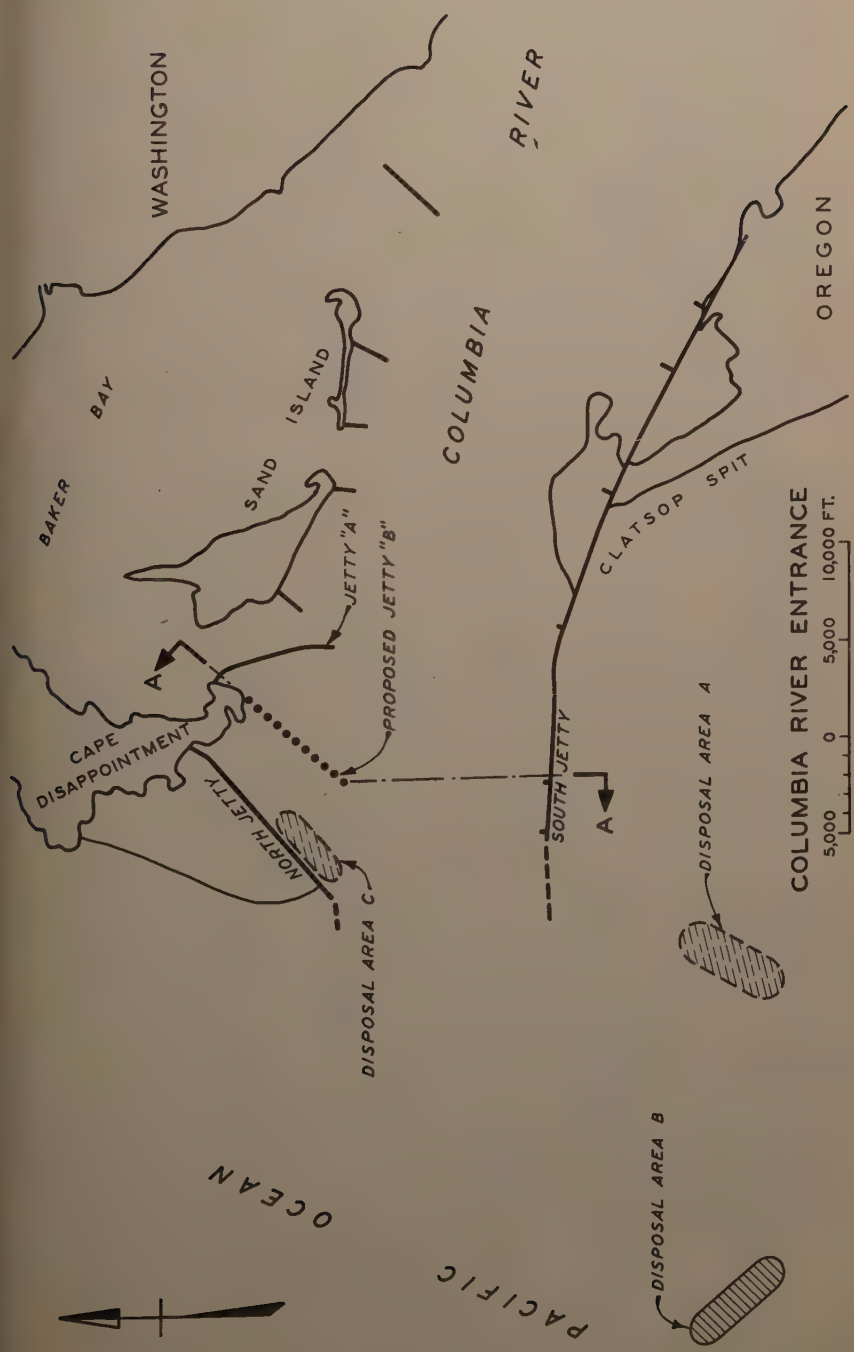
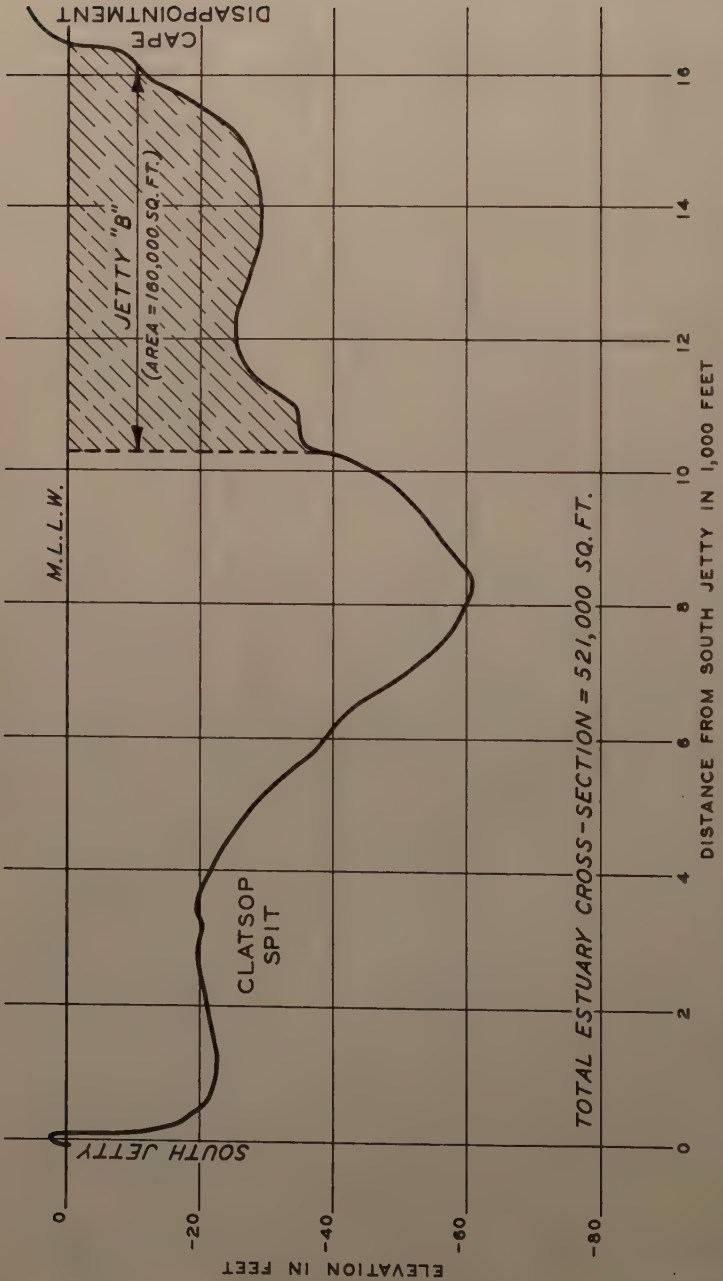


FIGURE 1



SECTION A-A

FIGURE 2

The structure might, instead, create eddy conditions which could conceivably threaten the stability of the North Jetty. Further, the main pattern of flow in this portion of the estuary would continue unaltered in the area immediately south of the structure where depths extend to 60 or more feet. It is believed that any density current action occurring in that principal flow area would, therefore, be unaffected by Jetty "B". Hence, there appears to exist reasonable basis for doubt regarding the true value of that structure.

The discussor claims that construction of Jetty "A" in 1939 stabilized the channel at that point, caused a widening of the channel for 800 feet to the south, and presently maintains depths in excess of 50 feet for a distance of over a mile downstream. Although the author does not disagree with evidence showing that the 40-foot contour opposite Jetty "A" moved southward following construction of that structure, it must be pointed out that this local improvement was accompanied by deteriorating conditions at the northwestern portion of Clatsop Spit, where the 40-foot contour in that area advanced markedly toward the north and west. Also, if we examine condition surveys prior to 1939, it will be noted that depths considerably in excess of 50 feet have prevailed for over a mile downstream from this point ever since improvement of the entrance was initiated. Accordingly, it is not clear how Jetty "A" may properly be credited with maintaining depths which have always existed in that area.

In regard to the effectiveness of Jetty "A" and the dikes along Sand Island, and the need for comprehensive study of the problem, the Committee on Tidal Hydraulics, in its reply of June 12, 1958 to Mr. Hickson's comments on Committee Technical Bulletin No. 2, which are both listed as unpublished references to the discussion, stated:

"... The second exception concerns your statement that the spur dikes along Sand Island and Jetty "A" have maintained an excellent channel since their completion about 1938; the Committee does not share your view that these structures have been as effective as is claimed. The Committee feels very strongly that the recommended program of prototype measurements and the model study of the Columbia River entrance should be carried out at the earliest practicable date. Until these missions have been accomplished, and the results obtained have been fully analyzed and evaluated, the Committee believes that the construction of any so-called improvement works in the problem area would be a grave mistake. The existence of such wide differences of opinion as to the basic reasons for this shoaling problem, among engineers who have devoted most if not all of their careers to the study of tidal hydraulics and related matters, is ample proof to the Committee that this problem is worthy of careful and complete study and evaluation prior to the expenditure of large sums for improvements".

Fortunately, the plan of action recommended by the Committee was initiated by Portland District with the completion in 1959 of a prototype measurement program which observed current velocities and directions as well as salinities and temperatures at some 23 stations along six ranges across the estuary. Observations of these phenomena were made at five depth levels on 30 minute intervals at each station throughout complete tidal cycles of 25 hours each under conditions of low, normal, and high river discharge. The observed data are currently being evaluated and the author hopes to present some of the findings of the program in a forthcoming paper.

Preliminary to the conduct of this measurement program, Portland District made several tests of current velocity and direction equipment offshore at Columbia River entrance. As it had been suspected that dredge spoil disposal in Areas A and C, which had been used for this purpose for many years, forced its way back into the entrance channel, tests were made in the vicinity of the particular spoil disposal areas. These tests revealed that such was quite likely the case and, beginning in 1958, Areas A and C were abandoned and practically all spoil was disposed in Area B, in depths of 120 feet or more. This change in disposal practices is considered by the author to be responsible for the securing in 1958 and 1959 of improved entrance depths by the end of each dredging season at substantially reduced dredging costs. It is believed that through this single expedient, the savings in dredging costs thus obtained will repay in a few years, the entire cost of the Columbia River tidal hydrographic investigation.

The author recognizes that it is quite probable that final evaluation of the prototype measurement program may not completely confirm all his views regarding the behavior of currents and other phenomena at the Columbia River entrance. That is of little importance. The important thing is, of course, the fact that positive action is now being taken to learn specific details regarding the forces controlling the regimen of the Columbia River estuary so that local and factual planning can precede actual design and construction of any further regulating structure or structures that may be found necessary.

Adoption of modified spoil disposal practices at the Columbia River entrance, based on modern technological knowledge in the field of tidal hydraulics, has already been rewarding and the continued application of this knowledge to the over-all problem holds the promise of still greater reward in the future.

HYDRAULIC ANALYSIS OF SURGE TANKS BY DIGITAL COMPUTER^a

Discussion by John P. Herak and Salvador Rodriguez

JOHN P. HERAK¹ and SALVADOR RODRIGUEZ,² A. M. ASCE.—The author has made an important and significant contribution to the literature of hydraulic transients. The writers know of no other contribution opening such large avenues to refinements in the treatment of transients and to eliminating the usually tedious labor of making arithmetic integrations. Two major points impress the writers. The first is the close check given to the manual computations for all large maneuvers. This also emphasizes again the inherent safety resulting from the use of orifices or equivalent restrictions. The second is the opportunity for making refinements in the solutions, thus producing mathematical results for many minor factors that heretofore were necessarily intelligent guesses, unless an undue amount of labor were to be expended. On the other hand this requires a closer look at the assumptions and coefficients used for these minor effects, in order for them not to be misleading. It is hoped that Yu-Tek Li's researches, referred to later, can be published soon, since they indicate considerable variation from some of the results, such as those of Vogel and McNown, previously available.

The following is an evaluation of the differences between manual and machine results of the surge tank stability study.

The evaluation of tee losses, in the arithmetic integration study, were based upon values obtained using Vogel's data for shape 2 (rounded edges) of tee connections, and a branch to main diameter ratio of 0.58. Tee loss coefficients prepared by the Omaha District, C.E., (Table 5) from Vogel's data were based on a branch to main diameter ratio of 0.667 and at zero discharge ratio have been set equal to zero, thereby modifying Vogel's results in the range of zero to 0.1 discharge ratios. The effects of these differences between manual and machine coefficients are noted below.

The arithmetic integration method of investigating the stability of the Oahe System was to subject the unit to a relatively small instantaneous load change and determine whether the amplitudes of the tank levels with respect to the new steady level were damped or magnified with increasing time. As stated by the author, manual stability computations proceeded directly from point M₁ to point M₂ (Fig. 11) and once entry onto the curve Bhp₂ was made, constant power was assumed to be delivered by the turbine up to full gate position. This transient produced oscillations in Q₅ and H along the constant Bhp₂ curve of 78000 without at any time reaching full gate. The arithmetic integration neglected the effects of governor dead time, friction losses, inertia, and

a. Proc. Paper 1996, April, 1959, by Nicholas L. Barbarossa.

1. Princ. Engr., Sverdrup & Parcel Eng. Co., San Francisco, Calif.

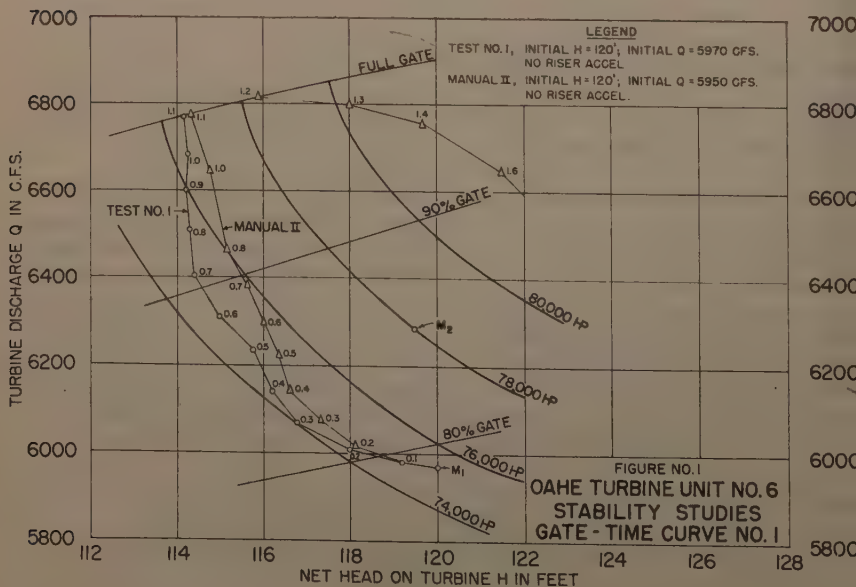
2. Senior Engr. II, Sverdrup & Parcel Eng. Co., San Francisco, Calif.

elastic energy in the hydraulic system downstream of the surge tanks; and energy storage and inertia of the rotating mass and electrical system. The computer studies included only partial corrections for some of these factors and did not consider the others. However, a step in the right direction was made by including the partial corrections.

A partial arithmetic integration study was made of Test No. 1, Fig. 11, using a time interval of 0.1 second and the results are shown on Fig. 1 as Manual II. The transient path to full gate position is similar to that obtained by computer, the variation apparently being due to differences in values of tee loss coefficients. In both cases, the transient attains full gate position before reaching the Bph_2 curve. We believe the computer should have been programmed to permit the transient to follow the full gate curve, as indicated in Manual II study to intersection with Bph_2 , with some overshoot due to governor dead time and system inertia. (Dead time considered equal to 0.05 second for Manual Study II.) At this point the governor takes over and begins closing the turbine gates since the turbine output exceeds the required horsepower. The path of the transient will approximate a spiral and will converge on the new steady state point, if the unit operation is stable. It is suggested that the computer be programmed as outlined above and allowed to traverse through several cycles to obtain a better measure of both surge tank and unit operational stability. It would be of interest to compare the decrement Δ (Jaeger), a measure of the damping of the hydraulic system, obtained by machine method with that obtained by the manual method.

The transient curves on Fig. 12 all indicate that the gates begin to close before the new steady state power Bph_2 is attained. This is not compatible with governor action.

As previously noted, differences in tee loss coefficients existed between manual and machine computations. A recent unpublished paper entitled "Head Loss in Tee Section and Manifold", (1) by Yu-Tek Li, February 1959, a form



member of our hydraulic section connected with the Oahe Reservoir Project, presents values of head losses at tee sections, head losses through orifices in the case where the plate orifice is installed in the lateral pipe, including values for different shapes of orifice plates, and for different locations of the orifice plate in the lateral pipe; together with a determination of head losses in a manifold system. Attention is being called to the paper, not with the intent of defending either set of tee loss coefficients used in these studies, but with the thought that while the writers agree with the author's ideas on the solution of transient problems by computer, additional basic work is still required in evaluating items such as head losses at tees before "exact solutions" can be made with computers.

In conclusion, the writers wish to commend the author on his excellent paper and look forward to the publication of his paper "Proposal for Comprehensive Study by Digital Computer of Hydraulic and Governing Transients Affecting Design and Operation of a Power Plant System".

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1. A dissertation presented to The Imperial College of Science and Technology London, England, 1959.

RESISTANCE EXPERIMENTS IN A TRIANGULAR CHANNEL^a

Discussion by J. van Malde

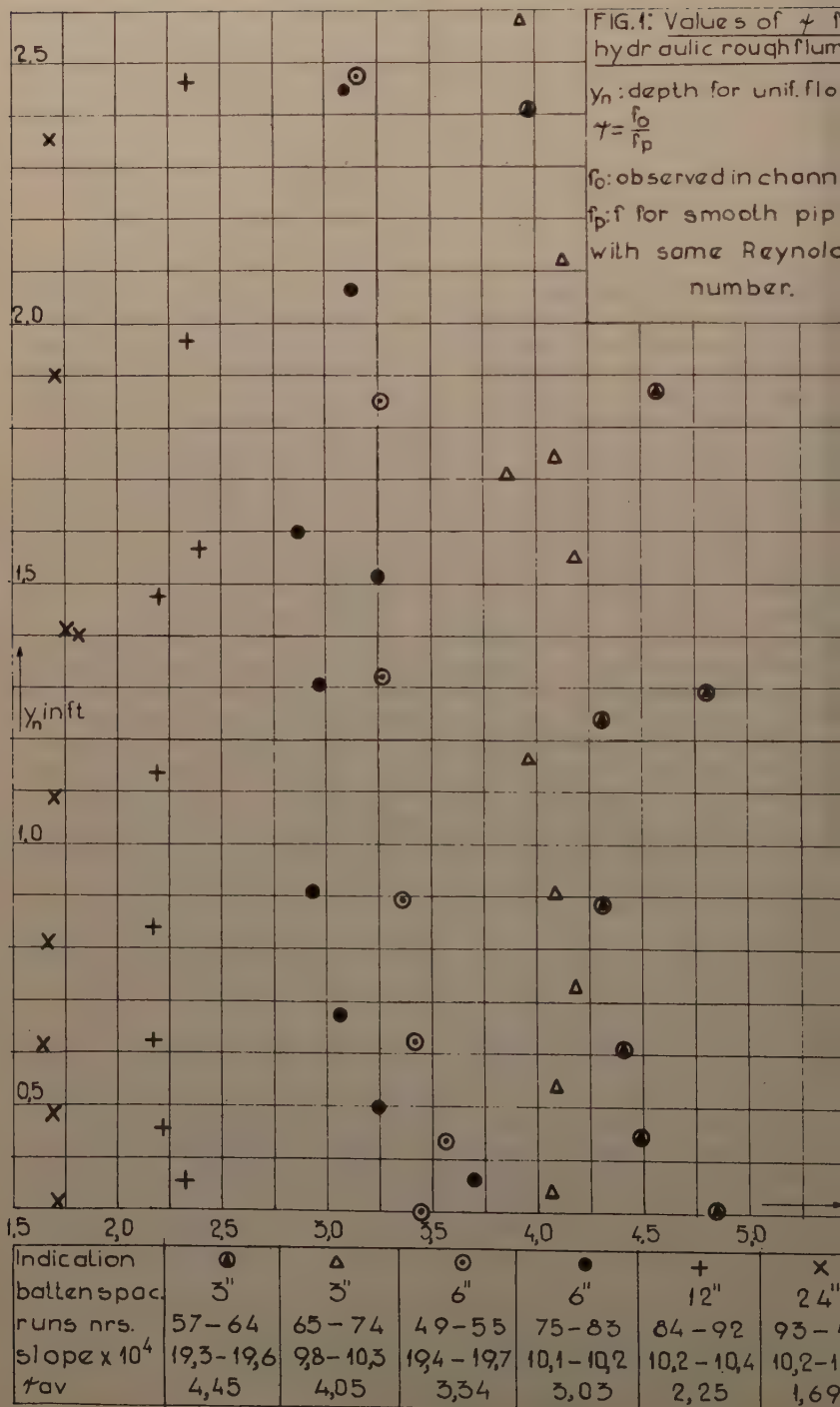
J. van MALDE.¹—This paper deserves study as the experiments described are unique regarding the sizes of the flume used, whilst the accuracy with which they were carried out make the results most valuable. The paper affirms that an open channel cannot be calculated with the well-known formulas of Prandtl (which were found from experiments with circular pipes) without making an error. According to their interpretation of the data the authors state that the average error in the mean velocity when using these formulas would amount to + 3,5% for the observed triangular channel being a smooth flume and - 2,2% in the case of the channel being a rough flume; these numbers refer to tranquil, non-critical flow. The authors conclude that in both cases the logarithmic formulas of Prandtl have no advantage over the Manning formula and that in the second case the formula of Morris for isolated-roughness flow has no advantage over Manning's or Prandtl's formulas either.

By studying the data for the hydraulic rough flume (runs 49-101) it appeared that the authors put together all values of ψ (the relation between f_0 observed and f_p for smooth pipe) for a certain roughness, irrespective of slope (S) and depth (y_n). O. Kirschmer, who as far as the writer knows first pointed to the resemblance in resistance of a hydraulic rough open channel to a hydraulic smooth circular pipe⁽¹⁾ mentions in one of his papers⁽²⁾ the possibility that ψ is influenced by S . This he concludes from some of the experiments of Bazin, taken in a rectangular flume with battens fastened to the walls in the direction of the flow; ψ would increase with increasing S . Theoretically there is no explanation for this influence, but as the theory for the uniform steady flow in open channels still doesn't take into consideration all the phenomena (such as secondary currents) there is no reason to exclude the possibility of this influence. For the same reason it may be possible that the depth of flow, y_n , influences ψ (though this seems unlikely as y_n is directly related to the Reynolds number, which is used to compute ψ).

Fig. 1 shows ψ (calculated with slide-rule accuracy) as a function of y_n for different values of S . From the points for runs 84-92 and 93-101 it may be concluded that for these cases ψ is independent of y_n ; the other series of points show a trend for ψ to decrease with increasing depth, but the points generally scatter too much to take this for granted. In the writer's opinion the points definitely show that ψ increases with increasing slope; this is the same result as Kirschmer did find.

Regarding smooth tranquil flow the authors found the indeed astonishing result that $\psi = 0.93$. For the runs concerned (1-16; 102-108) ψ was again

a. Proc. Paper 2018, May, 1959, by Ralph W. Powell and Chesley J. Posey.
 1. Civ. Engr., River Research Section, Directorate Upper Rivers, Rijkswaterstaat, The Netherlands.



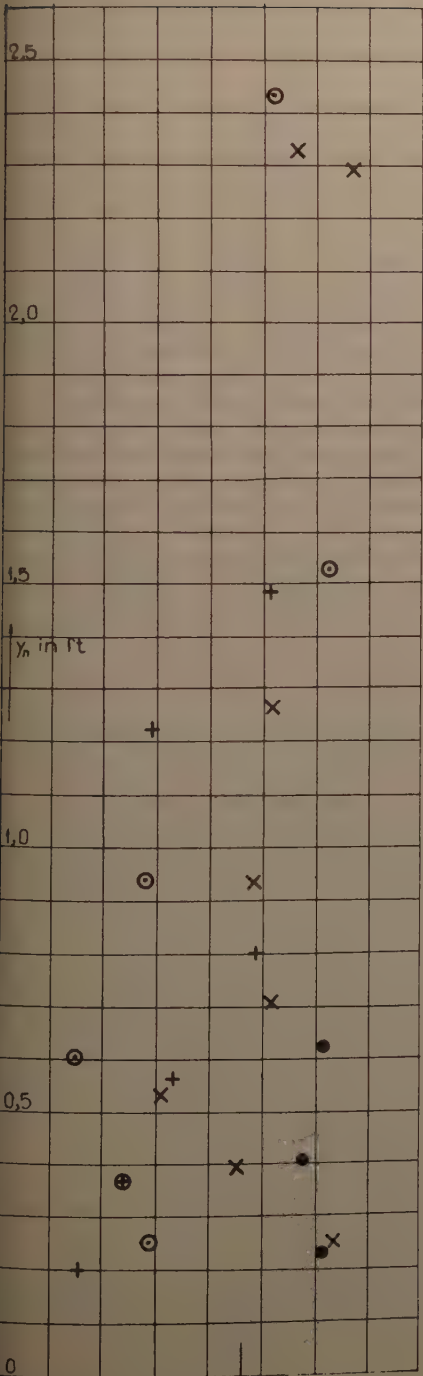


FIG. 2 :

$$\gamma = \frac{f_o}{f_p}$$
 for hydraulic smooth channel.

y_n : depth for uniform flow;
 f_o : observed in channel;
 f_p : f for smooth circular pipe with same Reynolds number.

	Slope	run nrs	γ_{av}
⊙	0,000448- 0,000480	1-6	0,889
+	0,000855- 0,000912	7-12	0,868
x	0,00100- 0,00103	13,102- 108	0,957
●	0,00198	14-16	1,002

} 0,919

plotted against y_n for different S (Fig. 2). The figure reveals that the low average is mainly determined by low values of y_n , whilst there seems to be a tendency for ψ to increase with increasing slope. It doesn't seem justified to state anything more definite, but the trend is the same as the one for the hydraulically rough flume.

The values of ψ as found here make it possible to find two equations for the application of Morris' theory⁽³⁾ is considered suitable: one for $S = 8.5$ to $10.4 \cdot 10^{-4}$ and one for $S = 19.3$ to $19.8 \cdot 10^{-4}$. The relevancy of Morris' formulas for practical computations is doubted by the writer as they seem too complicated; an empirical relation of ψ will suffice for "isolated-roughness flow" and "skimming flow".

The results of this analysis indicate that the conclusion of the authors regarding the equal relevancy of the formulas of Prandtl, Manning and Morris for a rough triangular channel may be premature. The writer had no opportunity to deal with this question in detail; maybe a definite answer can only be given after a new series of tests will have been carried out as even the excellent data given by the authors show for some runs (57-64) a considerable scattering of points. This question of equal relevancy of formulas may seem to be a matter of pure theoretics without much practical meaning for the flume studied, as the mean discrepancies in the mean velocity of all three formulas only amounts to 2 to 2.3%, whilst, according to the authors, the probable percentage error in determining the discharge for some of the runs may amount to the same percentage. When studying river problems however the influence of S on ψ may be of importance and for this reason attention is drawn to the conclusions drawn from further analysis of the data given by the authors.

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RESISTANCE PROPERTIES OF SEDIMENT-LADEN STREAMS^a

Discussions by Tsung-Lien Chou and D. B. Simons and E. V. Richardson

TSUNG-LIEN CHOU,¹ F. ASCE.—The authors are to be highly complimented for taking an important but long neglected problem into the laboratory and the library with special technical dexterity and mathematical insight of eliminating the interwoven elements and of bringing the required factors in focus. The results are neatly tabulated and the conclusions drawn are concise and precise. Since the problem is not only important for practical hydraulic engineers in dealing with river and canal works, but also is closely related with the more theoretical and broader side of boundary layer and turbulence in general, the writer would like to bring out some points which may help clarify the issue.

On Table 4, it is shown that in each set of observation, all important hydraulic characteristics are identical except the slope s and the sediment discharge concentration \bar{c} . With these two, the authors attribute a direct cause-effect correlation of friction factor f_b and \bar{c} . As repeated on Table 5, the figures are consistent and impressive, though not quite proportional. What this amounts to is that the structure of this part of the problem depends on the accuracy and reliability of the measurement of s which is always delicate and complicated. At this junction, it may be mentioned that in empirical formulas like Darcy-Weisbach $H_f = f \frac{L}{4r} \frac{U}{2g}$ or Chezy $U = C \sqrt{rs}$, the coefficients for c is used to fit in the observed values and they are supposed to take care of all losses which will, in modern hydrodynamical terminology, include losses due to skin friction, form resistance and turbulence in the pure fluid side and losses in knocking out solid particles from embedment, bringing them into suspension, and accelerating them to the same velocity as the dispersion medium, and host of other losses which are yet unknown. As widely observed, the velocity u varies with space and time, even in a uniform conduit under steady flow. With the empirical relation of u and s as indicated above, synchronically together with u varies the slope s , which may include many ups and downs, even negative values in backflow pockets. In natural streams, only mean values are taken for both. Therefore, in real sense, both u and s are statistical in nature and nothing more than mathematical fictions. Similar are the friction factors, Darcy-Weisbach for Manning n . Furthermore, these factors are not universal constants as shown in the following tabulation. Thus, each friction factor has its own structure and the relations among them are not homogeneous. The fact that they can be used to calculate flow in their special forms does not necessarily mean they can represent certain factors of

a. Proc. Paper 2020, May, 1959, by Vito A. Vanoni and George N. Nomicos.
1. Hydr. Engr., Clinton Bogert Engineers, New York, N. Y.

Empirical Formulas	Chezy	Darcy-Weisbach	Hazen-Williams	Manning
Original Forms	$v = c \sqrt{rs}$	$H_f = f \frac{L}{4r} \frac{v^2}{2g}$	$v = 1.318 c_1^{0.63} r^{0.54} s^{0.54}$	$v = \frac{1.486}{n} r^{2/3} s^{1/2}$
Friction Factors	c	f	c_1	n
Energy Slope	$s = \frac{v^2}{c^2 r}$	$s = \frac{v^2}{r} \left(\frac{f}{8g} \right)$	$s = \frac{v^{1.85}}{1.667 c_1^{1.35} r^{1.167}}$	$s = \frac{v^2 n^2}{2.203 r^{4/3}}$
For equal s , Ratios of Factors	$c = \sqrt{\frac{8g}{f}}$ $= 1.11 c_1^{0.927} r^{0.081}$ $= \frac{1.486}{n} r^{1/6}$	$f = \frac{8g}{c^2}$ $= \frac{17.5}{c_1^{1.95} r^{0.165} v^{0.15}}$ $= \frac{3.63 n^2}{r^{1/3}}$	$c = \frac{c_1^{1.08} v^{0.081}}{1.117 r^{0.093}}$ $= \frac{15.25}{f^{0.54} r^{0.089} v^{0.081}}$ $= \frac{1.372 r^{0.091} v^{0.031}}{1.08}$	$n = \frac{1.486}{c} r^{1/6}$ $= 1.486 r^{1/6} \left(\frac{f}{8g} \right)^{1/4}$ $= \frac{1.151 r^{0.004}}{c^{0.927} v^{0.075}}$

modern hydrodynamics. Now on Table 4, f is held to count for the variation of minute vortices and others. This is apparently outside the function assigned to it in the original design.

In applying the empirical formula in a flume like this, s should cover only that portion with established normal flow, and with the entrance and tailwater regions deducted and the cross-section of observation would be set at the geometrical center of the portion. Any average value from tilting of flume bed or the differences of water levels at the ends of the flume may bring in a drop-down or back water effect, which would not represent the true value.

By solidifying the flume bed after running with sediment, the authors tried to eliminate all factors due to bed configuration in order to emphasize the influence of sediment on hydraulic resistance. It may be worthwhile to realize that the turbulence produced on a movable bed is not the same as on similar bed which is fixed, even though the configurations are the same. On a movable bed, turbulence is evened out by picking up solid particles at points of strong intensity and dumping some of its load at points of weak intensity, or the distribution of turbulence contours is more or less conformal to the bed topography. Contrarily, on fixed bed, irregular configuration tends to amplify the turbulence at points of strong intensity and therefore greater turbulence is expected on fixed bed. Thus some part of the calculated value might be offset by this cause.

The velocity profile shown on Fig. 2 are apparently not taken from the experiments of this flume (10-1/2 inch wide). As the depth, the slope and the width are identical for the two profiles, the one with higher velocities will have higher mean velocity and thereby bigger discharge. In that case, more power is required by the flow⁽⁷⁾ and there is hardly any strict comparison of the two.

The authors maintain that the suspended sediment reduces hydraulic resistance by its damping effect on turbulence by the following reasoning: "To keep sediment in suspension, i.e. to prevent it from settling due to gravitational force, work must come from the vertical component of turbulence

fluctuations and must result in damping of turbulent motion" and "this means that the momentum transfer coefficient is also decreased thus allowing the velocity and velocity gradient to increase."

It should be remembered that turbulence is created at the expense of energy stock of the flow. That is to say, as soon as the turbulence is generated, a certain amount of energy is lost from the flowing stream, no matter whether energy is used in keeping sediment in suspension or in supplying Vortex motion of minute masses and finally dissipated into heat. To keep sediment continuously in suspension, an uninterrupted supply of the necessary energy through turbulence is required. However this damping action may have the effect of preventing the eddies spread out into the entire stream by localizing them to the lower strata. Finally in a stream where the saturated suspension tends to dump its load, there is a chance of releasing some energy for accelerating the flow. These conditions are limited to certain part or certain period of flood where suspended load is dumped.

Furthermore, the transportation processes of solid particles are different for different effective sizes and distributions. Only fine particles less than 0.2 mm can be thrown up into suspension by and move with turbulence generated purely by hydraulic resistance. On the other hand, bigger particles create extra turbulence by their projections into flowing stream and produce suction effect helping them into sliding, rolling and saltation. This demands more energy consumption and thereby increases the hydraulic resistance. The above conditions are well observed in pumping sands in suspension and suspension tests.

In addition to the direct experimentation in laboratory, the authors collect some interesting information of field data from national streams. Here the direct cause-effect correlation of f and \bar{c} is further overshadowed by additional factors of (a) source of solid supply, varying with the topographical and geological conditions of the river valley, (b) hydraulic properties of the stream, aggrading or degrading. The mutual action and reaction of these factors can easily produce contradictory effects.

In general, a rising stage has higher average velocities for two reasons. First, in rising stage, the energy slope is a drop-down curve with increasing velocities towards each downstream section. Thus the velocity is accelerated. Secondly, the potential head is converted into kinetic head with smaller loss.

In most cases, the amount and the intensity of sediment load vary with the stage. If a freshet is originated from an upstream watershed and gains strength in flowing downstream, it will pick up silt to its optimum capacity. This happens very often in the rising stage. After passing over the peak flow, the river tends to dump its suspended load. However, there may be exceptions to this rule in dealing with difference sources of solid supply. For instance, on the Yellow River at Tung-Kuang, the intensity of suspended sediment varies with the source of flood. If it comes from the north bank tributaries (Shansi Province) where steeper grades of river beds with thicker eolian deposits of loess, the River has a tendency to dump its load even in rising period. The contrary is true, if the flood is started from the south side. Similar situation is observed at Chingling Chi on the Yangtze River where the discharge from the south is clarified by passing the Tung-Ting Lake which acts as settling basin.

The above few examples just illustrate the complicated relations of various factors in natural streams and the cause-effect correlations can easily be reversed, unless comprehensive assessments are made for some period.

On the theoretical side, the sediment can reduce hydraulic resistance in the following ways:

- (1) Smoothing up the surface roughness by scouring off the mound and filling up the depressing
- (2) Evening off the aquatic growths
- (3) Exposing fresh surfaces which are more pliable
- (4) Compacting particles by selective deposition and distribution
- (5) Keeping down the vertical component of turbulence (as the authors' main item). Again, every factor must be properly appraised.

The problem of suspended load hydraulic resistance is complicated but very interesting. The authors have opened up a new vista for both practical and theoretical investigation. Perhaps, another way of attack may be the direct measurements of the turbulence itself for the similar condition of turbulent and clear flows.

D. B. SIMONS,¹ M. ASCE and E. V. RICHARDSON,² A. M. ASCE.—The authors are to be commended for their excellent explanation of many of the important aspects of the mechanics of flow in alluvial channels. Based upon similar studies conducted in both large and small flumes by the writers, the equipment used and a few of the concepts presented by the authors are elaborated on.

One of the problems associated with a study of this type is the difficulty of evaluating the effect of flume size on data and observed flow phenomenon. The writers found that the results obtained with a given bed material using a large recirculating flume 150 feet long, 8 feet wide, and 2 feet deep, could be reproduced in a smaller recirculating flume 60 feet long, 2 feet wide and 3 feet deep. For example, the forms of bed roughness observed in the order of increasing shear in the small flume were:

1. Plane bed before beginning of motion
2. Plane bed after beginning of motion
3. Dunes
4. Transition from dunes to plane bed and standing waves
5. Standing sinusoidal sand and water waves which were in phase
6. Antidunes

Using the same bed material in the large flume the forms of bed roughness in order of increasing shear were:

1. Plane bed before beginning of motion
2. Ripples
3. Ripples superposed on dunes
4. Dune
5. Transition from dunes to plane bed and standing waves
6. Standing sinusoidal sand and water waves which were in phase
7. Antidunes.

The large flume produced the forms of bed roughness observed in the small flume, whereas, the small flume did not. It was also noted that the height of dunes

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2. Hydr. Engr., Civ. Eng. Dept., U.S.G.S., Fort Collins, Colo.

related to the depth of sand bed. As depth of bed material is decreased the amplitude of the dunes is decreased. These differences in results, over and above those which can be compensated for by correcting for side wall effect, present a serious problem to those who are trying to utilize data collected by the various investigators, particularly that data gathered in very small flumes.

No mention is made by the authors of the fact that the form of bed roughness is related to size and gradation of bed material. This may be due to the limited range of conditions which they investigated. To illustrate using sand sizes ranging from $d = 0.2$ mm to $d = 0.5$ mm the writers have observed that the amplitude of the dunes is independent of size of bed material but the spacing and shape of the dunes are related to size of bed material. The spacing of the dunes increases with decreasing size and the angle the fore plane of the dune makes with the horizontal decreases with decreasing size of bed material. The net result is that resistance to flow is smaller with the fine sand than with the coarser sand by as much as 20 per cent. This discussion perhaps raises the question of bed form terminology. To avoid confusion, the major forms of bed roughness which have been observed in alluvial channels by the writers are illustrated in Fig. 1.* From this figure, it is apparent that the form of bed roughness described by the authors, according to this classification, is ripples.

Referring to the difference in the bed friction factor for sediment-laden flow over a loose sand bed and clear water flow over a fixed sand bed, of the same configuration, it is doubtful that the total difference can be explained by

*From "Forms of Bed Roughness in Alluvial Channels" by D. B. Simons and E. V. Richardson, personal communication.

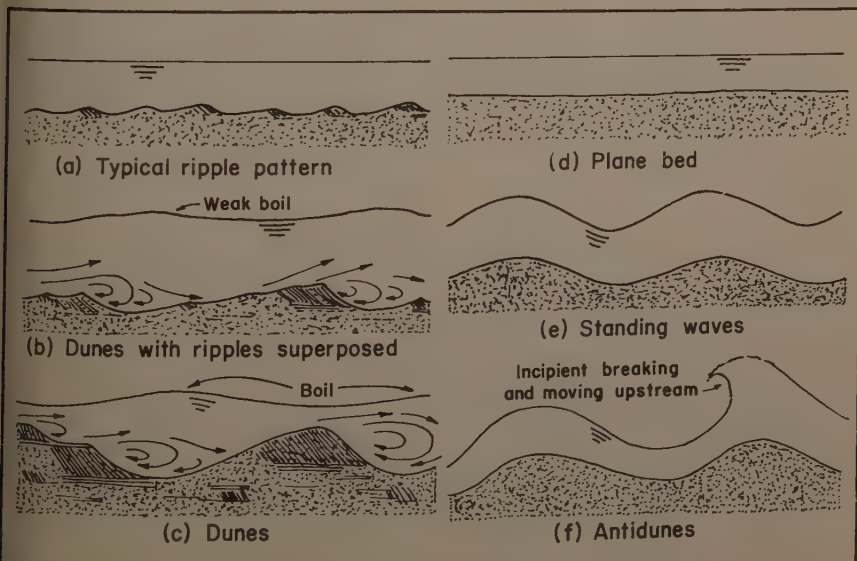


Fig. 1 Forms of Bed Roughness Observed in Alluvial Channels

the presence of the suspended sediment. The writers have observed that the scale and intensity of turbulence are entirely different on a rigid bed with bed material transport than on a loose sand bed having the order of 10 ppm of bed material transport, none of it in suspension, and other flow factors the same.

The effect of sediment load on resistance to flow is significant as illustrated. However, the authors' conclusion that all of the computed decrease in resistance to flow can be attributed to the damping effect of the suspended sediment on the turbulence is not concurred in. The presence of fine sediment causes effects on fluid properties which change the entire fluid bed material interrelationship. These factors appear to be of more importance than the dampening effect of the suspended sediment.

The presence of very fine sediment (clays) in the water also influences resistance to flow in alluvial channels. Based upon recent studies by the writers, concentrations of fine sediment on the order of 40,000 ppm can reduce resistance to flow as much as 40 per cent in the tranquil flow regime. However, under rapid flow conditions, the presence of fine sediment can increase resistance to flow.

The validity of the statement on page 104 which explains the existence of a smaller friction factor for the same discharge on a rising stage than on the falling stage, where the decrease in friction factor is presumed to result from a readjustment of the bed due to the larger sediment load is questioned. It is not more probable that the increase in load and the reduction in friction factor are both related to the fact that with the increase in shear with stage the bed form changes and that the change in bed form will lag the change on stage?

To better cope with the problems associated with fully developing and utilizing our water resources, additional work of this caliber, preferably done in larger flumes, should be initiated. Only by making studies of this type can we hope to explain the quantitative effect of size of bed material, gradation of material, temperature, geometry of channel, depth of sand bed, fine sediment load, and rate of change discharge with time on total bed material transport, the forms of bed roughness which develop, and the related resistance to flow.

HYDRAULIC MODELS OF THE ST. LAWRENCE POWER PROJECT^a

Discussions by G. T. Berry and G. B. Fenwick and F. R. Brown

GEORGE T. BERRY,¹ A. M. ASCE.—The St. Lawrence Power Project was unusual and challenging in many respects. Although the St. Lawrence River normally reaches its minimum annual discharge during January or February, the average January discharge for 98 years of record is 221,000 cfs and the average February discharge is 218,000 cfs. Comparing these averages with the average annual discharge of 246,000 cfs it is apparent that a low flow season for diversion or closure does not really exist. To divert such large flows required that the construction be performed in several stages.⁽¹⁾ At each construction stage the depth of water in the navigation channels and canals in the project area had to be maintained. The water surface elevation of Lake Ontario could not be altered from its natural condition and the discharge to downstream power plants could not be impaired.

The hydraulic model studies described by Mr. Bryce focused attention on the salient features of possible alternate solutions and permitted the hydraulic engineers to experiment with these variations until the optimum design was achieved.

This series of model studies is an outstanding example of the technique of complementing the general results of a small scale model by investigation of the critical details in larger scale models. The best location for the structures and the location, size and shape of the channels required by navigation and ice cover considerations were determined in the river models. These were at the scale of 1:500 horizontally and 1:100 vertically which is quite large for models of such a large river. Details of the stilling basins and gates were then studied in 1:50 scale partial models. Finally comprehensive models of Long Sault and Iroquois with enough upstream and downstream river channel to simulate prototype approach and discharge conditions were built and tested to determine the flow pattern and hydraulic performance for each construction stage.

Designs of the energy dissipators at Long Sault and Iroquois were developed from a long series of tests for each. Several forms of baffles and sills were tried. The final form which had no baffles and a solid end sill was the results of careful experimentation to obtain the lowest possible velocity in the channels below the dams. The ineffectiveness of baffles in these spillways which have high entrance velocities agrees with experiments conducted a few years ago at M.I.T. on the effects of Baffle Piers by Harleman.⁽²⁾ The excellent performance of the Long Sault stilling basin can be judged from the fact that, although the natural tailwater depth was insufficient to form a

a. Proc. Paper 2022, May, 1959, by John B. Bryce.

1. Civ. Engr., Chas. T. Main, Inc., Boston, Mass.

hydraulic jump, a jump was forced to form within the basin for the three different diversion conditions of (a) diversion through low sluices, (b) diversions through ports left in the dam and (c) flow over the completed dam. The channel downstream from the dam was rock in which the natural project velocity was in the same range as those achieved with the recommended spillway design. Similarly satisfactory energy dissipation was achieved at the Iroquois Dam.

The remarkable reduction of the hydraulic downpull effected by reshaping the bottom of the Long Sault port gates illustrates how important this factor is in designing gates and their hoisting equipment. Unbalanced hydraulic forces on gates of this type depend primarily on the local conversion of pressure head to velocity head in the restriction beneath the gate. The shape of the bottom of the gate and the approach conditions determine the magnitude of pressure reduction and the area over which it acts. Size of gate, depth of emergence, and location of skin plate and seals determine the magnitude of hydrostatic force which is not counterbalanced by vertical force on the bottom of the gate. The tests described cover a variety of gate conditions with upstream and downstream skin plates and with various depths of submergence. It is believed that structural and hydraulic engineers would find the data valuable if the author would show the physical details and dimensions of the gates, the pertinent boundary conditions and the head on the gate along with uplift and/or downpull at various gate openings. Available data correlating these factors are scarce.

It is impossible to accurately isolate the savings in construction cost attributable directly to the model studies. The substitution of dry excavation south of Gallop Island for wet excavation along the north channel resulted in a saving of about \$1.00 per cubic yard on almost 6 million cubic yards of excavation. This alone repaid manifold the cost of the model studies. Of equal importance was the fact that by these accurate model tests, the design velocities for both navigation and ice packing criteria could be met with confidence so that very little margin for unknown or unexpected conditions was needed. The definite detailed construction procedure which was developed and outlined in the specifications and the model demonstrations, which were in some cases made for the bidders to show the construction sequence and hydraulic conditions which would be encountered, undoubtedly resulted in substantial savings in cost.

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G. B. FENWICK,¹ F. ASCE and F. R. BROWN,² F. ASCE.—Mr. Bryce's paper portrays a classic example of the utilization of hydraulic models as a fundamental part of the design and construction of a major engineering project involving a wide variety of complex interrelated features. Wide experience has established the fact that model studies are an indispensable adjunct to analytical procedures in the development of optimum designs and construction sequences for major hydraulic channels and structures involving complex features and large expenditures of funds, and that model studies almost invariably produce major design improvements at costs that are disproportionately low in relation to project costs. For many hydraulic engineering problems, including some of the features of the St. Lawrence project, hydraulic model studies constitute the only reliable approach to satisfactory solutions.

It is noted from Mr. Bryce's paper that his lowermost river model (Model No. 9), utilized for studies in the powerhouse tailrace, terminated just below the head of Cornwall Island and included only the upper ends of the channels north and south of that island. He mentions that a portion of the excavation of the south channel, which contains the navigation channel approaching the Eisenhower Lock was accomplished by the United States Seaway Development Corporation. It might be of interest to note that two river model studies were conducted by the U. S. Army Engineer Waterways Experiment Station to develop optimum excavation and spoil-placement plans and construction sequences for the navigation channel in the river from Barnhart Island downstream to Lake St. Francis.

An interesting scale-distortion problem in connection with the modeling of this reach arose from the fact that 60 per cent of the river flow turns sharply to the south around the head of Cornwall Island and enters the navigation channel below the lock almost normal to the east-west navigation course. It was therefore necessary, among other things, to design a curved deflecting dike to realign this southward flow with the east-west navigation channel to eliminate intolerable cross currents in the latter. One model of the overall reach, with linear scales of 1:300 horizontally and 1:100 vertically, was used to develop plans throughout the Barnhart Island - Lake St. Francis reach, except in the vicinity of the above-mentioned sharp changes in flow direction near the head of Cornwall Island. Since it was considered that model scale distortion would introduce significant errors in the latter area owing to the sharply changing flow directions, this immediate area was duplicated in another model with an undistorted linear scale of 1:100. As was expected, comparisons of detailed data from both models demonstrated that the distorted model produced erroneous distributions of velocities in the vicinity of the sharp direction changes while data in other areas checked well in both models. Accordingly, the two models were used together very effectively to develop plan details for the overall reach.

The problem of energy dissipation downstream from the St. Lawrence structures is unique in that large discharges must be passed during the construction stage. On many other rivers, seasonal fluctuations are such that critical stages of construction can be accomplished when river flows are low. Because of insufficient tailwater it was necessary to force a hydraulic jump

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for diversion flows and for final operating conditions. In the case of the Sault Dam, this appears to have been accomplished by use of a 12.5-ft-high end sill. However, a question arises as to the necessity for a 40-ft-wide sill. A lesser width sill and longer apron at elevation 140 might improve conditions. Provision of a steeper slope at the toe of the overflow section or sloping diversion ports are other alternatives for improvement of energy dissipation.

The energy dissipator for the Iroquois Dam appears to be designed for diversion flows. It was stated that only ten sluices (or gate bays) were protected, which raises the question as to the treatment of the bed of the stream immediately downstream from the other 22 sluices. If economies were achieved by the omission of any protection below 22 sluices, the author is to be commended. Many times engineers are reluctant to omit any part of the energy dissipator, even where conditions are such that it can be done safely.

The effect of gate operation on energy dissipation is mentioned only briefly by Mr. Bryce. However, various combinations of gates and openings could have an important bearing on the strength and magnitude of river currents and on the design of an energy dissipator. On many river dams, gate operation to secure satisfactory energy dissipation is one of the most important phases of a model study.

Another important phase of the St. Lawrence work reported upon by Mr. Bryce concerns the hydraulic loading of gates. This topic could be the subject of an entire paper. The Waterways Experiment Station has made observations similar to those indicated in Fig. 16, both in the model and in the field.⁽¹⁾ However, in order to avoid too much uplift, a 45-degree lip is usually used. Efforts to use a gate with upstream seals and the upstream skin plate projecting below the bottom members were successful as far as hydraulic loading of the gate was concerned. However, the expansion of the jet laterally into the gate slot intensifies the possibility of cavitation of the conduit liner.

For submerged gates with downstream seals, the depth of water over the gate contributes materially to the downpull force. As the gate is raised and the top seal broken, the clearance on the downstream side of the gate becomes significant. The relative clearances on the upstream and downstream sides can maintain or release the column of water over the gate, and thus change the magnitude of the loading. It has been found best to maintain some water in the gate well over the gate for more positive control of the gate.

No mention was made of studies of the vibration characteristics of the St. Lawrence gates. With a flat lip, a gate tends to vibrate vertically in resonance with the vortex trail shed from the upstream edge. With a 45 or 60 degree lip, the vortices are shed from the downstream edge of the gate, thus eliminating any areas on which pressure pulsations can act. The shape of the lips of the gates in Fig. 17 cannot be determined, although on the basis of the tests of diversion port gates at Long Sault Dam, it is assumed that they are either 45 or 60 degrees. Other contributing factors to the hydraulic loading of gates are venting downstream from the gate and entrapment of water in the gate members.

Another problem area not mentioned in Mr. Bryce's paper concerns the emergency closure of ports or gated areas by means of stop logs or a series of built up members. These devices are designed to be lowered through flowing water to provide a barrier to flow in the event of damage to a gate. Tests and field experiences indicate that it is almost impossible to install emergency closure devices in flowing water. Flow over the top of and beneath the stop logs or built up steel members sets up vortex trails which

the closure devices to oscillate violently in a vertical direction. The amplitude of movement is such that the closure device could be damaged or wedged in a skewed position.

It is realized that space requirements prevented the author of the basic paper from going into detail on specific problems. However, it is hoped that in his closing remarks the author can expand on his tests to develop energy dissipators and investigate the hydraulic loading of gates.

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GROUND-WATER PROBLEMS IN NEW YORK AND NEW ENGLAND^a

Discussion by C. Biemond

C. BIEMOND.¹—As stated in the introduction to the paper, there is generally interchange between surface water and ground water under natural conditions. There can also be interchange by artificial processes: surface water can be brought to intermingle with and to diffuse in ground water by artificial replenishment, as practised in Sweden and in The Netherlands. The ground water captured in works of this nature is a mixture of natural ground water and artificial ground water, which has in many respects the same quality as natural ground water. Artificial recharge is accomplished in these cases by spreading in shallow ponds or canals. The writer does not know of examples in Europe of recharge with surface water by means of wells.

Intermingling of ground water and surface water is also brought about by what is called "Uferfiltration" in Germany and practised on a large scale along the Ruhr River and the Rhine River. From the paper it is deduced that, in the case of the Blackstone River near Woonsocket, R. I., this process, in which wells for the withdrawal of ground water are placed near or in the river bed, is being envisaged.

It is the purpose of the writer to report on some features resulting from this technique, that might be of interest.

The alluvial deposits in and along river beds—where the river has been aggrading—consist as a rule to considerable depths of a mixture of boulders, gravel and coarse sand—a favorable condition for ground-water withdrawal. When the wells are placed near the river and reach into the underlying deposits, the water withdrawn would partly percolate out of the natural ground-water body adjacent to the river bed, but, as a result of the hydraulic flow pattern, mostly through the river bed, and therefore would consist for the most part of ground water that was recently surface water in an open river. This water would contain the natural impurities of river water, such as silt and decayed algae, but also the products of artificial pollution, which might be organic substances, ammonia, iron, and others. Furthermore, there may be, as a result of heavy pollution, a very low oxygen content.

Pollution is kept away from the Ruhr River as much as possible; water percolating through the gravel and sand in the river bed reaches galleries and wells in a satisfactory condition.

The Rhine River is in a state of increasing pollution and the water withdrawn from wells bordering the river bed is gradually giving more trouble. The troubles consist firstly in deteriorating quality of the extracted water and secondly in a gradual clogging of the river bed. Deterioration of quality is

a. Proc. Paper 2056, June, 1959, by Joseph E. Upson.

1. Director Amsterdam Waterworks, The Netherlands.

particularly apparent in the taste. Clogging, in the opinion of the writer, be caused by high entrance-velocities for the water penetrating into the alluvial deposits, thus introducing insoluble particles into the interstices of deposits. It might also be caused by introducing oxygen-poor water into a medium which doubtless contains ferric deposits that might thus go into solution, to be deposited subsequently in interstices where intermingling with oxygen-rich water would take place.

In the opinion of the writer artificial replenishment, in comparison with wells located in alluvium along rivers, has certain advantages that may be worthwhile to consider.

FLOODS OF THE FLORIDA EVERGLADES^a

Discussion by J. R. Bowman

J. R. BOWMAN,¹ M. ASCE.—There are so few large inland bodies of water in the United States that are subjected to hurricane winds that opportunities to compare their tidal characteristics are rare. Pamlico Sound, in eastern North Carolina, is one of the few such waters that can be likened to Lake Okeechobee. This sound is land-locked by virtue of a string of sand barrier reefs, known as the Outer Banks, which separate the sound from the Atlantic Ocean. It has a surface area of nearly 2,000 square miles, and it drains an area of some 25,000 square miles in Virginia and North Carolina. Its principal outlets to the ocean are three relatively narrow and shallow inlets through the Outer Banks; thus the normal sound levels can be considered to be independent of ocean tides, except for local effects in the immediate vicinities of the inlets. Unlike Lake Okeechobee, Pamlico Sound is not a "huge saucer", but assumes the form of a rough parallelogram bounded on the north and west by the relatively steep shores of the marshy flatwoods on the mainland, and on the east and south by the shoal waters contiguous to the Outer Banks. The area of maximum depth, varying from 20 to 25 feet, is found in a narrow reach along the northeast-to-southwest major axis of the sound.

Hurricanes are not strangers to eastern North Carolina. In fact, a list of these storms that have directly affected Pamlico Sound alone since 1900 would be as long as the author's list associated with his Fig. 6. Analysis of Fig. 6 and its companion list shows that the Okeechobee area is crossed by two general groups of hurricane paths. One is associated with storms from the Atlantic Ocean that generally strike southeastern Florida prior to their recurvature. Storms of the other group approach from the Caribbean Sea and the Gulf of Mexico, striking the west side of the peninsula after having recurved eastward. It is noted that the highest wind speeds generally have been more closely associated with the storms of the former group. Because of the orientation of the Florida peninsula in relation to its watery environs, excessive rainfall is not necessarily peculiar to either group. However, it is interesting to note that the 1924 hurricane produced exceptionally heavy rainfall in areas north of Lake Okeechobee, whereas the storm center passed at least 50 miles south of the lake; the pronounced eastward swing in the path of this storm (number VI) suggests the proximity of a stationary front, which could account for the widespread area of excessive precipitation.

Because Pamlico Sound is situated some eight degrees north and five degrees east of Lake Okeechobee, the majority of its hurricanes approach from the Atlantic Ocean during or after recurvature. When a hurricane recurves

a. Proc. Paper 2058, June, 1959, by E. W. Eden, Jr.

1. Civ. Engr., Erik Floor & Assocs., Inc., Chicago, Ill.

in the higher latitudes, its forward speed usually increases considerably, carrying the storm rapidly past or across the sound; therefore, the regions maximum winds do not have sufficient time in which to develop severe wind tides over the longer fetches, unless the radius to the maximum wind field is exceptionally large. Three of the most severe storms in this region have amply met this requirement; along the Outer Banks on the south side of Pamlico Sound, the winds and sound tides of the "Great Atlantic Hurricane" of 1944 are remembered as the most severe in half a century.

As a tropical storm enters its zone of recurvature, it is not unusual for forward speed to decrease somewhat; on occasion, a blocking ridge of high pressure will cause the storm to stall and wander aimlessly. Hurricane Iona of 1955 reached a virtual standstill immediately after crossing the North Carolina coastline in the vicinity of Cape Lookout (located about 30 miles due south of the western end of the sound); it then proceeded to drift slowly and erratically westward, covering a path distance of only 50 miles in nine hours before veering northeastward and picking up speed. During the "stagnant" period, the western portion of Pamlico Sound experienced winds of 75 to 100 miles per hour, and storm tides reached unprecedented heights in many municipalities in that area.

Not infrequently the varying conditions of surface friction presented to a hurricane passing over land masses will cause the eye of the storm to become distorted near the surface, thereby altering surface wind patterns. The scarcity of surface observations made during the passage of Hurricane Barbara of 1953 leaves the writer somewhat uncertain as to the configuration of the surface eye immediately after it crossed the coastline in the immediate vicinity of Cape Lookout. The storm was an infant, as hurricanes go, its maximum steady winds just reaching hurricane force. Weather Bureau summaries of the storm do not suggest the presence of unusual characteristics, nor does the reconstructed path, based on available information, reveal significant irregularities. Had the eye passed undistorted across the western part of Pamlico Sound, the maximum setup should have occurred along the north shore. Although flooding did occur where expected, the highest setup appeared to have occurred along the northern portion of the east shore of the sound; the writer observed a fresh high water line, estimated at 10 to 12 feet above normal sea level, extending for several miles. Moreover, structural damage, which was largely confined to an area on the Outer Banks about 20 miles further north, all occurred within the same half-hour period; the damage was inflicted by a southeast wind at least 60 miles in advance of the apparent eye. Curiously enough, the path of the storm eventually passed 25 miles west of the damage zone, but the eye was not observed within 10 miles of that area.

Florida also has experienced some unusual behavior on the part of hurricanes; among the more recent examples were the small but intense Miami storm and the looping Cedar Key storm, both in 1950.⁽¹⁾ The former (number XVIII in Fig. 6) maintained a tight circulation pattern around a very small eye from Miami to Lake Okeechobee; however, the published observations indicate a considerable reduction in wind speed in the Okeechobee area, with winds only gale force observed in the vicinity of the lake. After crossing the lake the eye broadened and full hurricane intensity was regained.

Although the Cedar Key storm of September, 1950, was well removed from the Everglades, the writer wonders how it might have affected the project area, assuming the loop, the subsequent path and the associated precipitation pattern were superimposed on Lake Okeechobee and the Everglades. Cedar

Key received 25.2 inches of rain in a little over 24 hours, and depths of 7 to 20 inches were recorded elsewhere along the path of the storm.

The author's estimates as to the effectiveness of the Central and Southern Florida Flood Control Project are impressive, but one wonders whether the general public fully appreciates their value. In the writer's contact with municipal flooding problems, he has observed that having assumed a considerable obligation in approving a bond issue, one derives small comfort from the knowledge that the new storm relief system probably will reduce the depth of water in his basement from four feet to only two.

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ROLL WAVES AND SLUG FLOWS IN INCLINED OPEN CHANNELS^a

Discussion by F. F. Escoffier

F. F. ESCOFFIER,¹ M. ASCE.—The author has made an interesting study of certain waves of instability that occur in fairly shallow water. Apparently either surface tension or laminar flow plays an essential part in the generation of these waves. In the interest of providing a somewhat more complete picture of the subject of waves of instability the writer feels that it may be helpful to present his understanding of how some waves of instability originate.

The type of waves the writer has in mind normally occurs in relatively deep and fully turbulent flow. An interesting photograph of such a roll wave having a height of five feet was published in 1936 by W. H. Holmes⁽¹⁾ together with some rather detailed measurements on this and other roll waves observed by him. News is received from time to time of highly destructive roll waves that occur in canyons in the western part of the United States. These are generally referred to as "walls of water."

The writer will use substantially the same reasoning that was used in 1950⁽²⁾ except that in the interest of simplicity it will be assumed that there exists a uniform distribution of velocity in the channel cross-section, an assumption that greatly simplifies the results.

The dynamic equation and the equation of continuity, for a uniform channel, which serve as a starting point for the derivation are

$$\frac{\partial v}{\partial t} + v \frac{\partial v}{\partial x} + g \frac{\partial y}{\partial x} = gS \left(1 - \frac{v^2}{v_0^2} \right) \quad (1)$$

and

$$w \frac{\partial y}{\partial t} + A \frac{\partial v}{\partial x} + VW \frac{\partial y}{\partial x} = 0 \quad (2)$$

where V is the mean velocity, t is time, x is the distance measured downstream, g is the acceleration of gravity, y is the depth of water at the lowest point of the cross-section, S is the bottom slope, V_0 is the normal velocity, w is the water-surface width, and A is the cross-sectional area.

Eq. (2) is now multiplied by r and the result is added to Eq. (1) to obtain

$$\frac{\partial v}{\partial t} + (V + rA) \frac{\partial v}{\partial x} + rw \frac{\partial y}{\partial t} + (g + rvw) \frac{\partial y}{\partial x} = gS \left(1 - \frac{v^2}{v_0^2} \right) \quad (3)$$

The condition for the left-hand side of Eq. (3) to be a directional derivative in the (x,t) plane is

a. Proc. Paper 2085, July, 1959, by Paul Mayer.

1. Hydr. Engrs., Mobile Corps of Engrs., Mobile, Ala.

$$\frac{1}{v + ra} = \frac{rw}{g + rvw}$$

which yields the following solution for r

$$r = \pm \sqrt{\frac{g}{wa}}$$

Substituting this back into Eq. (3) leads to the equation

$$\begin{aligned} \frac{\partial v}{\partial t} + (v \pm \sqrt{\frac{ga}{w}}) \frac{\partial v}{\partial x} \pm \left[\sqrt{\frac{gw}{a}} \frac{\partial y}{\partial t} + (\pm \sqrt{\frac{ga}{w}} + v) \sqrt{\frac{gw}{a}} \frac{\partial y}{\partial t} \right] \\ = gS \left(1 - \frac{v^2}{v_0^2} \right) \end{aligned}$$

or

$$\frac{\partial v}{\partial t} + (v \pm c) \frac{\partial v}{\partial x} \pm \left[\frac{\partial w}{\partial t} + (v \pm c) \frac{\partial w}{\partial x} \right] = gS \left(1 - \frac{v^2}{v_0^2} \right)$$

where

$$c = \sqrt{\frac{gA}{w}}$$

and

$$w = \int_0^y c \frac{dA}{A}$$

w is a stage variable which is used instead of y to represent the level of the water in the channel. In a rectangular channel it assumes the familiar form

$$w = 2c$$

If

$$z = v \pm w$$

and

$$F = gS \left(1 - \frac{v^2}{v_0^2} \right)$$

are substituted into Eq. (5) there is obtained the equation

$$\frac{\partial z}{\partial t} + (v \pm c) \frac{\partial z}{\partial x} = F$$

Use is now made of the identity

$$dz = \frac{\partial z}{\partial t} dt + \frac{\partial z}{\partial x} dx$$

to eliminate the partial derivative $\frac{\partial z}{\partial t}$ from Eq. (10) with the result

$$\frac{dz}{dt} - \frac{\partial z}{\partial x} \left[\frac{dx}{dt} - (v \pm c) \right] = F$$

from which it can be seen that along the path in the (x,t) plane defined by

$$dx = (v \pm c) dt \quad (12)$$

the expression in brackets disappears and

$$dz = F dt$$

which in the original terminology becomes

$$d(v \pm w) = gS \left(1 - \frac{v^2}{v_0^2}\right) dt \quad (13)$$

Eqs. (12) and (13) are the equations of characteristics. The characteristic corresponding to the upper or plus sign represents a pulse or wave point traveling downstream with a velocity $V + c$ and shall be designated a forward characteristic. The one corresponding to the lower or minus sign represents a pulse or wave point traveling with a velocity $V - c$, and consequently moving either upstream or downstream accordingly as V is less than or greater than c . It shall be designated a backward characteristic.

It is now assumed that small variations δV , δV_0 , and δw occur in an otherwise uniform flow in which

$$V = V_0$$

and, therefore

$$\delta \left[gS \left(1 - \frac{v^2}{v_0^2}\right) \right] = \frac{2gS}{V} (\delta v_0 - \delta v) \quad (14)$$

Along the forward characteristic

$$d(\delta v + \delta w) = \frac{2gS}{V} (\delta v_0 - \delta v) dt \quad (15)$$

and along the backward characteristic

$$d(\delta v - \delta w) = \frac{2gS}{V} (\delta v_0 - \delta v) dt \quad (16)$$

In Fig. 1 the lines MP and BO represent forward characteristics and the line Op a backward characteristic. The zone to the right of BO is assumed to be undisturbed uniform flow. A pulse or disturbance starts at M and moves downstream along MP. Ordinarily the effect of channel friction will be to reduce the magnitude of this pulse as it moves downstream. However, in unusually steep channels the opposite may be true and the pulse will continue to increase in magnitude. In such a case the flow is said to be unstable.

The line OP has been taken short enough for the right-hand side of Eq. (16) to be a negligible quantity so that

$$\delta v - \delta w = 0 \quad (17)$$

Eq. (17) is now used to eliminate V from Eq. (15) with the result

$$d\delta w = \frac{gS}{V} (\delta V_0 - \delta w) dt \quad (18)$$

Since V_0 is a function of w it is possible to write

$$\delta V_0 = \frac{dV_0}{dw} \delta w \quad (19)$$

The substitution of Eq. (19) into Eq. (18) yields

$$d\delta w = \frac{gS}{V} \left(\frac{dV_0}{dw} - 1 \right) \delta w dt$$

An examination of Eq. (20) will show that δw will increase or decrease in magnitude as it moves downstream accordingly as $\frac{dV_0}{dw}$ is greater than or less than one. The requirement for the formation of waves of instability is, therefore

$$\frac{dV_0}{dw} > 1$$

Eq. (20) can be integrated. The result is

$$\delta w = C e^{\frac{gS}{V} \left(\frac{dV_0}{dw} - 1 \right) t}$$

where C is a constant of integration.

The same line of reasoning with the roles of the forward and backward characteristics interchanged yields the formula

$$\frac{dV_0}{dw} < -1$$

as the requirement for the formation of waves of instability along the backward characteristics. This is an unusual condition in that it requires the normal

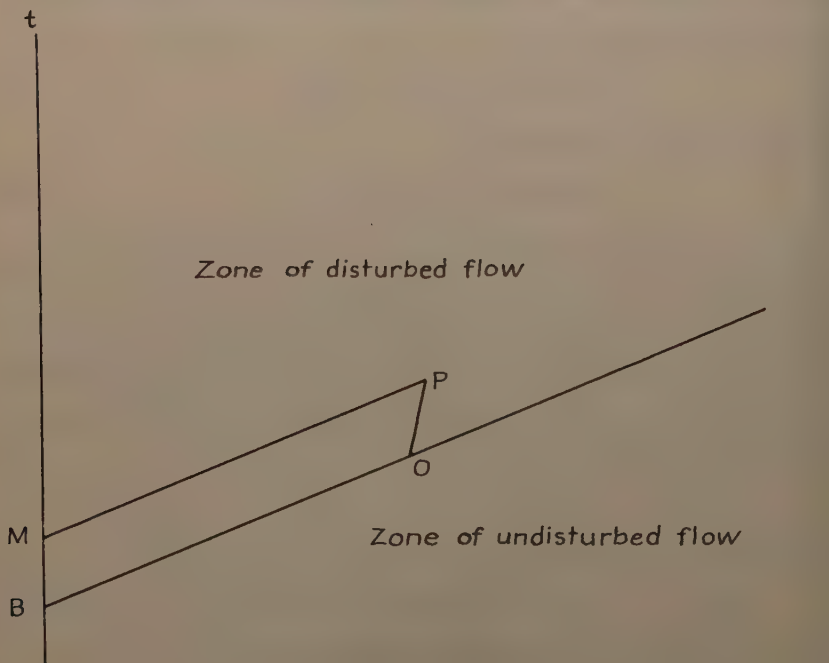


Fig. 1

velocity V_0 to diminish as the stage w increases. However, it is a condition that can be met in a closed conduit flowing nearly full.

It remains to show that Eq. (21) will upon appropriate substitution yield the more familiar formulas in general use. In a rectangular channel of infinite width

$$V_0 = k \sqrt{S} y^n \quad (24)$$

and

$$w = 2 \sqrt{gy} \quad (25)$$

Therefore

$$\frac{dV_0}{dw} = \frac{k \sqrt{S} n y^{n-1} dy}{\sqrt{g} y - \frac{1}{2} dy} = \frac{k \sqrt{S} n}{\sqrt{g}} y^{n - \frac{1}{2}} \quad (26)$$

The formula for the critical slope obtained by equating c as given by Eq. (6) to V_0 as given by Eq. (24) is

$$S_c = \frac{g}{k^2} y^{1-2n} \quad (27)$$

Eliminating g , k , and y between Eqs. (26) and (27) there is obtained

$$\frac{dV_0}{dw} = n \sqrt{\frac{S}{S_c}} \quad (28)$$

Eq. (21) becomes

$$n \sqrt{\frac{S}{S_c}} > 1$$

whence

$$S > \frac{S_c}{n^2} \quad (29)$$

For the Chezy formula $n = \frac{1}{2}$ and Eq. (29) becomes

$$S > 4 S_c$$

which is the result obtained by Jeffreys⁽³⁾ and by Thomas.⁽⁴⁾ If the Manning exponent $n = \frac{2}{3}$ is assumed Eq. (29) becomes

$$S > \frac{9}{4} S_c$$

which is the result obtained by Keulegan and Patterson.⁽⁵⁾

It is also possible to derive Vederikov's formula from Eq. (21). Vederikov^(6,7) starts with the equation

$$S = \frac{a V_0^p}{R^{1+\beta}} \quad (30)$$

where R is the hydraulic radius.

The use of this equation would require the substitution of the exponent p for the two exponents on the right-hand side of Eq. (5). However, this change leaves Eq. (21) unaltered.

Solving Eq. (30) for V_0 there is obtained

$$V_0 = \left(\frac{S}{a}\right)^{\frac{1}{p}} R^{\frac{1+\beta}{p}}$$

Taking the derivative of this expression, and then dividing this derivative by the expression itself yields

$$\frac{dV_0}{V_0} = \frac{1+\beta}{p} \frac{dR}{R}$$

Since

$$dw = c \frac{dA}{A}$$

it is possible to write

$$\frac{dV_0}{dw} = \frac{1+\beta}{p} \frac{A V_0}{R c} \frac{dR}{dA}$$

but

$$\frac{dR}{dA} = \frac{d}{dA} \left(\frac{A}{p}\right) = \frac{1}{p} \left(1 - R \frac{dp}{dA}\right)$$

and therefore

$$\frac{dV_0}{dw} = \frac{(1+\beta) V_0}{p c} \left(1 - R \frac{dp}{dA}\right)$$

where P is the wetted perimeter.

Letting μ be the velocity of a pulse moving along a forward characteristic so that

$$\mu = V_0 + c$$

c can be eliminated from Eq. (32) with the result

$$\frac{dV_0}{dw} = \frac{(1+\beta) V_0}{p (\mu - V_0)} \left(1 - R \frac{dp}{dA}\right)$$

The expression on the right-hand side of Eq. (34) has been called the Vedernikov number by Powell⁽⁸⁾ who states that when this number exceeds one the flow is ultra-rapid, roll waves form and the flow cannot be steady. Vedernikov's criterion is, therefore, equivalent to Eq. (21). However, both have been developed here for a uniform distribution of velocity in the channel cross-section. Since Vedernikov introduces the velocity-distribution coefficient into his formula in a way different from that followed by the writer in his original derivation, the two criteria would differ somewhat for a non-uniform distribution of velocity.

Craya⁽⁹⁾ has shown that Eq. (21) is equivalent to

$$u > V_0 + c$$

where

$$u = \frac{dQ_0}{dA}$$

is the celerity of Seddon, Q_0 being the normal discharge. Eq. (35) can be derived by substituting

$$dV_0 = \frac{AdQ_0 - Q_0dA}{AZ}$$

and

$$dw = c \frac{dA}{A}$$

into Eq. (21). If the same substitution is made into Eq. (23) there is obtained

$$u < V_0 - c \quad (37)$$

which is the corresponding condition for waves of instability along the backward characteristics.

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REVISED COMPUTATIONS OF A VELOCITY HEAD WEIGHTED VALUE^a

Discussions by Byron N. Aldridge and T. R. Anand

BYRON N. ALDRIDGE,¹ A. M. ASCE.—The author's main objection to the use of the present method of determining a weighted velocity head, $\frac{\sum K^3}{(\sum K)^3} \frac{\sum A^2}{(\sum A)^2}$ appears to be the cumbersome figures obtained. Since only the relative values of the numerator and denominator are needed, the work can be reduced considerably by arbitrarily shifting the decimal point in both the area and conveyance columns and making mental calculations of K^3/A^2 . The U. S. Geological Survey uses this method in much of its slope-area work. The method is illustrated below with the example used by the authors in their presentation.

Approximate values						
A_1	K_1	$\frac{K_1}{A_1}$	$(\frac{K_1}{A_1})^2$	$(\frac{K_1}{A_1})^2 K_1$	Slide rule reading	$\frac{K_1^3}{A_1^2}$
1,895	175,000	10	100	1,700	149	1,490
176	3,670	2	4	12	160	16
82.4	935	1	.1	9	122	122
						1,492 82
2,153.4	179,605	8	64	1,150	125	1,250
						alpha = $\frac{1,492}{1,250} = 1.19$

The lines through the A and K columns are drawn arbitrarily to obtain small numbers. The decimal can be shifted any amount and need not be shifted the same amount in the area column as it is in the conveyance column.

The columns headed "approximate" can be obtained by rapid mental calculation to give the location of the decimal point for the shifted computations, and need not be recorded.

Where there is a large variation in conveyance it is often more convenient to use decimals in place of whole numbers. In the above example the decimal in the area column could be shifted 2 places instead of 3, which would move

a. Proc. Paper 2149, September, 1959, by J. M. Lara and K. B. Schroeder.
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the decimal one place further to the left in the $\frac{K_1}{A_1}$ column and 2 places further to the left in the remaining columns.

The value of alpha is the same by either method and is correct even though the absolute values of K^3/A^2 are not known.

When the mental calculations show that $\frac{K^3}{A^2}$ of small areas will not effect the last significant figure of the total it is not necessary to compute $\frac{K^3}{A^2}$ for these areas. The data that the writer would record on the computation sheet for the example is shown below.

A	K	$\frac{K^3}{A^2}$	Alpha
1,895	175,000	14.9	
176	3,670	-	
82.4	935	-	
<hr/>		<hr/>	
		14.9	
2,153.4	179,605	12.5	1.19

Velocities in the subsections are computed for the final discharge (which is usually obtained by weighting the discharge from 2 or 3 subreaches) in the same manner as that shown by Lara and Schroeder.

T. R. ANAND,¹ A. M. ASCE.—A study of the "revised" formulas recommended by the authors shows that these are essentially the same as the "old" formula, viz;

$$\frac{V^2}{2g} = \frac{1}{2g} \cdot \frac{\sum \frac{Kp^3}{Ap^2}}{\sum \frac{Kp^3}{Ap^2}} = \frac{1}{2g} \cdot \alpha$$

Where α is the velocity head correction and suffix "p" a portion of the channel. This can be proved as follows, starting with the following "new" steps of the author's.

$$V_p = \frac{Q_p}{A_p} \quad \text{--- (2) col. 8, pp. 7}$$

$$h_v = \frac{\sum V_p^2 Q}{Q \cdot 2g} \quad \text{--- col. 10, pp. 7}$$

$$= \frac{\sum \left(\frac{Q_p}{A_p} \right)^2 Q}{Q \cdot 2g}$$

1. Engr., The Shawinigan Eng. Co., Ltd., Montreal, Quebec.

$$= \frac{\sum \frac{Q_p^3}{A_p^2}}{Q \cdot 2g} \quad \text{---} \quad (3)$$

$$\begin{aligned} \text{now } Q_p^{\wedge} &= A_p^{\wedge} \cdot V_p^{\wedge} \\ &= A_p^{\wedge} \cdot \frac{1.486}{n} R^{\frac{2}{3}} S^{\frac{1}{2}} \\ \text{or } Q_p^{\wedge} &= K_p^{\wedge} \cdot S^{\frac{1}{2}} \quad \text{---} \quad (4) \end{aligned}$$

Substituting (4) in (3) we get

$$h_v^{\wedge} = \frac{\sum S^{\frac{1}{2}} \cdot \frac{K_p^3}{A_p^2}}{2g \sum K_p S^{\frac{1}{2}}} = \frac{1}{2g} \frac{\sum K_p^3 / A_p^2}{\sum K_p^{\wedge}}$$

which is the "old" formula given by (1).

As for the convenience and saving in computational work, the writer fails to see how working out value of α from formula (1) in one step can be more cumbersome than filling up a seventeen column table. If velocities and discharges are required in a subdivision, they can be easily worked out using the various values of K_p .

Also, it is objectionable to introduce the conveyance term $K_p = \frac{1.486}{n} AR^{2/3}$ in hydraulic calculations when a similar term $K = 1.486 AR^{2/3}$ is already well established. This latter term depends solely on channel geometry and has a predictable shape when plotted for various types of channels. Mannings "n" can differ from reach to reach for the same channel and should be used separately.

The technical papers published in the past year are identified by number below. Technical-division sponsorship is indicated by an abbreviation at the end of each Paper Number, the symbols referring to: Air Transport (AT), City Planning (CP), Construction (CO), Engineering Mechanics (EM), Highway (HW), Hydraulics (HY), Irrigation and Drainage (IR), Pipeline (PL), Power (PO), Sanitary Engineering (SA), Soil Mechanics and Foundations (SM), Structural (ST), Surveying and Mapping (SU), and Waterways and Harbors (WW), divisions. Papers sponsored by the Department of Conditions of Practice are identified by the symbols (PP). For titles and order coupons, refer to the appropriate issue of "Civil Engineering." Beginning with Volume 82 (January 1956) papers were published in Journals of the various Technical Divisions. To locate papers in the Journals, the symbols after the paper number are followed by a numeral designating the issue of a particular Journal in which the paper appeared. For example, Paper 1859 is identified as 1859 (HY7) which indicates that the paper is contained in the seventh issue of the Journal of the Hydraulics Division during 1958.

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